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***Wettability influence on the onset temperature of pool boiling:  
experimental evidence onto ultra-smooth surfaces.***

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21 **Abstract**

22 In this article we study systematically the effect of wettability on the onset of boiling on the same  
23 nanometrically smooth surface. By grafting different monolayers of molecules, we were able to  
24 explore the wettability from the equilibrium static contact angle,  $\theta_0 = 0^\circ$  to  $\theta_0 = 110^\circ$ , without  
25 changing the surface topography. The superheat temperature at the onset of pool boiling was  
26 measured and eventually a non-classical trend of  $T_{\text{ONB}}$  as a function of wettability was observed. The  
27 nucleation site densities for the different grafting cases were also measured by image analysis.  
28 Moreover, we propose a novel theoretical interpretation to this phenomenon linking nucleation and  
29 the molecular diffusion coefficient. MD simulation results support this approach.

30 **Keywords:** pool boiling, wettability, surface treatment, superheat, heat transfer.

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## 48 **1. Introduction**

49 When a liquid is increasingly heated by a solid surface, boiling eventually occurs. While this  
50 phenomenon has been observed in everyday life and in industry, its control is of crucial importance.  
51 In particular, one of the most efficient cooling systems consists in using the phase change of a liquid  
52 and its latent heat to transfer heat from one point to another. The transfer of the heat from the solid  
53 surface to the fluid is a function of several parameters such as heat flux, materials properties, and the  
54 geometry of the interface. When increasing the heat flux, the solid temperature increases, and the  
55 heat transfer from the solid to the liquid passes through various regimes. The first regime is the  
56 single-phase convection, which has a moderate ability to transfer the heat, and at a given superheat,  
57 i.e., at a temperature higher than the equilibrium saturation temperature, the so-called onset of  
58 nucleate boiling (ONB) occurs. During nucleate boiling, the successive formation and, in gravity  
59 conditions, the departure of bubbles are observed to be associated with high heat fluxes. When the  
60 heat flux increases even more and reaches the so-called critical heat flux (CHF), an insulating gas  
61 layer covers the solid surface, which results in a strong and sudden increase of the solid temperature.  
62 In all cases, superheat, which is the difference between the temperature of the solid,  $T_w$ , and the  
63 saturation of the liquid,  $T_{sat}$ , is necessary to start boiling. This superheat is generally a function of the  
64 characteristics of the solid/liquid interface [1-3]. In particular, surface topography is of primary  
65 importance, since boiling is usually associated with the presence of cavities. While many studies  
66 have shown the effect of changing these material-related parameters on the boiling onset [4-19],  
67 using nano-coatings [16] or nano-fluids [20] for example, only a few have managed to decouple the  
68 surface structure with other effects more linked to the chemical properties of surface, such as the  
69 surface wettability [21-24]. A recent review of the influence of topography and wettability effects in  
70 pool boiling has been done by Cheng et al [25]. Recently, using highly polished metallic surfaces,  
71 we have shown [26] that roughness amplitudes as low as a few nanometers can induce changes in  
72 the necessary superheat to achieve boiling. To decouple topographical effects (with their inherent  
73 complexity of characterization) from wettability, it is necessary to use even smoother surfaces such  
74 as glass or silicon wafers. The effect of wettability has already been shown using water on ungrafted  
75 glass plates (hydrophilic) and grafted octadecyltrichlorosilane (OTS) (hydrophobic) plates [27]. In

76 such a case, the contact angle is null for a hydrophilic surface while, for a hydrophobic surface, it is  
77 about  $115^\circ$  for the advancing contact angle and  $95^\circ$  for the receding contact angle. The ONB  
78 superheat is significantly lower (more than 18K) when using a hydrophobic surface compared to a  
79 hydrophilic one. More recently, Jo et al. [28] studied the heterogeneous nucleation on ideally smooth  
80 surface. They have found that their results did not match with classical nucleation theories. They  
81 thus have adapted a new model based on the thermal boundary layer. Their model includes the  
82 kinetic dynamics of the superheated liquid and the thermodynamic stability of the generated vapor,  
83 inside three different thermal boundary layers.

84 In this paper the surface wettability was changed by grafting different monolayers on the same  
85 ultrasmooth surface, without modifying its topography. A non-linear decrease of the superheat  
86 temperature at the onset of boiling has been observed for increasing contact angles. Moreover, this  
87 behavior cannot be predicted by classical nucleation theories, but also by Jo et al. model's [28]. The  
88 nucleation site density was also estimated by measuring the number of bubbles over the different  
89 treated surfaces [29-31]. These results can be linked at least qualitatively to a molecular diffusion  
90 coefficient, estimated by molecular dynamic (MD) simulations. The use of MD simulations to  
91 describe the mobility of the liquid molecules in the vicinity of the surface offers a novel theoretical  
92 approach to the nucleation phenomenon.

93

## 94 **2. Experimental methods**

### 95 *2.1. Materials and surface treatment*

96 Following the goal of measuring the isolated effect of the wettability on the onset of nucleate  
97 boiling, possible roughness contribution has been minimized as much as possible. For that, we used  
98 special glass surfaces (float glass; Oedenkoven Fr S.A.), which are very smooth. A 3D picture and a  
99 topography profile of the surface acquired by AFM is shown on Figure 1. The main roughness  
100 parameters are presented in Table 1. Moreover, glass was chosen because of its strength to prevent  
101 any damage on the surface during boiling in contrast to metallic surfaces [26] and for its sturdiness  
102 compared to silicon wafers. Milli-Q water was used during the boiling experiments. Several silanes

103 were used to modify the wettability of the glass substrate by chemical grafting of self-assembling  
104 monolayers (SAM).  
105 Octadecyltrichlorosilane (OTS) was purchased from Sigma-Aldrich (Germany), 2-[Methoxy-  
106 (polyethyleneoxy)propyl]-trimethoxysilane (MPEGPTMS) and 11-acetoxyundecyltriethoxysilane  
107 (AcOUTES) were purchased from Gelest, Inc. (USA). All chemicals were used without further  
108 purification. Each glass surface was cleaned and then grafted with a specific silane. The cleaning and  
109 grafting procedures have been optimized for MPEGPTMS [32], OTS [33,34] and AcOUTES [35]  
110 surfaces.  
111 The non-grafted surface was cleaned with a piranha solution and rinsed abundantly with Milli-Q  
112 water. After the cleaning procedure, the surface becomes super-hydrophilic. This means that when  
113 we deposit the drop of water on the surface, a thin film of water is rapidly created and it is  
114 impossible to measure any non-negligible contact angle (i.e.  $\theta_{adv.} = \theta_{rec.} = 0^\circ$ ).

115 **Insert Figure 1**

116

117

## 118 2.2. *Experimental setup and boiling procedure*

119 We characterize the boiling behavior of the various substrates by recording the temperature at the  
120 onset of boiling and visualizing the appearance of bubbles on the surface using a high-speed camera  
121 (HCC-1000, VossKühler GmbH, Germany). The details of the experimental setup can be found in a  
122 previous paper [27] and are summarized here.

123 Figure 2 presents a schematic of the chamber where boiling on the surface is activated. The surface  
124 to be tested, a glass square plate of 45 mm wide and 1 mm thick, is encapsulated in a chamber in  
125 which vacuum has been performed before filling with boiled water. The vacuum achievement (30  
126 mbars) obtained ensures a low level of dissolved air in the water [36]. Pressure is adjusted with a  
127 controller, and experiments are performed at atmospheric pressure; a bellows allows for dilatation of  
128 the working fluid during the experiment. A heater (ceramic cartridge, Acim Jouanin, 175 watts) is  
129 put in contact with the center part (20 mm diameter) of the back side of the glass plate using a  
130 spring. Between the heater and the plate, a heat-flux meter (Captec, France) enables the direct  
131 measurement of the heat flux; the thermal contact among various components is improved by using a  
132 conductive paste (RS, Heat Sink Compound Plus). The heating cartridge is insulated using Teflon.  
133 To compensate for possible thermal leakage, additional heaters are placed inside the chamber and on  
134 the aluminum walls of the chamber. The water in the chamber is maintained as close as possible to  
135 the saturation temperature by using these heaters, K-thermocouples in the chamber, and a PID  
136 controller.

137

138 **Insert Figure 2**

139

## 140 **3. Results and discussion**

141 Table 2 presents the various wettabilities obtained with the different surface treatments (cleaning  
142 only, and cleaning followed by the self-assembling of various silane monolayers). The low  
143 difference between values before and after boiling shows that grafting is of good quality. Moreover,  
144 the hysteresis of contact angles on glass surface is compatible with literature data [32-35].

145 The complete boiling curves are reported and discussed in Appendix 1. In this paper we focus  
146 mostly on the surface superheat at which the first bubbles are appearing versus the wettability of the  
147 solid surface, which is plotted vs. the surface wettability in Figure 3. In this figure, the wettability  
148 presented (angle  $\theta_0$ ) is calculated using the mean value of the four measured values of the advancing  
149 and receding static contact angles before and after boiling for each surface. In the same way, the  
150 error bars are estimated using the maximum of the hysteresis of the static contact angles (between  
151 the one before and the one after boiling) for each surface. The influence of the wettability is clear:  
152 Increasing the wettability on a very smooth, non-conductive surface results in an increase of the  
153 necessary superheat to get the appearance of the first bubble. The relationship is clearly not linear. If  
154 one would consider the theoretical case of perfect wetting (null static contact angles) with  
155 heterogeneous nucleation on an ideal smooth surface, an additional data point has to be added at  
156  $\Delta T \approx 200\text{K}$  [37]. Such a superheat is predicted by the classic heterogeneous nucleation theories  
157 without surface cavities, based on the thermodynamic calculations of the difference in terms of free  
158 energy; these theories are fully described in Carey's book [37]. In boiling experimentations, the  
159 systems used are far from ideal one. Even if the water and the chamber are well degassed, a small  
160 amount of air remains in the liquid. We have measured the partial pressure of air inside the chamber  
161 by subtracting the saturated pressure of the water from the total pressure of the chamber [36]. The  
162 estimated maximum partial pressure of air in the chamber during boiling experiments was about 2.1  
163 kPa. Moreover, the values reported in Figure 3 are of the same order of magnitude than other  
164 experimental works that have been presented in [28]. In this article, Jo et al. experimentally  
165 demonstrate that there is a decrease of the superheat needed to have boiling when the wettability of  
166 the surface is decreased. They predict theoretically the bubble nucleation condition by considering  
167 kinetic dynamics of water molecules and thermal equilibrium of the instantly generated bubble  
168 nucleus within three different boundary layer sizes. They also compare their predictions of the  
169 required superheat for the thermal equilibrium with two other works in literature [38,39] and show  
170 that the experimental results can be predicted by their model. Noteworthy, by comparing the  $R_a$   
171 parameter, in all of these works, the smoothest surface used is at least two times rougher than in the



172 present work. Moreover, the roughness of the samples is changing when the wettability of the  
173 surfaces is modified. Jo et al. did not mentioned the contact angle hysteresis of their substrates and  
174 its possible role. Nevertheless if we put our results in terms of superheat on their prediction graph  
175 (Figure 9 in Appendix 1), we can see that only two of our four points are inside their prediction  
176 curve. The OTS case is in the “Not Activated” area while the non-grafted case is completely out of  
177 the range of the predicted superheat.

178 Finally, in the present article, we are able to show a trend (Figure 3) for the superheat needed to get  
179 boiling for a wide range of wettabilities, without modifying the surface topography by surface  
180 treatments. These results, until now and to the best of our knowledge, have never been presented and  
181 cannot be predicted by any theory. To check the repeatability of the experiments, and also to  
182 experimentally point out the fact that there is no preferential nucleation site over the surfaces (no  
183 cavities), we have visualized three times the location of the first bubble over the surface. We have  
184 realized that the first bubble appeared each time at a different location supporting the fact that only  
185 the wettability is acting on the boiling onset and no deterministic point linked to a surface  
186 heterogeneity is generating the bubble onset.

187 **Insert Figure 3**

188 **Insert Figure 4**

189  
190 The upper part of Figure 4 shows the different boiling behavior between the AcOUTES (left) and the  
191 OTS (right) treatments at the same heat flux ( $12 \text{ kW/m}^2$ ). For the OTS case, bubbles nearly cover the  
192 entire heated surface; however, in the AcOUTES case, only a few bubbles are present. This means  
193 that there is an important proliferation in the number of nucleation sites passing from a nearly  
194 hydrophobic surface ( $\theta_0 \approx 75^\circ$ ) to a hydrophobic one ( $\theta_0 \approx 105^\circ$ ). The bottom part of this figure  
195 represents the quantification in terms of the number of nucleation sites per unit area (left) and vapor  
196 volume flux (right) at this heat flux for the three grafted surfaces (OTS, AcOUTES, and  
197 MPEGPTMS). The nucleation site density (bottom left) is in fact the mean number of bubbles

198 present on the surface. The error bars are the standard deviations. The vapor volume flux is  
 199 calculated by an estimation of the detached bubble volume per time unit. The measure of the volume  
 200 has been made by estimating the size of the bubbles, considering an axial symmetry. The error bars  
 201 are estimated by the error on the X and Y pixels on the images. The bubbles are considered to be  
 202 spherical. The non-grafted case is not represented because no bubbles were visualized at this level of  
 203 heat flux. Finally the two graphs in Figure 4 show the same exponential growing trend in function of  
 204 the contact angle. One can see that there is still a difference passing from a hydrophilic case to a  
 205 hydrophobic one. This behavior, to the best of our knowledge, has never been studied in the  
 206 literature and is not described by classical nucleation theories.

207 From these observations, we can conclude that there are more nucleation points on top of a  
 208 hydrophobic surface than on a hydrophilic surface. Moreover, from the topography of our surfaces  
 209 represented on Figure 1, and from the above-mentioned random location of nucleation sites in  
 210 consecutive experiments, we can assume that the nucleation sites are not due to the cavities present  
 211 over the substrate. Indeed, if we calculate the theoretical critical radius of nucleation, which is given

212 by  $r_c = \frac{2\sigma_{lv}}{P_{sat}(T_l) \exp\left[\frac{v_l(P_l - P_{sat}(T_l))}{RT_l}\right] - P_l}$  (details in Carey's book [37]), we find a critical radius value of

213 9  $\mu\text{m}$  for a superheat of about 3K (which corresponds to the most hydrophobic case) and a critical  
 214 radius value of 1  $\mu\text{m}$  for a superheat of about 20K (which corresponds to the most hydrophilic case).

215 These values are of course much higher than the cavity size present over our surfaces (Figure 1).

216 How can we explain this new finding? The relative low values of the superheat at the onset of  
 217 nucleate boiling is partially explained by the presence of nanobubbles over the surface. Indeed, it has  
 218 been shown [41] that air nanobubbles are always present over hydrophobic surfaces and that it is  
 219 possible that they remain on the surface even after one hour of degassing. No bubbles are detected  
 220 over hydrophilic surfaces, even if checked at room temperature [41]. In the present case, the  
 221 presence of such nanosystems is linked to the presence of both vapour nanobubbles and of air  
 222 nanobubbles. Eventually the liquid close to the surface is superheated, i.e in a thermodynamical  
 223 condition where it is possible to create vapor nanobubbles. These nanobubbles are usually unstable,

224 i.e. there is a statistical phenomenon of evaporation and condensation, but, in presence of an  
225 interface with chemical heterogeneities, they tend to settle on the surface with a higher probability to  
226 become stable and remain attached to the surface. Considering the air nanobubbles, there is a contact  
227 angle hysteresis on all surfaces we used, and the contact angles for the most hydrophilic case (after  
228 boiling experiment) are about  $9^\circ$  and  $30^\circ$  for the receding and advancing contact angles,  
229 respectively, which is in turn very similar to the MPEGPTMS case. The contact angle hysteresis is  
230 related to pinning [42] and therefore linked to the bubble liquid/air interfaces mobility on and from  
231 the surface. Therefore in our case we cannot exclude the presence of air nanobubbles even on  
232 hydrophilic surfaces. Moreover, we have measured the partial pressure of air inside the chamber and,  
233 from the Henry's law, we know that there is still a minimum quantity of air dissolved in the water  
234 (the estimated maximum partial pressure of air in the chamber during boiling experiments was about  
235 2.1 kPa). This dissolved gas, by diffusion, will be also attached on the chemical heterogeneities of  
236 the surface, acting as nucleation sites and thus leading to the small superheat temperature at the onset  
237 of boiling, even on hydrophilic smooth surfaces.

238 If we look now at the molecular level, how we can assume that the bubble nucleation starts with a  
239 nanobubble of air in contact with the solid surface? The growth of this bubble will be due to the  
240 incorporation of water molecules in the gas phase and will be limited by the mechanisms controlling  
241 the mobility of the corresponding phase line. As known, this contact line will be subject to pinning  
242 (contact angle hysteresis) and to some dynamics. For a low viscosity liquid such as water, the  
243 dynamics is well described by the molecular kinetic theory [40], which links the corresponding  
244 speed of displacement to the mobility of molecules in contact with the surface.

245 Moreover, the water molecules move with a speed related to the temperature. In the vicinity of the  
246 wall container (the heat source), it is expected that the water molecules will move faster than in the  
247 liquid bulk. However, it seems natural to expect that the nucleation points will be directly related to  
248 that mobility. Indeed, a nucleation point is an assembly of high-speed water molecules that will be  
249 well separated from each other during a nano-gas phase. The mobility of molecules can be described  
250 by the diffusion constant. The previous observations are thus related to an increase of the diffusion

251 constant versus the value of the contact angle. We know that the diffusion constant will increase  
252 with temperature.

253 Is that also the case with wettability? To check this idea, we have performed large-scale molecular  
254 dynamics simulations. Our molecular dynamics algorithm and interactions used in this study are  
255 standard except that the fluid is made of chains of Lennard-Jones atoms to control the viscosity of  
256 the liquid. In this way, we attempt to come closer to the experiments. The basic interaction between  
257 pairs of atoms is of the Lennard-Jones type:

$$258 \quad V_{ij}(r) = C_{ij} \left( \left( \frac{\sigma}{r} \right)^{12} - \left( \frac{\sigma}{r} \right)^6 \right)$$

259 Where  $i$  and  $j$  represent solid or fluid atoms, and the fluid atoms are grouped into chains of length 8  
260 with the additional pairwise (confining) potential:

$$261 \quad U(r) = A \frac{r}{\sigma^6}$$

262 between adjoining atoms. Aside from the issue of molecular size, the chain structure has the effect of  
263 strongly reducing the volatility of the fluid so as to bring the simulation closer to experiments. The  
264 substrate is a lattice made of one layer of face-centered cubic (fcc) unit cells, and of which an atom  
265 of mass 50 is at each site (so as to have comparable timescales for the atomic motion in fluid and  
266 solid). For computational convenience, the tail of the potentials are cut off at  $r_c = 2.5$ , in units of the  
267 fluid core size. Given the potential, the motion follows from integrating Newton's equations, using a  
268 fifth-order predictor-corrector algorithm. In the remainder of the section, we non-dimensionalize by  
269 using  $\sigma$ ,  $\epsilon$ , and the fluid monomer mass as the units of distance, energy, and mass, respectively. This  
270 model is simplistic in terms of interactions, but is self-consistent in terms of physics.

271 The diffusion coefficient is obtained by measuring the mean square displacement of atoms over long  
272 period of times:

$$273 \quad D_x = \frac{1}{2} \lim_{t \rightarrow \infty} \frac{d}{dt} \langle \sum (x_i(t) - x_i(t_0))^2 \rangle$$

274 Where  $x_i(t_0)$  is the x position of atom i at starting time  $t_0$ , and  $x_i(t)$  its position at time t. The brackets  
275  $\langle \rangle$  stand for the average over several starting times. More details about the system can be found in  
276 [40], where a detailed study of wettability has been performed. For instance, it has been shown that  
277 there is a direct link between the amplitude of interaction C and the static contact angle  $\Theta_0$ .

278 For the same systems, it has been shown that the density of the liquid in the vicinity of the wall is  
279 also affected by the coupling C. Here, we have studied the values of the diffusion constants in the  
280 liquid, close to the wall for a fixed temperature. The results are given in Figure 5.

281 **Insert Figure 5**

282 Moreover, if we take the diffusion coefficient in the vicinity of the solid surface from [40]  
283 corresponding to comparable static contact angles in our experiments, we are able to plot those  
284 values on the graph representing the number of nucleation sites versus the contact angle at a given  
285 heat flux (bottom left of Figure 4). This leads to Figure 6 where we can see that the trend is  
286 compatible with our experiments, strengthening the link between the mobility of the molecules very  
287 close to the surface and the nucleation site density over different surface wettabilities.

288 **Insert Figure 6**

289 This clearly shows that the considered molecules are much more mobile in the vicinity of the wall  
290 whenever the static contact angle  $\Theta_0$  is large, in agreement with the experiments described in this  
291 paper. Of course, the agreement here is purely qualitative, since we are unable to simulate in details  
292 the behavior of real water molecules. But nevertheless, we believe that these results are going in the  
293 right direction and will help to build a future predictive theory.

#### 294 **4. Conclusions**

295 Pool boiling experiments of water on smooth glass surfaces in stationary conditions were performed.  
296 The wettability was changed by grafting different monolayers on the surface. In this way, the  
297 wettability was modified without changing the roughness of the surface. We studied four different  
298 cases : (1) a non-grafted case, which has  $0^\circ$  contact angles (CA) before boiling; (2) an MPEGPTMS

299 case, which has about  $33^\circ$  CA; a AcOUTES case, which has about  $70^\circ$  CA; and an OTS case, which  
300 is the most hydrophobic one and has about  $102^\circ$  CA. The boiling curves have been measured for  
301 each case, and the temperature of the surface at the onset of boiling has been recorded. We have  
302 observed a non-linear decrease of the superheat temperature at the onset of boiling decreasing the  
303 wettability of the surface. We have also quantified the nucleation site density and the vapor volume  
304 flux at a given heat flux for each case. Once more, the trend is not linear with the wettability of the  
305 substrate, showing a big jump in passing from a hydrophilic surface to a hydrophobic one. An  
306 explanation for this property is given in terms of molecular diffusion constant. In fact, for liquid  
307 molecules in the vicinity of a solid wall, we have shown, using MD simulations, that the diffusion  
308 constant is larger when the wettability is poor. The trend of diffusion coefficient with static contact  
309 angle shows a nonlinear trend qualitatively similar to that of nucleation site density, which allows  
310 inferring a link between these two quantities. On the other hand, with a larger diffusion constant, we  
311 can expect a larger mobility and thus a larger probability to form a nanobubble of gas. This is in  
312 qualitative agreement with our experimental observations. The results are useful for a wide set of  
313 present and future applications, since the enhancement of boiling can be useful for microfluidic  
314 evaporators, such as in micro- and nanorefrigerators, micro-HVAC systems, thermalized bacteria  
315 detectors, and micro heat pipes.

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324

325 **Appendix 1**

326 Figure 7 represents the boiling curve recorded for each surface treatment. The error bars correspond  
327 to the standard deviation calculated on at least three measurements. We can see that only the most  
328 hydrophobic case (OTS on the graph) shows a change in the slope of the boiling curve. The three  
329 other curves shows a linear behavior. We interpret this result by the fact that the thermal  
330 conductivity of the glass surface is too low and the boiling area is too small to record any local  
331 cooling of the surface from the bottom of it (indeed, the thermocouples are situated below the glass  
332 surface).

333 **Insert Figure 7**

334

335 Figure 8 represents the heat transfer coefficient (HTC) versus the heat flux for each surface  
336 treatment. Again, we observe a linear behavior of the curves corresponding to the three more  
337 hydrophilic treatment. There is a jump in the HTC only in the case of the OTS treatment, where the  
338 boiling is fully developed over the entire surface.

339 **Insert Figure 8**

340 Figure 9 represents a comparison of the predicted superheats (Jo et al. [28]) as a function of contact  
341 angle for different thermal boundary layer thicknesses with different experiments. As we can see, the  
342 superheat measured in this study cannot be predicted by their model. Our most hydrophobic case  
343 (OTS case) is located in the “Not Activated” area of the graph and our most hydrophilic case (not  
344 grafted case) is far from the predicted superheat.

345 **Insert Figure 9**

346

347

348 **6. REFERENCES**

- 349 1. M. Mann, K. Stephan, P. Stephan, Influence of heat conduction in the wall on nucleate boiling  
350 heat transfer, *International Journal of Heat and Mass Transfer* 43 (2000) 2193–2203.
- 351 2. I.L. Pioro, W. Rohsenow, S.S. Doerffer, Nucleate pool-boiling heat transfer. I: review of  
352 parametric effects of boiling surface, *International Journal of Heat and Mass Transfer* 47 (2004)  
353 5033–5044.
- 354 3. H.T. Phan, R. Bertossi, N. Caney, P. Marty, S. Colasson, A model to predict the effect of surface  
355 wettability on critical heat flux, *International Communication of Heat and Mass Transfer* 39  
356 (2012) 1500–1504.
- 357 4. J.Y. Lee, M.H. Kim, M. Kaviany, S.Y. Son, Bubble nucleation in microchannel flow boiling  
358 using single artificial cavity, *International Journal of Heat and Mass Transfer* 54 (2011) 5139–  
359 5148.
- 360 5. C. Hutter, D.B.R. Kenning, K. Sefiane, T.G. Karayiannis, H. Lin, G. Cummins, A.J. Walton,  
361 Experimental pool boiling investigations of FC-72 on silicon with artificial cavities and  
362 integrated temperature microsensors, *Experimental Thermal and Fluid Science* 34 (2010) 422-  
363 433.
- 364 6. C. Hutter, K. Sefiane, T.G. Karayiannis, A.J. Walton, R.A. Nelson, D.B.R. Kenning, Nucleation  
365 site interaction between artificial cavities during nucleate pool boiling on silicon with integrated  
366 micro-heater and temperature micro-sensors, *International Journal of Heat and Mass Transfer* 55  
367 (2012) 2769–2778.
- 368 7. P. Papon, J. Leblond, P.H.E. Meijer, *The physics of phase transitions*, second ed., Springer-  
369 Verlag, Berlin, 2006, pp. 35-51.
- 370 8. H. Honda, J.J. Wei, Enhanced boiling heat transfer from electronic components by use of surface  
371 microstructures, *Experimental Thermal and Fluid Science* 28 (2004) 159-169.



- 372 9. L. Dong, X. Quan, P. Cheng, An analysis of surface-microstructures effects on heterogeneous  
373 nucleation in pool boiling, *International Journal of Heat and Mass Transfer* 55 (2012) 4376–  
374 4384.
- 375 10. T.G. Theofanous, J.P. Tu, A.T. Dinh, T.N. Dinh, The boiling crisis phenomenon Part I:  
376 nucleation and nucleate boiling heat transfer, *Experimental Thermal and Fluid Science* 26 (2002)  
377 775-792.
- 378 11. S. Vemuri, K.J. Kim, Pool boiling of saturated FC-72 on nano-porous surface, *International*  
379 *Communication of Heat and Mass Transfer* 32 (2005) 27–31.
- 380 12. T. Chen, J.F. Klausner, S.V. Garimella, J.N. Chung, Subcooled boiling incipience on a highly  
381 smooth microheater, *International Journal of Heat and Mass Transfer* 49 (2006) 4399–4406.
- 382 13. Y. Nam, J. Wu, G. Warriar, Y.S. Ju, Experimental and numerical study of single bubble  
383 dynamics on a hydrophobic surface, *Journal of Heat Transfer - Transactions of the ASME* 31  
384 (2009) 121004.
- 385 14. S. Kakaç, *Boilers, Evaporators, and Condensers*; Wiley, New-York, 1991.
- 386 15. A.E. Bergles, W.M. Rohsenow, The determination of forced-convection surface-boiling heat  
387 transfer, *Journal of Heat Transfer* 86 (1964) 365-370.
- 388 16. H.T. Phan, N. Caney, P. Marty, S. Colasson, J. Gavillet, Surface wettability control by  
389 nanocoating: The effects on pool boiling heat transfer and nucleation mechanism, *International*  
390 *Journal of Heat and Mass Transfer* 52 (2009) 5459–5471.
- 391 17. A.R. Betz, J. Xu, H. Qiu, D. Attinger, Do surfaces with mixed hydrophilic and hydrophobic  
392 areas enhance pool boiling?, *Applied Physics Letters* 97 (2010) 141909.
- 393 18. Z. Yao, Y.W. Lu, S.G. Kandlikar, Effects of nanowire height on pool boiling performance of  
394 water on silicon chips, *International Journal of Thermal Sciences* 50 (2011) 2084–2090.

- 395 19. H.J. Jo, H.S. Ahn, S.H. Kang, M.H. Kim, A study of nucleate boiling heat transfer on  
396 hydrophilic, hydrophobic and heterogeneous wetting surfaces, *International Journal of Heat and*  
397 *Mass Transfer* 54 (2011) 5643–5652.
- 398 20. S.J. Kim, I.C. Bang, J. Buongiorno, L.W. Hu, Surface wettability change during pool boiling of  
399 nanofluids and its effect on critical heat flux, *International journal of heat and mass transfer* 50  
400 (2007) 4105-4116.
- 401 21. O.W. Thomas, R.E. Cavicchi, M.J. Tarlov, Effect of surface wettability on fast transient  
402 microboiling behavior, *Langmuir* 19 (2003) 6168-6177.
- 403 22. K.M. Balss, C.T. Avedisian, R.E. Cavicchi, M.J. Tarlov, Nanosecond imaging of microboiling  
404 behavior on pulsed-heated Au films modified with hydrophilic and hydrophobic self-assembled  
405 monolayers, *Langmuir* 21 (2005) 10459-10467.
- 406 23. R. Rioboo, M. Marengo, S. Dall'Olio, M. Voué, J. De Coninck, An innovative method to control  
407 the incipient flow boiling through grafted surfaces with chemical patterns, *Langmuir* 25 (2009)  
408 6005-6009.
- 409 24. C. Choi, J.S. Shin, D.I. Yu, M.W. Kim, Flow boiling behaviors in hydrophilic and hydrophobic  
410 microchannels, *Experimental Thermal and Fluid Science* 35 (2011) 816-824.
- 411 25. P. Cheng, X.J. Quan, S. Gong, L.N. Dong, F.J. Hong, Recent studies on surface roughness and  
412 wettability effects in pool boiling, *Proceedings of the 15<sup>th</sup> International Heat Transfer*  
413 *Conference, IHTC-15, August 10-15, 2014, Kyoto, Japan.*
- 414 26. B. Bourdon, R. Rioboo, M. Marengo, E. Gosselin, J. De Coninck, Influence of the wettability on  
415 the boiling onset, *Langmuir* 28 (2012) 1618-1624.
- 416 27. B. Bourdon, P. Di Marco, R. Rioboo, M. Marengo, J. De Coninck, Enhancing the onset of pool  
417 boiling by wettability modification on nanometrically smooth surfaces, *International*  
418 *Communication in Heat and Mass Transfer* 45 (2013) 11-15.

- 419 28. H. Jo, M. Kaviany, S.H. Kim, M.H. Kim, Heterogeneous bubble nucleation on ideally-smooth  
420 horizontal heated surface, *International Journal of Heat and Mass Transfer* 71 (2014) 149-157.
- 421 29. N. Basu, G.R. Warrier, V.K. Dhir, Onset of nucleate boiling and active nucleation site density  
422 during subcooled flow boiling, *Journal of Heat Transfer* (2002), Vol 174 / 717.
- 423 30. G. Barthau, Active nucleation site density and pool boiling heat transfer- an experimental study,  
424 *International Journal of Heat and Mass Transfer* (1992), Vol. 35, No 2, pp 271-278.
- 425 31. R.J. Benjamin, A.R. Balakrishnan, Nucleation site density in pool boiling of saturated pure  
426 liquids: effects of surface microroughness and surface and liquid physical properties,  
427 *Experimental Thermal and Fluid Science* 1997; 15:32-42.
- 428 32. Papra, A.; Gadegaard, N.; Larsen, N.B. *Langmuir* **2001**, 17, 1457-1460.
- 429 33. S.R. Wasserman, Y.T. Tao, G.M. Whitesides, Structure and reactivity of alkylsiloxane  
430 monolayers formed by reaction of alkyltrichlorosilanes on silicon substrates, *Langmuir* 5 (1989)  
431 1074-1087.
- 432 34. R. Rioboo, M. Voué, M.H. Adão, J. Conti, A. Vaillant, D. Seveno, J. De Coninck, Drop impact  
433 on soft surfaces: beyond the static contact angles, *Langmuir* 26 (2010) 4873-4879.
- 434 35. M. K. Chaudhury, Surface free energies of alkylsiloxane monolayers supported on elastomeric  
435 polydimethylsiloxanes, *Contact Angle, Wettability and Adhesion*, 691-697, Ed. K. L. Mittal.
- 436 36. C. D. Henry, J. Kim, B. Chamberlain, Heater size and heater aspect ratio effects on subcooled  
437 pool boiling heat transfer in low-G, *Third International Symposium on Two-Phase Flow*  
438 *Modeling and Experimentation*, Pisa, 22-24 September 2004.
- 439 37. Van P. Carey, *Liquid-Vapor Phase-Change Phenomena*, Second Edition, 2008, Taylor & Francis  
440 Group, LLC.
- 441 38. H.J. Jo, S.H. Kim, M.H. Kim, Nucleate boiling performance on nano/microstructures with  
442 different wetting surfaces, *Nanoscale Res. Lett.* 7 (2012) 1-9.

- 443 39. Y. Takata, S. Hidaka, T. Uraguchi, Boiling feature on a super water-repellent surface, Heat  
444 Transfer Eng. 27 (2006) 25-30.
- 445 40. E. Bertrand, T. Blake, J. De Coninck, Influence of solid-liquid interactions on dynamic wetting:  
446 a molecular dynamics study, Journal of Physics Condensed Matter 21 (2009), 464124.
- 447 41. X.H. Zhang, G. Li, N. Maeda, J. Hu, Removal of induced nanobubbles from water/graphite  
448 interfaces by partial degassing, Langmuir (2006), 22, 9238-9243.
- 449 42. P. Collet, J. De Coninck, F. Dunlop et A. Reignard« Dynamics of the contact line: contact angle  
450 hysteresis »Phys. Rev. Lett. 1997, 79 (19), 3704–3707
- 451
- 452

453 **7. Tables**

454 Table 1: Main surface roughness parameters of the glass substrate measured by AFM in tapping  
 455 mode on a 1  $\mu\text{m}$  x 1  $\mu\text{m}$  area.

8. Roughness parameters	9. Values (nm)
10. $S_a$	11. 0.43
12. $S_q$	13. 0.62
14. $S_z$	15. 7.8
16. $S_v$	4.2

456

457

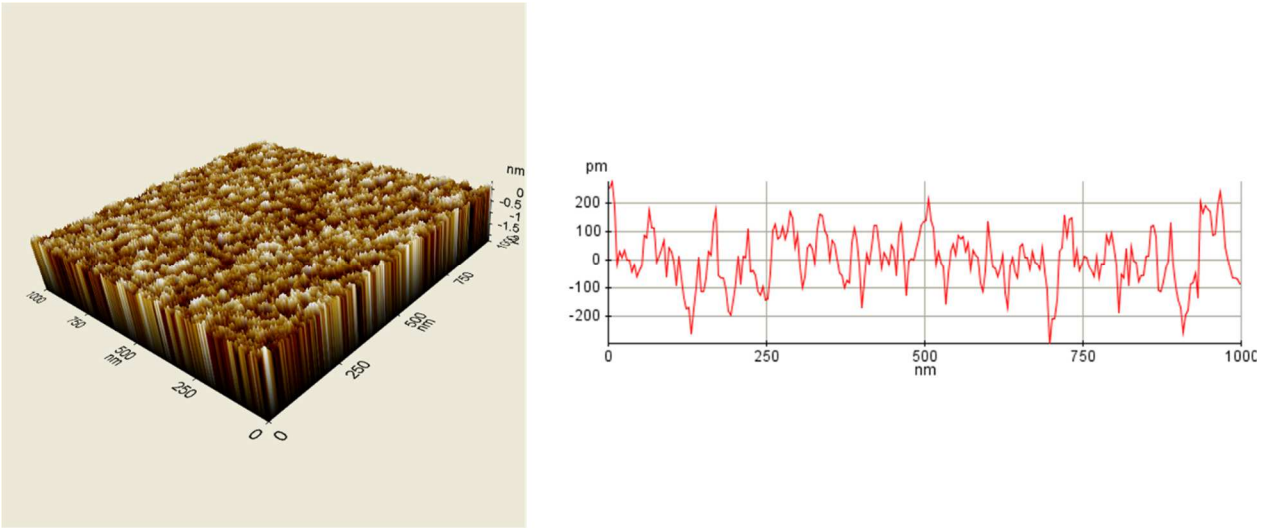
458 Table 2: Different treated glass surfaces' wettability and the contact angle hysteresis before and after  
 459 boiling experiments. The error bars are calculated using standard deviation over at least 5 measures.

Surface treatment	Before boiling experiments			After boiling experiments		
	$\theta_r$ (°)	$\theta_a$ (°)	$H = \theta_a - \theta_r$ (°)	$\theta_r$ (°)	$\theta_a$ (°)	$H = \theta_a - \theta_r$ (°)
Not grafted	0	0	0	$8 \pm 1.5$	$36.5 \pm 1.9$	$28.5 \pm 3.4$
MPEGPTMS	$33.5 \pm 1.7$	$37.1 \pm 0.3$	$3.6 \pm 2$	$26.1 \pm 1.9$	$37.1 \pm 0.1$	$11 \pm 2$
AcOUTES	$66.4 \pm 1.2$	$88.3 \pm 4.4$	$21.9 \pm 5.6$	$46.8 \pm 2.1$	$81.2 \pm 4.1$	$34.4 \pm 6.2$
OTS	$94.6 \pm 0.8$	$113.9 \pm 2.9$	$19.3 \pm 3.7$	$90.2 \pm 5.4$	$107.8 \pm 1.2$	$17.6 \pm 6.6$

460

461 **17. Figure captions**

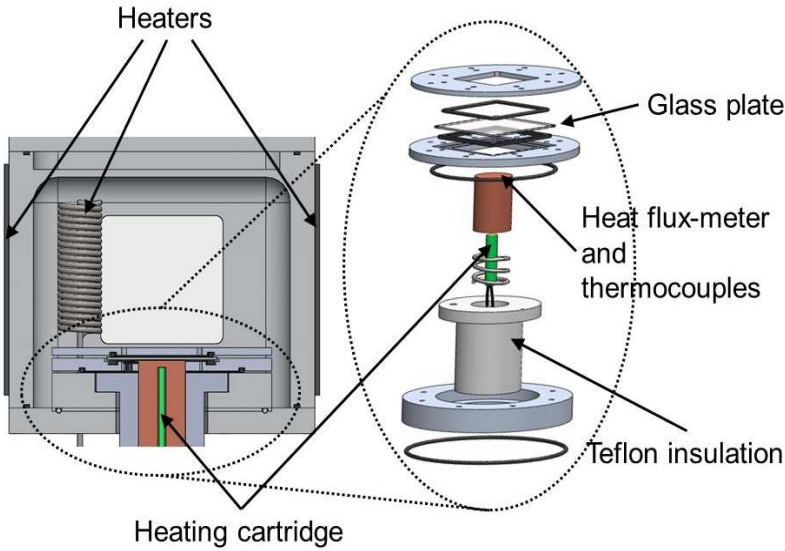
462 Figure 1



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464

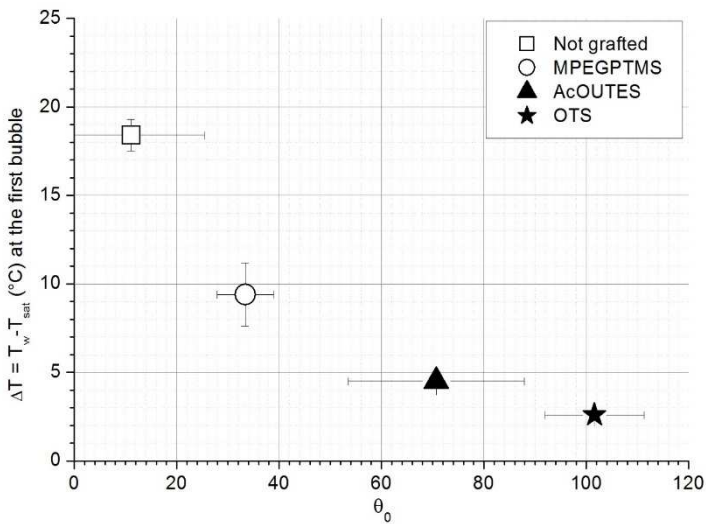
465 Figure 1: 3D view of the surface (left) and profile view (right) measured with AFM in tapping mode.



466

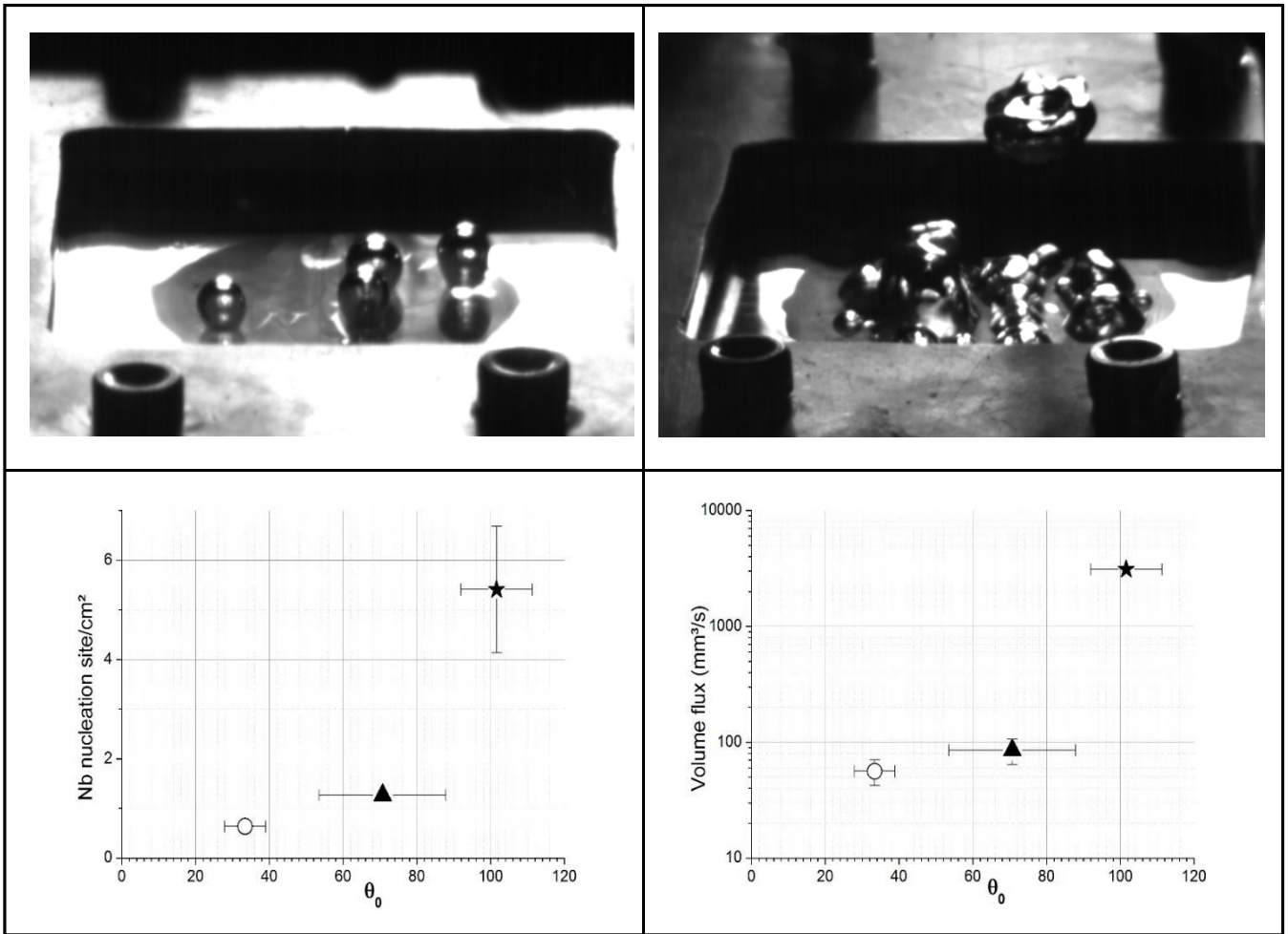
467

468 Figure 2: Schematic of the chamber. The glass plate, grafted with various silanes, is our studied  
 469 surface.



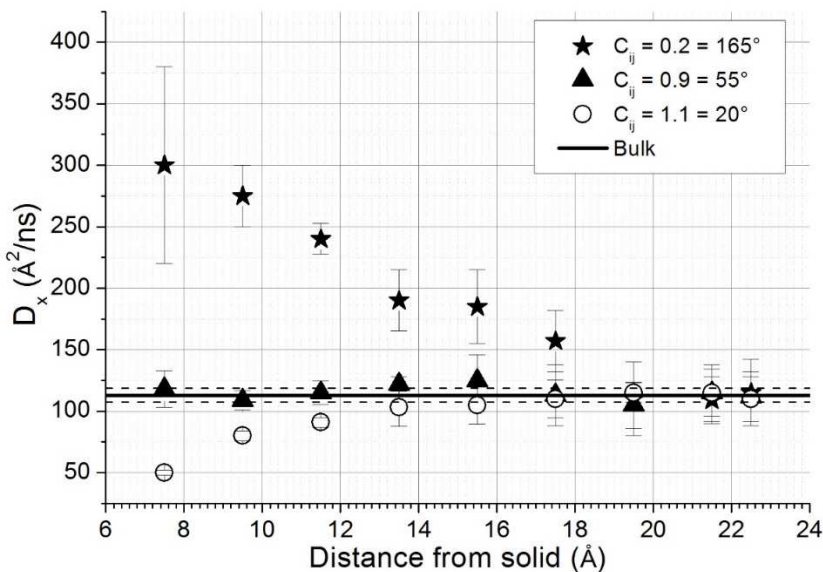
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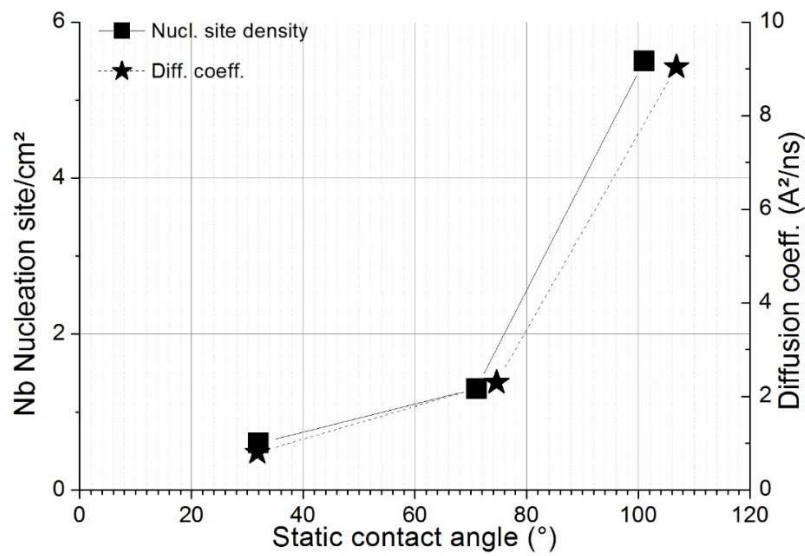
472

473 Figure 4: Comparison between surfaces boiling at the same heat flux (12kW/m²) on the smooth  
 474 surfaces. Top: images from a tilted angle presenting the difference of boiling intensity on the surface  
 475 (left: AcOUTES; right: OTS). The diameter of the heated disk is two centimeters. Bottom:  
 476 quantification in terms of nucleation site density (left) and vapor volume flux (right) of the boiling  
 477 process in function of the wettability. Symbols correspond to, from left to right, MPEGPTMS,  
 478 AcOUTES and OTS.  
 479



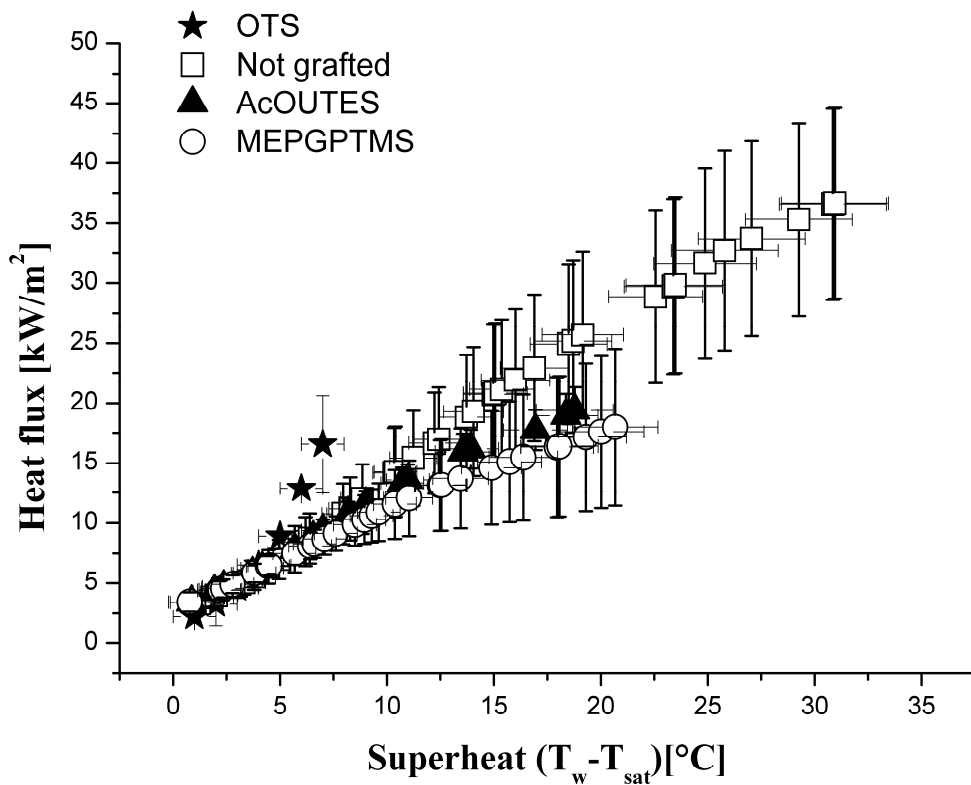
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481 Figure 5: Diffusion constant versus the distance from the solid for three different coupling values.  
 482 The coupling values 0.2, 0.9, and 1.1 correspond to a static contact angle of 165°, 55°, and 20°,  
 483 respectively.  
 484



485

486 Figure 6: Nucleation site density (from our experiments) and diffusion coefficient (from molecular  
 487 dynamics simulations) versus the static contact angle.  
 488



489

490 Figure 7: Boiling curves for the different surface treatments. Only the OTS case shows a change in  
 491 the slope of the curve when boiling occurs.

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501 **18. Figures**

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