

Motorcycle emergency steering assistance: A systematic approach from system definition to benefit estimation and exploratory field testing

Mirco Bartolozzi^{a,*}, Adelmo Niccolai^a, Cosimo Lucci^a, Giovanni Savino^a

^a*Department of Industrial Engineering, University of Florence, Via di Santa Marta 3, Florence, 50139, Tuscany, Italy*

Abstract

Braking assistance systems are already contributing to improving motorcyclists' safety; however, research on emergency systems acting on the steering is lacking. These systems, already available for passenger cars, could prevent or mitigate motorcycle crashes in which safety functions based only on braking are ineffective. The first research question was to quantify the safety impact of diverse emergency assistance systems acting on the steering of a motorcycle. For the most promising system, the second research question was to assess the feasibility of its intervention using a real motorcycle.

Three emergency steering assistance systems were defined in terms of Functionality, Purpose, and Applicability: Motorcycle Curve Assist (MCA), Motorcycle Stabilisation (MS) and Motorcycle Autonomous Emergency Steering (MAES). Experts evaluated each system's applicability and effectiveness based on the specific crash configuration (using Definitions for Classifying Accidents - DCA), the Knowledge-Based system of Motorcycle Safety (KBMS), and the In-Depth Crash Reconstruction (IDCR). An experimental campaign was conducted with an instrumented motorcycle to assess the rider's reaction to external steering input. A surrogate method for an active steering assistance system imparted

*Corresponding author

Email addresses: mirco.bartolozzi@unifi.it (Mirco Bartolozzi), adelmo.niccolai@unifi.it (Adelmo Niccolai), cosimo.lucci@unifi.it (Cosimo Lucci), giovanni.savino@unifi.it (Giovanni Savino)

external steering torques in correspondence with a lane change to analyse the effect of the steering inputs on motorcycle dynamics and rider controllability.

MAES globally got the best score for each assessment method. MS received better evaluations than MCA in two out of three methods. The union of the three systems covered a sizeable fraction of the crashes considered (maximum score in 22.8% of the cases). An estimation of the injury potential mitigation, based on injury risk functions for motorcyclists, was made for the most promising system (MAES). The field test data and video footage showed no instability or loss of control, despite the high intensity ($> 20 \text{ N m}$) of the external steering input. The rider interviews confirmed that the external action was intense but manageable.

For the first time, this study presents an exploratory assessment of the applicability, benefits, and feasibility of motorcycle safety functions acting on the steering. MAES, in particular, was found applicable to a relevant share of crashes involving motorcycles. Remarkably, applying an external action to produce a lateral avoidance manoeuvre proved feasible in a real-world test setting.

Keywords: Road safety, Motorcycle steering assistance, Injury mitigation, Emergency avoidance, Crash prevention, Experimental testing

1. Introduction

1.1. Background

The safety performance of road vehicles has seen significant improvement in the past two decades due to recent technological advancements and the introduction of advanced driver assistance systems. This development has also extended to Powered Two-Wheelers (PTWs - which includes motorcycles, scooters, and mopeds), for which several systems like the Anti-lock Braking System (ABS), Traction Control (TC), and Motorcycle Stability Control (MSC) have already gained recognition in preventing crashes [1, 2, 3].

Despite significant improvements, PTWs still represent a high-risk option compared to other modes of transportation due to the increased likelihood of

12 severe injuries and fatalities in the event of a crash [4]. In order to further
13 enhance the safety performance of PTWs, various assistance systems are cur-
14 rently under design or in early-stage testing, and they could become available
15 in the future. Such systems include warning systems, collision avoidance and
16 intersection support, human-machine interfaces, and vision assistance [5]. Ac-
17 cording to a recent systematic review, among the active onboard systems under
18 development, those capable of autonomously modifying vehicle dynamics are
19 considered the most promising [5].

20 An example is Motorcycle Autonomous Emergency Braking (MAEB), a sys-
21 tem designed to deploy a braking action autonomously without requiring input
22 from the rider when an imminent collision is detected to mitigate rider injuries
23 by reducing impact speed. Its applicability has been investigated in different
24 traffic environments [6], with promising outcomes in reducing injuries [7], and its
25 intervention resulted manageable by ordinary riders in real-world conditions [8].
26 Although MAEB was shown to be applicable also during lane change manoeu-
27 vres [7], its application is essentially designed for straight-line riding conditions
28 with limited roll angles.

29 There is a non-negligible proportion of crashes in which MAEB cannot be
30 employed, or its effectiveness is modest [6]. These are the crash configurations
31 in which an avoidance manoeuvre or a trajectory adjustment is more effective
32 than a braking action in avoiding the crash [9], such as crashes without the
33 direct involvement of other vehicles or crashes caused by vehicle loss of control.
34 At present, no active assistance system for PTWs controlling the steering of
35 the vehicle to modify the trajectory autonomously is currently available, as
36 identified by the aforementioned systematic review [5].

37 *1.2. Objective and outline*

38 This paper aims to provide an exploratory assessment of the potential of
39 innovative safety systems for PTWs based on emergency steer control actions
40 aiming to modify or stabilise the trajectory of a PTW to prevent or mitigate
41 crashes. The assessment will be based on their applicability to different crash

42 scenarios and configurations and on the estimate of their effectiveness in avoid-
43 ing or mitigating crashes. The most promising system shall also be evaluated
44 concerning its benefits in reducing the risk of injuries for the rider and the
45 feasibility of its action in the real world through preliminary field trials.

46 The article is structured as follows. Section 2 describes the three safety sys-
47 tems considered in the article, the three investigation methods used to assess the
48 applicability and effectiveness of each function, the approach used to estimate
49 the injury reduction in a selected case of real crashes, and the test protocol
50 used to experimentally test the feasibility of changing the PTW’s lateral posi-
51 tion through external steering actions. Section 3 presents the results regarding
52 applicability and effectiveness. Additional results regarding injury mitigation
53 potential and experimentally tested feasibility are provided for the most promis-
54 ing system. Section 4 presents a detailed discussion regarding these results and
55 their significance. Finally, Section 5 summarises these findings, their potential
56 consequences, and potential future uses.

57 **2. Materials and Methods**

58 *2.1. Safety Functions Considered*

59 This work employs the concept of *Safety Function* (SF). Following the defi-
60 nition of Gil et al. [10], a SF “unequivocally describes the desired outcome for a
61 safety solution, emphasising its goals regardless of the constitutive mechanisms
62 or sub-systems”. The three SFs proposed and evaluated in this work are Mo-
63 torcycle Curve Assist (MCA), Motorcycle Stabilisation (MS), and Motorcycle
64 Autonomous Emergency Steering (MAES), defined as follows:

- 65 • MCA: Helps the rider to approach or negotiate a curve when the current
66 speed or trajectory is inappropriate [11].
- 67 • MS: Helps the rider to assure the vehicle stability or dampen the oscilla-
68 tions after some perturbation which might cause the loss of control (road
69 unevenness, wind, momentary loss of friction).

70 • MAES: Acts autonomously or assists the rider in modifying the motorcycle
71 trajectory to avoid an imminent collision or a crash.

72 Each SF is considered in terms of Functionality, Purpose, and Applicability,
73 described in detail in Table 1. In the article, the SFs will be evaluated through
74 the concepts of *Applicability* (“Does the SF apply to the crash scenario? Is
75 the SF relevant in the crash scene?”) and *Effectiveness* (“If the safety function
76 applies to the scenario, how helpful is it?”).

Table 1: The Functionality, Purpose, and Applicability of each Safety Function (SF) considered in the study.

SF	Functionality	Purpose	Applicability
MCA	Utilises a motorcycle model, digital map, GNSS, and an IMU to estimate the motorcycle state and compute control actions for safe road keeping. Intervenes through steering torque and deceleration adjustments if the actual manoeuvre deviates from that computed over a threshold.	Helps the rider to stay on the road and in their lane while approaching and navigating curves by applying countermeasures, namely reducing speed or adjusting lane position, to prevent loss of control or veering off the road.	Applicable when the rider may be approaching or navigating a curve with excessive speed, incorrect trajectory, or improper inputs.
MS	Monitors the motorcycle dynamics and adjusts steering torque to prevent or reduce potential loss of control or oscillation.	Assists rider in controlling the vehicle during disturbances (e.g. lateral wind, loss of friction, wobble) to minimise oscillation or maintain control.	Applies when the rider struggles to maintain motorcycle stability due to external disturbances or inherent oscillation modes
MAES	Scans surroundings using sensors, predicts collisions, and applies steering torque to adjust trajectory based on constraints (lateral, longitudinal grip) and boundary conditions (road width, vehicles positions) when the time to collision falls below a threshold.	Prevents imminent crashes or assists the rider in avoiding them by adjusting the vehicle's trajectory	Applicable when it detects an obstacle and is feasible to trigger a new trajectory by obtaining the required lateral acceleration. It can be applied when there are other vehicles or obstacles present in the surroundings.

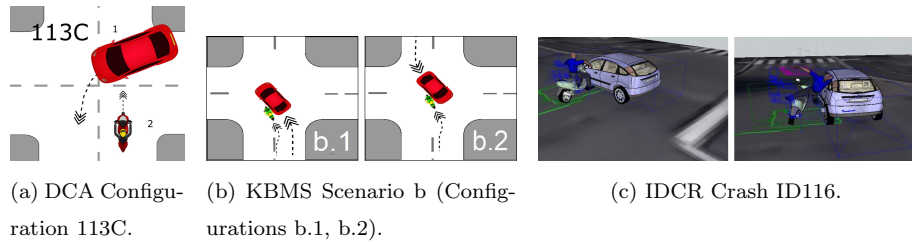


Figure 1: Examples of crash scenarios for the DCA, KBMS and IDCR approaches.

77 *2.2. Crash Data Investigation*

78 *2.2.1. DCA*

79 The VicRoads Definitions for Classifying Accidents (DCA) is a coded chart
 80 used to report crashes in Australia and to describe the crash configurations [12].
 81 Savino et al. [13] expanded the number of configurations from 81 to 152 to un-
 82 equivocally describe the trajectory of the motorcycles concerning the opposing
 83 vehicle. Each configuration was represented through a specific pictogram: an
 84 example is shown in Figure 1a (crash configuration 113C - ‘Adjacent direction,
 85 PTW into car’).

86 A four-class code system was developed to describe the *Applicability*. The
 87 possible classes, or scores, were ‘1’ (“The system would definitely *not* have ap-
 88 plied to crashes belonging to this specific scenario”), ‘2’ (“Would possibly have
 89 applied”, controversial), ‘3’ (“Would probably have applied”, technical chal-
 90 lenges still need to be solved), and 4 (“Would have applied”, typical application
 91 of the system).

92 In the current article, detailed and specific rules were defined for each SF
 93 considered and each rating class; this reduced the possibility of an incorrect
 94 interpretation by the examiners during the evaluation process. The examiners
 95 were aided by one flowchart for each SF (provided in Appendix A). Scores
 96 were given only on whether a system would be relevant to the crash scenario;
 97 the possible, consequent crash avoidance or mitigation was not considered. Two
 98 authors independently assigned a score to each SF for the DCA scenario. When
 99 the two evaluators disagreed, a third examiner provided an additional score, and

100 the score given twice was chosen. If all three evaluators disagreed, as it happened
101 in two scenarios, the median of the three scores was taken. The categorisation
102 agreement was analysed through Cohen’s quadratically weighted kappa coefficient
103 and used as a measure of inter-rater reliability statistics [14, 15]. Weights
104 of 0, 0.55, 0.88, and 1 were used for instances of complete agreement, a differ-
105 ence of one class, a difference of two classes, and a difference of three classes,
106 respectively. Consequently, higher degrees of disagreement were weighted more
107 than lower ones to reflect the unequal distinction between categories.

108 In this work, the Prato-X database was used for the DCA assessment. The
109 database includes the crash reports collected by the police in 2018 on the roads
110 of the municipality of Prato (Italy). In particular, only the crashes involving
111 at least one Powered Two-Wheeler (PTW) were used: these were extracted
112 from the database by Terranova et al. [6]. A total of 285 crashes were classified
113 following the DCA, using additional variables in some scenarios, like the presence
114 of loss of control, to specify the circumstances of each crash better.

115 2.2.2. KBMS

116 The Knowledge-Based system of Motorcycle Safety (KBMS) was used in a
117 previous work by Gil et al. [10] to evaluate the *Effectiveness* of SFs for PTWs.
118 A summary of the methods is given here: refer to Gil’s work for a more detailed
119 description. The process is divided into two phases:

- 120 1. *Collecting Phase*. Crashes are extracted from crash databases and divided
121 into subsets by crash configuration (26 crash scenarios, grouped into 9 gen-
122 eral scenarios) based on a set of queries. Figure 1b shows, as an example,
123 the general scenario ‘b’, divided into the two crash scenarios ‘b.1’ and
124 ‘b.2’. A panel of experts is defined; each evaluator assesses the *effective-*
125 *ness* of each SF for each motorcycle road crash scenario. A scoring scale
126 was defined to guarantee consistency in the scores assigned by evalua-
127 tors, and it is provided in Table 2. The scores ranged from ‘0’ (“The SF
128 *never* activates or produces *no effect*”) to ‘4’ (“Assuming activation, the
129 outcomes are *excellent*”) and were given concerning each of the following

Table 2: Scoring scale used to evaluate the *Effectiveness* of each Safety Function with respect to the three intervention mechanisms: ‘Prevention’ (the SF prevents the occurrence of a dangerous situation), ‘Avoidance’ (the SF intervenes in a dangerous situation and avoids the crash), and ‘Mitigation’ (the SF intervenes in a dangerous situation and mitigates the crash consequences).

Score	Meaning
0	The SF <i>never</i> activates / produces <i>no effect</i>
1	Assuming activation, the outcomes are <i>poor</i>
2	Assuming activation, the outcomes are <i>minor</i>
3	Assuming activation, the outcomes are <i>good</i>
4	Assuming activation, the outcomes are <i>excellent</i>

130 intervention mechanisms: *Prevention*, *Avoidance*, and *Mitigation*.

131 2. *Processing Phase*. A crash database is chosen. All information collected
 132 about crashes, like the statistical relevance of each type of crash and the
 133 potential of each SF given by the expert, are implemented through the
 134 equations described by Gil et al. to obtain a list of prioritised SFs.

135 In this article, the KBMS method was employed considering three years of
 136 the ISTAT database (2010-2012, comprising 205,272 PTW crashes that occurred
 137 in Italy). The KBMS was populated through the assessment of six experts in
 138 the motorcycle road safety field (the four authors and two external evaluators),
 139 who estimated the potential of each of the three SF proposed in this article.

140 2.2.3. *IDCR*

141 The In-Depth Crash Reconstruction (IDCR) method evaluates the effective-
 142 ness of the SFs on real crash scenarios, contrary to the DCA and KBMS meth-
 143 ods, where the crashes were schematised and simplified in appropriate crash
 144 configurations. Therefore, the IDCR method requires more time to investigate
 145 a single crash. This method allows checking whether the results obtained by
 146 the SFs when using a large number of less detailed crashes are coherent with
 147 those obtained considering a smaller number of crashes described in-depth.

148 In this work, the method was applied to crashes in the In-SAFE database,
149 which contains crashes in the area of Florence (Italy), where at least one PTW
150 was involved, in the 2009-2013 period [16]. The pre and post-crash dynamics
151 of each were reconstructed in detail: the travelling speed, the trajectory of
152 the vehicles, and other parameters, as the weather and lighting conditions, are
153 known. Figure 1c provides an example, showing the reconstructed crash ‘ID116’.
154 Three researchers used this information to evaluate 19 cases; the final score for
155 the safety function in the specific crash is obtained from the discussion and
156 agreement between the three researchers. The scoring scale is the same used in
157 the KBMS method (Table 2).

158 *2.2.4. Injury Mitigation*

159 Lucci et al. [11] estimated the predicted injury risk reduction due to a system
160 that slowed down the motorcycle when approaching a corner at excessive speed.
161 This safety function, called Motorcycle Curve Assist, had a similar aim to the
162 version proposed in the current article (which also acts on the steering). For
163 MS, this method for estimating injury mitigation was not appropriate, as it was
164 based on reducing relative crash speed; in fact, MS focused on crash avoidance
165 instead of mitigation. Therefore, the approach was applied only to MAES.

166 A subset of the crashes employed in the IDCR method was used to evaluate
167 the injury reduction benefits of MAES intervention, even when there was in-
168 sufficient time to avoid the opposing vehicle since the system was activated. In
169 particular, nine crashes (more than the number of crashes that received scores ‘3’
170 or ‘4’ in IDCR, equal to eight) were considered among those where another ve-
171 hicle was involved. After reconstructing the crash scenario, the same crash was
172 simulated with the hypothesis of a MAES intervention which changed the ve-
173 hicle trajectory. Three MAES activation simulations were done for each crash,
174 using three lateral acceleration values (0.3g, 0.5g, 0.7g). Given the potential
175 complexity of MAES control logic, and the exploratory scope of this work, a
176 simple kinematic approach was used. The activation of the system modified
177 the vehicle trajectory: it produced a lateral acceleration, inducing a yaw an-

178 gle variation and a lateral displacement over time. The vehicle speed did not
 179 change compared to the same crash simulated without MAES activation. The
 180 variation of the vehicle lateral acceleration was instantaneous as soon as MAES
 181 activated, going from zero to a constant value with no transient. The idea be-
 182 hind this hypothesis was to evaluate the impact of the system regardless of the
 183 rider’s action, the vehicle dynamics or the constructive constraint, like whether
 184 the torque needed to steer the motorcycle would be compatible with a specific
 185 electromechanical system. Giovannini used this simplified approach to model an
 186 evasive manoeuvre; as in that work, the initial small outwards yaw rate typical
 187 of PTWs was neglected [9].

Under the previous hypothesis, the equations that govern the vehicle motion through time are the following:

$$\psi(t) = \psi_0 + \int_0^t \frac{a_y}{v(\tau)} d\tau, \quad (1)$$

$$v_x(t) = v(t) \cos(\psi(t)), \quad (2)$$

$$v_y(t) = v(t) \sin(\psi(t)), \quad (3)$$

where ψ is the yaw angle, a_y is the lateral acceleration (0.3g, 0.5g, 0.7g), v is the vehicle’s speed, and $v_{x,y}$ are the x, y components of the vehicle velocity in the ground frame of reference. The x and y vehicle coordinates were then obtained by integrating Equations (2) and (3) with respect to time. Avoidance of the crash was obtained in some cases. The time when the system was active was different for each crash and depended on the dynamics of the crash. The relative crash speed was computed as the magnitude of the relative velocity between the PTW and the obstacle at the time of the crash:

$$v^{\text{rel}} = \|\mathbf{v}^{\text{rel}}\| = \|\mathbf{v} - \mathbf{v}^{\text{obstacle}}\| = \sqrt{(v_x - v_x^{\text{obstacle}})^2 + (v_y - v_y^{\text{obstacle}})^2}. \quad (4)$$

188 The change of PTW yaw angle caused a variation of the ‘Relative Heading Angle’
 189 between the vehicles, which was responsible for the variation of the relative
 190 speed v^{rel} when the system activated. The relative speed was then employed to
 191 calculate the injury risk reduction provided by system intervention, using the

192 Risk Functions proposed by [17]. These are multivariate injury risk models for
193 PTW users to estimate the risk of sustaining different levels of injuries based
194 on the relative speed and crash characteristics. Absolute and relative injury
195 risk reductions were calculated, as detailed in a previous study [7], based on the
196 variation of the relative impact speed of the PTW thanks to MAES intervention.
197 Three levels of injury severity were considered: ‘MAIS2+F’, ‘MAIS3+F’, and
198 ‘Fatal’ injuries, where MAIS is the maximum injury score reported by the rider
199 using the Abbreviated Injury Scale [18].

200 *2.3. Experimental Test*

201 The most promising system, concerning applicability and effectiveness, was
202 tested in terms of feasibility using a real motorcycle. MAES was the SF with
203 the highest applicability and effectiveness, as shown in Section 3; consequently,
204 the rider’s reaction to its external steering input was tested in its most typical
205 scenario: a lane change.

206 An experimental test was conducted using an instrumented motorcycle,
207 shown in Figure 2a. An inertial measurement unit (XSens 680G) acquired the
208 vehicle’s motion, measuring its orientation, position, and corresponding deriva-
209 tives. The steering torque was computed through the measurement made by
210 two pairs of strain gauges; each pair was applied to each half-handlebar. The
211 strain gauge reading (a voltage value linked to its deformation) was converted
212 into a steering torque around the steering axis through a calibration procedure.
213 The steering torque τ was computed as the difference between the right and left
214 measurements [19]. In the current work, the ISO 8855 [20] signs convention was
215 used (Figure 2b): the roll angle ϕ around the forward, longitudinal axis was
216 positive when the motorcycle was leaning towards the right; the yaw angle ψ
217 around the upward, vertical axis was positive when the motorcycle was headed
218 towards the left; lastly, the steering torque and the steering angle were defined
219 around the steering axis, and were positive when anti-clockwise when seen from
220 above.

A surrogate method for an active steering assistance system was employed:

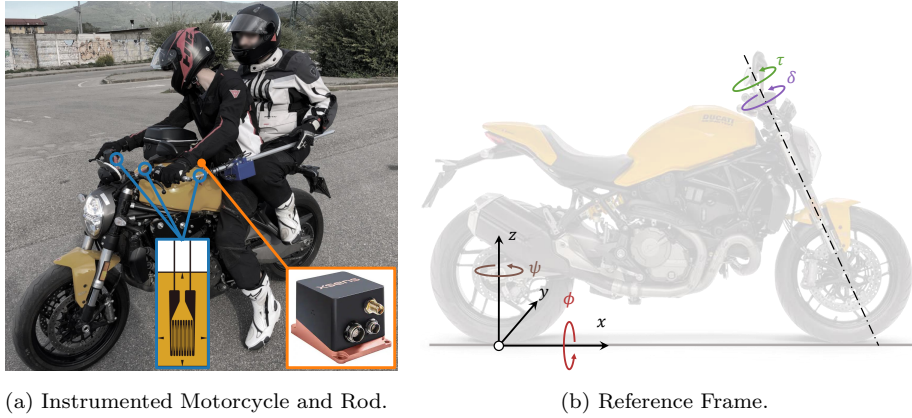


Figure 2: The instrumented motorcycle and rod used in the experiment and the coordinate frame used in the study showing the positive signs for roll ϕ and yaw ψ motions and steering torque τ and angle δ .

the external steering torque was applied by the pillion passenger through an instrumented rod, shown in Figure 2a. The pillion passenger held one side of the rod in his left hand while the other was connected to the handlebar through a spherical joint. By pushing the rod, the passenger could apply a clockwise steering torque; pulling the rod, instead, generated anti-clockwise steering torque. This method was straightforward and, therefore, more appropriate for an initial feasibility evaluation than a mechatronic system acting on the front assembly through a power steering or steer-by-wire action. The system was simple and unaffected by electrical failures or bugs, making the test safer and not influenced by the specific control properties of the system: this exploratory test aimed to evaluate the rider's response to an external, concurrent steering action and not the control logic of the system itself. Due to the behavioural aim of the test, the effect of the added inertia due to the passenger was not considered a limiting factor. A strain gauge was applied to a rod section and measured its axial deformation. The sensor was calibrated by manually applying sinusoidal steering inputs to the handlebar through the rod itself, while no other actions were present on the handlebar. A linear regression between the steering torque computed from the left semi-handlebar strain gauges readings and the strain

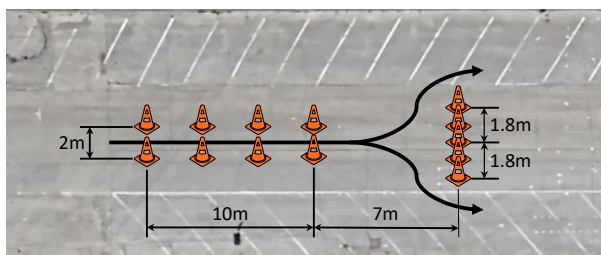


Figure 3: The experimental test protocol. The rider performed a $1.8\text{ m} \times 7\text{ m}$ lane change, in a predefined direction. The rod always exerted an additional steering torque in the initial phase. In the case of the Double Actuation trial, it also acted in the straightening phase.

gauge applied on the rod provided the sensitivity coefficient needed to compute the external steering torque τ_{ext} applied by the rod. The rider and the passenger contributed to the total steering torque, measured by the strain gauges on the handlebar. The steering torque contribution τ_{rider} applied by the rider was then derived as:

$$\tau_{\text{rider}} = \tau - \tau_{\text{ext}}. \quad (5)$$

221 The test comprised two trials performed on a cone course in a parking lot
 222 closed to traffic. Each trial consisted of four lane change manoeuvres in each
 223 direction. Figure 3 shows the manoeuvre geometry: the motorcycle performed
 224 a lane change with 1.8 m lateral offset and a 7 m transition distance at the end
 225 of a narrow gate, at approximately 10 m s^{-1} . In the first trial, called *Single*
 226 *Actuation*, an external steering torque was imparted at the beginning of the
 227 manoeuvre. The passenger used the rod to initiate the cornering phase; the
 228 external torque returned to zero, leaving the rider alone in performing the second
 229 part of the manoeuvre. The second trial, named *Double Actuation*, was identical
 230 to the previous one in the corner entry phase; in addition to the initial steering
 231 input, the passenger applied an external steering torque to straighten the bike
 232 midway through the manoeuvre. For example, in the case of a leftward lane
 233 change, the passenger first pushed the rod to apply a clockwise¹ steering torque

¹For most riding conditions, the steering torque to be applied has a sign opposite to the

234 to make the bike lean leftward; after the roll angle became maximum, he would
235 apply anti-clockwise steering torque to make the motorcycle straighten and lean
236 to the right to set the conditions for the last part of the manoeuvre. The rider
237 could act in any manoeuvre section, independent of the external torque. In
238 particular, evaluating the rider’s reaction to the external steering action during
239 this relatively demanding transient manoeuvre was of interest.

240 At the end of both trials, the rider filled out a questionnaire to provide sub-
241 jective feedback. The questions concerned the intensity of the external steering
242 action, the controllability of such an action by an inexperienced rider during
243 everyday riding, taking back control of the motorcycle after the activation, and
244 whether he seconded or opposed the external action.

245 **3. Results**

246 *3.1. Crash Data Investigation*

247 *3.1.1. DCA*

248 Figure 4 shows the evaluation results of each Safety Function or combination
249 of SFs, regarding the number of crashes in the Prato-X database whose DCA-
250 classification received a given *applicability* score.

251 MCA received score ‘4’ (“would have applied”) in 13 cases out of 285 (4.6%).
252 Concerning the other crashes, it never received score ‘3’ (“would probably have
253 applied”) and received score ‘2’ (“would possibly have applied”) in just 2 cases
254 (0.7%). The first score class (“would definitely *not* have applied”) covered the
255 vast majority of cases (270, or 94.7%). MS was at least category 3 relevant in
256 69 cases (24.2%). MAES was at least category 3 relevant in 82 cases (28.8%).
257 MAES was the SF with the highest number of crashes classified in category 4
258 (28, or 9.8%), followed by MS (24, or 8.4%) and finally MCA (13, or 4.5%).
259 MAES would have definitely not applied in only 89 cases, or 31.2%.

yaw rate. This phenomenon is called *counter-steering* [21].

	Score (%)			
	1	2	3	4
MCA	94.7	0.7	0.0	4.6
MS	73.3	2.5	15.8	8.4
MAES	31.2	40.0	18.9	9.8
MCA + MS	71.9	1.8	12.3	13.0
MCA + MAES	26.3	40.4	18.9	14.4
MS + MAES	7.7	39.3	34.7	18.2
MCA + MS + MAES	7.7	38.2	31.2	22.8

Figure 4: The DCA results. Each row corresponds to a Safety Function (SF) or combination of SFs. Each column corresponds to an applicability score. Each cell contains the percentage of crashes in the Prato-X database in which a given SF or combination of SFs received a given applicability score.

260 Figure 4 also shows the system-relevant number of crashes that could be
261 covered by combining two or three systems. By definition, the sum of the
262 crashes classified as categories 3 and 4 for the combinations of multiple systems
263 increased compared to each SFs. In particular, the combination of the three
264 systems (MCA + MS + MAES) was category 4 relevant for 65 (22.8%) crashes,
265 which coincided with the sum of the number of crashes where each system was
266 category 4 relevant. Therefore, there was no overlap between the SFs concerning
267 this category: the SFs were complementary, and when one would have definitely
268 applied, the other two would not have. Therefore, their typical applications
269 were mutually exclusive. Including category 3, the SFs combination captured
270 154 crashes (54%), just ten less than the arithmetic sum of the results of the
271 three SFs. The highest weighted kappa value, describing inter-rater agreement,
272 was obtained by MCA (0.979), followed by MS (0.785) and MAES (0.559).

273 3.1.2. KBMS

274 Applying the KBMS method to the 2010-2012 ISTAT database, a prioritised
275 list of SFs is obtained. The higher the priority, the higher the potential to avoid
276 and mitigate the greatest possible number of motorcycle crashes in the database
277 (Italy).

278 Figure 5 shows the results: each row corresponds to a Safety Function (SF),
279 and each column to one of the nine macro-scenarios grouping the 26 crash
280 scenarios. The final result obtained by each SF, from 0 to 4, is in the rightmost

	A	B	C	D	E	F	G	H	I	System Total
MAES	0.52	0.22	0.25	0.00	0.21	0.32	0.23	0.25	0.09	2.08
MS	0.42	0.18	0.15	0.24	0.08	0.17	0.11	0.15	0.09	1.58
MCA	0.22	0.11	0.01	0.29	0.01	0.09	0.05	0.02	0.08	0.89

Figure 5: The KBMS results. Each row corresponds to a Safety Function, and each column to a crash scenario. Each cell contains the product of the average score received in that scenario and the scenario’s weight, so the SFs must be compared by columns. The three safety functions’ total scores, from 0 to 4, are in the rightmost column, corresponding to the sum of the cells on the same row. The SFs are prioritised based on their KBMS metric (larger numbers indicate greater importance).

281 column. MAES achieved the highest score (2.08), followed by MS with a 1.58
 282 score and MCA with a score of 0.89.

283 3.1.3. IDCR

284 Applying the In-Depth Crash Reconstruction method to the 19 cases from
 285 the In-SAFE database provided the results of effectiveness evaluation shown by
 286 Table 3. Each row corresponds to a Safety Function (SF) or combination of
 287 SFs. Each column corresponds to a scoring class, from 0 to 4.

288 MCA had the most crashes classified in category 4 (“excellent outcomes,
 289 assuming activation”) (4, 21%) than the sum of the other two SFs (2, 11%).
 290 Concerning the other crashes, it was placed 13 times (68%) in category 0 (“no
 291 effect”), never in categories 1 (“poor outcomes, assuming activation”), twice
 292 (11%) in category 2 (“minor outcomes, assuming activation”), and never in
 293 category 3 (“good outcomes, assuming activation”). MS obtained the worst
 294 result, with zero crashes classified as category 4 and just one (1, 5.6%) as cat-
 295 egory 3. Like MCA, MS was not relevant (category 0) for more than half of
 296 the crashes. MAES provided at least *good* outcomes (category 3 or 4) in more
 297 cases (6, 31.7%) than the other SFs combined. Furthermore, fewer cases were
 298 categorised as 0 (2 10.5%); the sum of categories 1 and 2 covered more than
 299 half of the crashes (11, 57,8%).

300 Combining more SFs led to significantly improved results. MCA and MS
 301 combination still had over half the crashes classified as category 0 (10 52.6%).

Table 3: The IDCR results. Each row corresponds to a Safety Function (SF) or combination of SFs. Each column corresponds to an effectiveness score. Each cell contains the number of crashes out of the 19 crashes from the In-SAFE database in which a given SF or combination of SFs received a given score. The corresponding frequency, in percentage, is shown in brackets

SF	Score				
	0	1	2	3	4
MCA	13 (68%)	0 (0%)	2 (11%)	0 (0%)	4 (21%)
MS	13 (68%)	3 (16%)	2 (11%)	1 (5%)	0 (0%)
MAES	2 (11%)	9 (47%)	2 (11%)	4 (21%)	2 (11%)
MCA + MS	10 (53%)	2 (11%)	2 (11%)	1 (5%)	4 (21%)
MCA + MAES	2 (11%)	6 (32%)	3 (16%)	2 (11%)	6 (32%)
MS + MAES	2 (11%)	8 (42%)	2 (11%)	5 (26%)	2 (11%)
MCA + MS + MAES	2 (11%)	6 (32%)	2 (11%)	3 (16%)	6 (32%)

302 Lastly, the combination of the three systems (MCA + MS + MAES) was at
303 least category 3 relevant for nine crashes (47,3%). As with the DCA method
304 result, there was no overlap between the SFs for category 4: each system’s
305 effectiveness was complementary to that of the other systems when one system
306 would have had excellent outcomes. There was also no overlap between SFs
307 concerning scores equal to or greater than 3 for every combination of two SFs.
308 In particular, the combination of MCA and MAES provided results analogous
309 to the combination of all the SFs.

310 3.1.4. Injury Mitigation

311 From the 19 cases included in the IDCR analysis from the In-SAFE database,
312 ten were excluded because they were unsuitable for MAES application; nine
313 were reconstructed (an example is shown in Figure 6) for the analysis concerning
314 MAES potential for injury mitigation. The nine crashes included in the analyses
315 were characterised by different crash configurations (including rear-end, vehicles
316 from adjacent directions, and manoeuvring), with a mean speed of 52.3 km/h

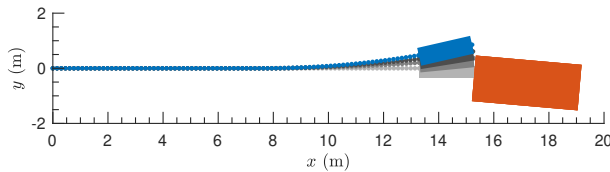


Figure 6: Comparison between PTW trajectory without MAES intervention (light grey) and simulated PTW trajectories employing three levels of MAES lateral acceleration (0.3g in medium grey, 0.5g in dark grey and 0.7g in blue), relative to the ‘ID115’ crash. The 0.7g lateral acceleration value led to avoiding the obstacle (a parked car, in orange). The corresponding animation is available in the online version of the article.

317 (SD 14.23 km/h). The time for MAES intervention used in the simulation
 318 ranged from 0.3 s to 1.2 s, according to the crash configuration (mean value
 319 0.6 s, SD 0.32 s).

320 In one case, MAES prevented the crash thanks to an avoidance manoeuvre
 321 with 0.3g of lateral acceleration, in one case with an acceleration of 0.5g, and
 322 in a third one with 0.7g. In the remaining six crashes, MAES did not prevent
 323 the crash even with 0.7g lateral deceleration but resulted in reduced relative
 324 crash speed, resulting in reduced injury risk. The calculated relative injury
 325 risk reduction for each case, calculated for MAIS2+F, MAIS3+F, and Fatal
 326 injuries, is displayed in Figure 7. The relative injury risk reduction has a wide
 327 variability among cases, but more severe injuries achieve higher values of injury
 328 risk reduction, up to 15-20%.

329 3.2. Experimental Test

330 Figure 8 presents the signals describing two runs of the Single Actuation
 331 trial. The upper subplot shows the steering torque inputs: the rider action is
 332 represented in blue; the external action is shown in orange; their sum is the
 333 resulting steering torque plotted in green. The middle subplot shows the re-
 334 sulting motorcycle lateral response in terms of roll angle (red), steering angle
 335 (purple) and yaw rate (brown). Lastly, the lower subplot shows the motorcycle
 336 trajectory during the manoeuvre, superimposed over a hypothetical roadway
 337 as a reference (lane width equal to 2.5 m, a typical value for European urban

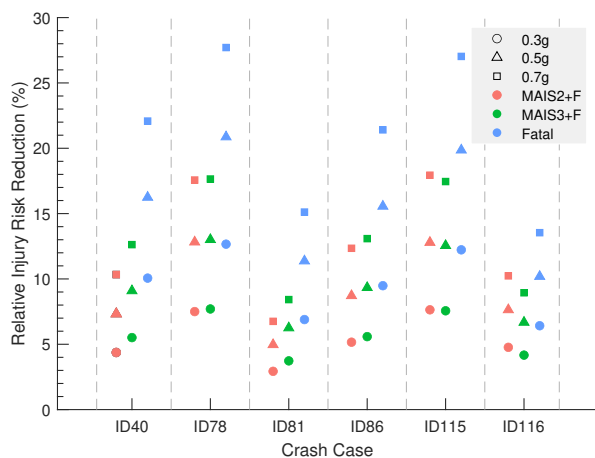


Figure 7: Calculated relative injury risk reduction due to MAES intervention for MAIS2+F, MAIS3+F and Fatal injuries for six cases (ID 40, 78, 81, 95, 115, 116) reconstructed from the In-SAFE database. For each case, the relative injury risk reduction is presented for MAES intervention characterised by 0.3g, 0.5g, and 0.7 g lateral acceleration.

338 roads). The part of the run where the rod applies a steering torque is high-
 339 lighted in yellow. Notice that the upper and middle subplots use “time since
 340 actuation” as the independent variable; in contrast, the lower subplot uses lon-
 341 gitudinal distance. As the speed is not perfectly constant during the trial, the
 342 abscissae shift slightly through each run, as can be appreciated by comparing
 343 the highlighted sections in the subplots.

344 Figure 8a, in particular, shows the very first run of the first trial (lane
 345 change towards the right). Although the external action was still declared and
 346 performed in a controlled environment, as for all the runs, this action should
 347 result in the most genuine rider reaction due to the lack of previous experience
 348 concerning this condition. The motorcycle initially travelled straight: the roll
 349 angle, steering angle and yaw rate were minimal, and the rider applied minimal
 350 steering torque to correct the small oscillations. As the external steering torque
 351 was null, the total steering torque was produced by the rider action alone. The
 352 pillion passenger then applied a positive (anti-clockwise) steering torque: the
 353 rider reacted by exerting a smaller and negative (clockwise) steering torque ac-

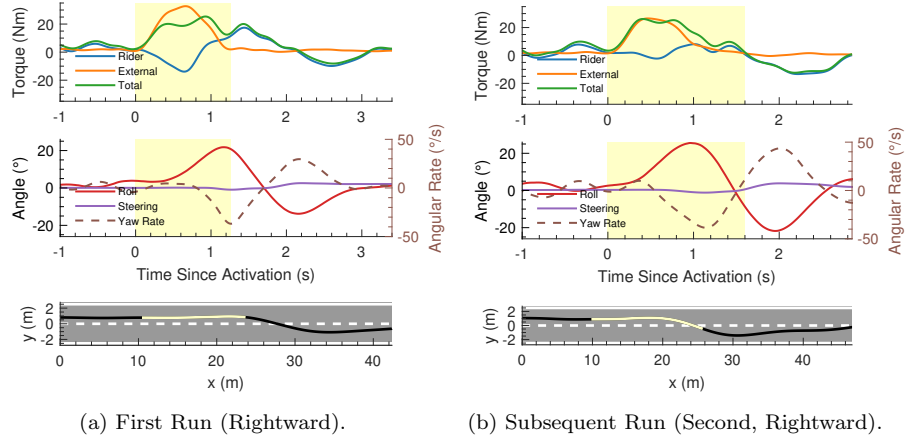


Figure 8: Steering torque inputs (top), motorcycle response signals (middle) and trajectory (bottom) during two lane changes with *single* steering actuation (on corner entry).

354 tion; the total steering torque had the same sign as that applied through the
 355 rod and initially grew with similar dynamics. Then, the rider action became
 356 more intense, while the external steering action reached its maximum: the total
 357 steering torque became perceptibly lower than that applied through the rod.
 358 The net, positive (anti-clockwise) steering torque applied made the motorcycle
 359 lean towards the right (positive roll) and turn to the right (negative yaw rate)
 360 with a clockwise (positive) steering angle. The external steering torque then
 361 decreased, reaching zero when the entity of the motorcycle response was maxi-
 362 mum. Meanwhile, the rider changed the sign of the steering torque he applied:
 363 the total steering torque was positive as in the previous part but was now due
 364 to the rider's action and not exerted through the rod. The total steering torque
 365 progressively reduced, and the motorcycle tended to straighten due to its stabil-
 366 ity properties [22]. The rider performed the second part of the lane change with
 367 no external action: he applied a negative (clockwise) steering torque to make
 368 the motorcycle lean, steer, head towards the left, and complete the manoeuvre.
 369 The motorcycle trajectory shows that the external steering torque made the
 370 motorcycle head towards the right. Its effect grew with its duration, so the

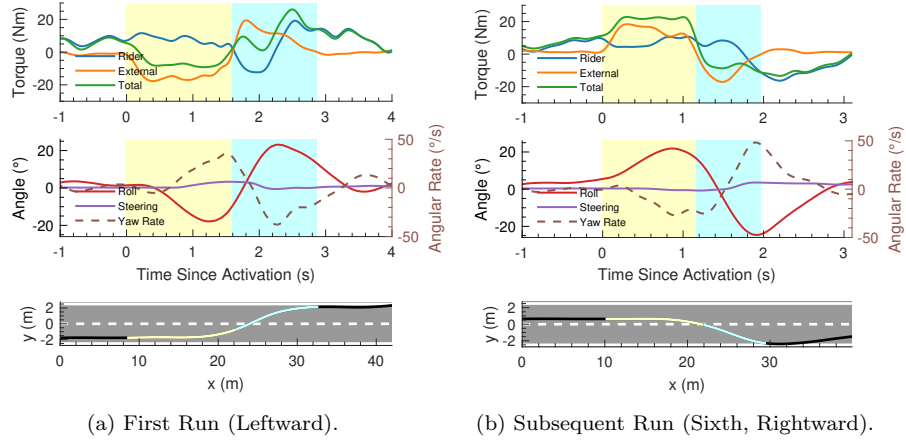


Figure 9: Steering torque inputs (top), motorcycle response signals (middle) and trajectory (bottom) during two lane changes with *double* steering actuation (on corner entry and midway through).

371 heading change became remarkable only after some time, although the torque
 372 applied was significant (exceeding 20 N m for several tenths of a second). At the
 373 end of its action, the yaw rate was maximum, so the heading of the motorcycle
 374 was changing quickly towards the right. The rider decreased the yaw rate to
 375 reduce the rate at which the maximum yaw angle was reached to then restore
 376 the null yaw angle with a shifted lateral position onto the roadway.

377 Figure 8b shows the following run. This time, the rider applied just a tiny
 378 steering torque while the passenger applied the external action: the total steering
 379 torque almost coincided with the latter contribution. The second part of the
 380 manoeuvre was similar to the previous run: the external steering action
 381 declined, making the motorcycle straighten itself; after some tenths of a sec-
 382 ond, the rider applied a negative steering torque to perform the last part of the
 383 manoeuvre and to restore the initial heading direction. In this second run, the
 384 motorcycle had more intense dynamics, with higher amplitude of the roll angle,
 385 steering angle and yaw rate produced. The maximum lateral displacement was
 386 slightly larger than in the previous run.

387 Figure 9 shows the previous quantities for two runs of the Double Actuation

388 trial. The part relative to the second external steering action is highlighted in
389 blue. Figure 9a shows the first run of the first trial (left): the external action
390 did not change the rider’s action, and the total steering torque became negative.
391 The motorcycle leaned and turned towards the left; the passenger applied a
392 second external steering action, with a sign opposite to the previous one: this
393 happened when the yaw rate and roll angle were close to their maximum values.
394 The sudden change of the external steering torque (from ≈ -20 N m to ≈ 20 N m)
395 produced a sign change of the rider’s steering torque; the total steering torque
396 became positive. The effect of this second external steering action was to change
397 the signs of the signals describing the motorcycle response. The external steering
398 torque was then removed, and the rider performed the last part of the manoeuvre
399 restoring the initial heading direction. The total lateral displacement during the
400 manoeuvre was significant, around 4 m.

401 Figure 9b shows a subsequent run (the sixth, towards the right) of the same
402 trial. In this run, in the corner entry phase, the rider applied a steering torque
403 with the same sign as the external steering torque: consequently, the total
404 steering torque was higher than both contributions. The passenger then changed
405 the sign of the steering torque he applied, making the total torque change sign
406 even though the rider’s steering action did not change for a few tenths of a
407 second. As the external steering torque became less negative, the rider applied
408 a growing negative contribution keeping the total torque approximately constant
409 in the last part of the manoeuvre.

410 A summary of the experimental results is provided by Table 4 for the *single*
411 *actuation* trial and Table 5 for the *double actuation* trial. Each table reports the
412 maximum values of the lateral acceleration, external steering torque, roll angle
413 and lateral displacement during the entry phase of each run of the corresponding
414 trial, along with the mean and standard deviation of each. The external steering
415 torque reached high values on average (24.7 N m in the single actuation trial and
416 20.0 N m in case of double actuation), producing moderate lateral acceleration
417 values (0.425 g and 0.425 g, respectively). Test repeatability was high: the
418 lateral acceleration produced had a modest standard deviation (0.031 g and

Table 4: Maximum values of the lateral acceleration, external steering torque, roll angle and lateral displacement during the entry phase of each run of the *Single Actuation* trial. The mean and standard deviation are in bold.

Run	Maximum			
	a_y (g)	τ_{ext} (N m)	ϕ ($^\circ$)	Δy (m)
1	0.396	32.7	21.8	2.33
2	0.424	26.5	25.5	2.67
3	0.403	25.7	24.6	3.04
4	0.383	31.8	23.2	2.88
5	0.476	17.2	21.2	3.36
6	0.428	21.4	20.0	2.97
7	0.435	18.7	19.5	3.25
8	0.457	23.3	19.6	5.08
Mean	0.425	24.7	21.9	3.20
SD	0.031	5.7	2.3	0.83

419 0.038 g, respectively). The lateral displacement produced was, on average,
 420 3.2 m in the case of single actuation and 3.7 m when the actuation was double.

421 Concerning the survey, the question ‘how intense do you think the action
 422 on the handlebars was? (0: Very low intensity, 10: very high intensity)’ was
 423 answered ‘6-7’ in both trials, indicating a moderate-high intensity. ‘If such a
 424 trigger occurred during a real lane change manoeuvre, would an inexperienced
 425 driver be able to maintain control? (0: they would not, 10: they easily would)’
 426 was answered ‘6’ after both trials, meaning that the rider would probably do it
 427 albeit with effort. To the question ‘At the end of the activation, were you able
 428 to regain control of the motorbike? (0: I was not, 10: I did it easily)’, the rider
 429 answered ‘7’ in case of single activation and ‘8-9’ in case of double activation.
 430 Lastly, he answered ‘8’ in both trials to the question ‘During the activation, did
 431 you second the external action or oppose it? (0: I completely opposed it, 10: I

Table 5: Maximum values of the lateral acceleration, external steering torque, roll angle and lateral displacement during the entry phase of each run of the *Double Actuation* trial. The mean and standard deviation are in bold.

Run	Maximum			
	a_y (g)	τ_{ext} (N m)	ϕ ($^\circ$)	Δy (m)
1	0.415	17.7	17.9	4.54
2	0.400	18.1	19.6	4.26
3	0.432	21.2	20.1	3.55
4	0.396	20.3	19.7	4.78
5	0.316	23.1	18.2	2.42
6	0.371	18.4	21.9	4.06
7	0.351	20.3	20.2	2.21
8	0.411	20.8	22.2	3.75
Mean	0.387	20.0	20.0	3.70
SD	0.038	1.8	1.5	0.94

432 completely seconded it)', meaning that he definitely seconded it.

433 4. Discussion

434 4.1. Crash Data Investigation

435 The investigation, conducted through the three investigation methods (DCA,
436 KBMS, IDCR), aimed to evaluate the potential benefits, concerning crash avoid-
437 ance or mitigation, of steering assistance for motorcycles when applied to real
438 crash scenarios. The hypothetical impact of these systems on road safety was
439 evaluated concerning *applicability* and *effectiveness*.

440 The DCA method showed that MAES might be the most applicable sys-
441 tem, with the most crashes covered by categories 3 and 4, followed by MS and
442 MCA. The latter was not applicable for a consistent number of crashes (270,
443 or 94.7%). This result, however, was coherent with the characteristics of the
444 database used in the study: crashes in bends were underrepresented due to the
445 urban context considered (Prato municipality). For the same reason, MAES was
446 the most applicable SF because changing the trajectory to avoid an obstacle was
447 more compatible with crashes involving other vehicles, which are the majority
448 of the crash scenarios in urban areas. However, PTWs are more often subject
449 to these crashes, particularly the “looked-but-failed-to-see” case, because they
450 are smaller and less visible than cars. The result obtained by the combina-
451 tion of the three systems is noteworthy: it was deemed inapplicable only in a
452 few cases (22, or 7.7%), while the majority of the cases were covered by cate-
453 gory 2 (“Would possibly have applied”, controversial), and category 3 (“Would
454 probably have applied”, technical challenges still need to be solved). The SFs
455 considered were complementary, and when one ‘would have applied’, the other
456 two would not have had the same degree of applicability. This result is remark-
457 able because the three systems would be based on the same hardware, leading to
458 sharing the cost of implementation while adding up the benefits. Advancements
459 in technology concerning obstacle detection and the control logic and simula-
460 tion or experimental campaigns could reduce the uncertainty concerning this

461 system. According to Landis and Koch [23], the kappa value obtained by MCA
462 (0.979) can be interpreted as ‘almost perfect inter-rater agreement’; that for MS
463 (0.785) as ‘substantial agreement’, and the one of MAES (0.557) as ‘moderate
464 agreement’. The strength of agreement for each system reflects the different
465 applicability characteristics of each system: MCA typical application is more
466 focused (it only covers bends); instead, MAES has broader applications (it can
467 cover many types of collision); MS is in between the two, as it can also apply
468 outside of bends but it is not as general as MAES. The more focused the scope
469 of a system, the easier it was for the examiners to give an applicability rating.

470 The concept of applicability describes the number of crashes the system cov-
471 ers; however, it gives no information about the effects in terms of mitigation or
472 avoidance. These aspects are covered by the other key concept of this investiga-
473 tion, effectiveness, evaluated through the KBMS method. The advantage of the
474 KBMS is that it provides a quantitative metric which allows one to interpret the
475 results and rank the systems directly. MAES was, again, the SF with the best
476 score (2.08), followed by MS (1.58) and MCA (0.89). Thus, MAES was rated
477 more important than MS and over twice as influential as MCA. The database
478 used is the same as in Gil et al. [10], as is the way the KBMS was applied as
479 well². Therefore, the KBMS metrics for the three SFs considered in the current
480 article can be compared to the 10 SFs evaluated by Gil et al. for a total of
481 13 SFs. MAES ranked 6th, MS 7th and MCA 11th. MAES SF was about as
482 effective as the SF that restricts the speed of the PTW to the legal limit (2.16,
483 5th) and more than the SF that dissipates the rider’s kinetic energy during a
484 crash (1.51, 8th). These systems were less effective on average than the SFs
485 acting on braking evaluated by Gil et al. [10]. Although the different pool of
486 experts could impact the outcome, this result was coherent with the assumption
487 that a braking action implies a vehicle speed reduction and so injury reduction,
488 as predicted by the injury risk functions. In fact, the two systems that aimed at
489 reducing the speed significantly got the two best scores, close to the theoretical

²The panel of experts is different, potentially influencing the results

490 maximum. The SFs proposed by the authors of this article do not change the
491 vehicle speed, with the possible exception of MCA. Therefore, these systems are
492 more suitable for avoidance or prevention than mitigation, as mitigation would
493 primarily be caused by a different impact angle and relative speed. The rank
494 got by the systems like MAES was coherent with the other SFs acting mainly
495 on prevention, like restricting the vehicle's maximum speed, alerting the rider
496 of an oncoming vehicle, and sending a signal to slow/stop the other vehicle.

497 The IDCR method allowed testing of whether the systems would impact
498 real-world crashes. MCA obtained better coverage in the highest class but was
499 inapplicable for most scenarios, as shown through the DCA assessment. Indeed,
500 it is a system conceived to perform a particular task. MAES received the fewest
501 instances of the lowest effectiveness score: this result was coherent with the
502 fact that this system might intervene to modify the trajectory; however, the
503 forecast and application are challenging. MS obtained the worst result than
504 the other systems, while in the other investigations, it consistently scored above
505 MCA. This evidence could be explained by the fact that the pictograms used in
506 the KBMS were not sufficiently detailed to represent the cause of the possible
507 loss of control: in the DCA scenarios, the loss of control was often specified as
508 the 'scenario' variable; this was not the case for the KBMS. In both DCA and
509 IDCR datasets, the crashes collected occurred in the urban context. This bias
510 influences both the type of collisions (sideswipe, Crossing, cut PTW off) and
511 the type of PTW involved in the crash, more moped, which often lacks ABS.
512 Locking up the rear wheel in response to an imminent collision was the leading
513 cause of instability, not oscillatory mode or external perturbation. Thus, an
514 intervention on the lateral dynamics could not be correctly performed by MS.

515 In conclusion, MAES obtained the best results (1st, 1st, 2nd) on two out
516 of three investigations (DCA, KBMS, IDCR); MS was evaluated second at best
517 (2nd, 2nd, 3rd) and MCA has the best result in one method (3rd, 3rd, 1st).
518 Thus, MCA could perform a specific task very well, but it was not suitable for
519 the majority of cases; MS could apply in more scenarios but with poor or even
520 negligible effects (as was the case in the IDCR investigation); finally, MAES was

521 the most applicable and had good or relevant effects on the crash scenario.

522 *4.2. Injury Mitigation*

523 The analysis aiming to investigate the effectiveness of MAES intervention
524 in reducing injury risks in different crash configurations finally included nine
525 crashes reconstructed in a simplified 2D simulation environment, using time for
526 MAES intervention ranging from 0.3 s to 1.2 s, depending on the crash configura-
527 tion. The results indicate that MAES intervention successfully prevented three
528 crashes by implementing an avoidance manoeuvre with lateral accelerations of
529 0.3g, 0.5g, and 0.7g, respectively. In the remaining six crashes, MAES did not
530 prevent the crash despite 0.7g lateral deceleration, but it did reduce the relative
531 crash speed, thereby reducing injury risk. The relative injury risk reduction
532 varied widely among cases but reached high values of injury risk reduction (up
533 to 20% for severe and fatal injuries).

534 Although the crashes simulated in this study are genuine and realistic, they
535 constitute only a small sample size. Thus, the outcomes obtained lack statistical
536 significance and cannot be used as a robust estimate of MAES's capability to
537 mitigate injuries. Nonetheless, a non-random sample demonstrates that there
538 are real-world crashes where MAES can avert severe or fatal crashes, even when
539 using conservative time for intervention (similar to that considered for the au-
540 tonomous braking system [8]) and moderate lateral accelerations.

541 These findings suggest that MAES intervention may effectively reduce in-
542 juries in different crash configurations; however, its success may depend on
543 factors such as the type of crash, time for intervention, and lateral acceleration
544 implemented. The findings also highlight the importance of implementing such
545 interventions in time to prevent crashes or mitigate their severity. Further re-
546 search is needed to investigate the potential of MAES intervention in reducing
547 injuries using detailed crash reconstructions (which can also account for varia-
548 tions of the point of impact) and a comprehensive sample of cases to achieve
549 statistical significance.

550 *4.3. Experimental Test*

551 The experiment evaluated the feasibility of changing the motorcycle's state
552 of motion through external steering actions. The external steering torque ap-
553 plied was significant, often reaching 20 N m, and was applied for longer than a
554 second. Consequently, the motorcycle response was pronounced, with the roll
555 angle exceeding 20°. The high external steering torque was also applied when
556 the roll angle and yaw rate were close to their maximum values, as in the ter-
557 minal part of the yellow segment in Figures 8,9. No instability phenomena were
558 detected in the acquired data, nor were they underlined by the rider at the end
559 of the experiment.

560 The value and duration of the total steering torque determined the motor-
561 cycle response, independent of the value of the single contributions (due to the
562 rider and the rod). However, when interpreting the results, one cannot neglect
563 how the two combine, for example, whether the rider strongly opposes the ex-
564 ternal steering action significantly, if they are indifferent to it, or if they even
565 second it. An active assistance device acts together with the human controller,
566 and it must be compatible with the rider's action to be effective and not dan-
567 gerous. In the case of this study, the rider's and external torques were exerted
568 in parallel as in a power steering system. During the tests, the rider either mod-
569 erately opposed (as in Figures 8a, 9a) or was indifferent to the external action
570 (as shown by Figure 8a). In one instance shown (Figure 9b, entry section), he
571 applied a steering torque concordant with the external one, producing a very
572 high total steering torque. Compared to the previous instants, one can also
573 notice that the external steering torque shifts the rider's steering torque that
574 opposes the external action.

575 In particular, the rider acts both as a dynamical system, with its specific
576 inertia, damping and stiffness properties, and as a controller with physiologi-
577 cal limits on the forces they can apply, the movement speeds they can reach,
578 and the time required to sense a change in the state [24]. Combining the two
579 aspects should explain what is seen at the beginning of the first run (Figure
580 8a). When the positive external steering torque is applied, the rider's action be-

581 comes negative, growing with a slope that is a fraction of the one of the external
582 torque. Therefore, the resulting steering torque grows similarly to the external
583 steering torque, albeit with a smaller derivative. This fact is probably the effect
584 of the stiffness of the rider's arms: the positive (anti-clockwise) external steer-
585 ing torque pushes the left handle against his hand and pulls the right handle
586 from his other hand. This action produces a reactive, negative rider steering
587 torque proportional to the external action. Around 0.2s after the beginning,
588 this relationship breaks up: the total steering torque has a dynamics different to
589 the external steering action, as the rider's steering torque is now growing faster
590 than the external steering torque. In this phase, the rider probably sensed the
591 change in motorcycle motion and reacted by applying an additional conscious
592 effort to impose the total steering torque. One can compare it to the next run
593 (Figure 8b), where the rider's action in the entry phase is much tinier: in the
594 very first run, the rider was probably more concerned about the consequences of
595 the external action, so he held the handlebar more tightly, producing a higher
596 reactive torque. After the first run, his action following the external steering
597 torque was much more modest, as shown in all the other runs.

598 A steering action requires time to generate tangible results: the steering
599 torque produces a yaw rate, which must be maintained through time to gen-
600 erate a change in the yaw angle and, at last, a lateral displacement over the
601 roadway. Therefore, a steering assistance device should apply a steering ac-
602 tion soon enough to change the motorcycle's state and guide the rider towards
603 the correct evasive action. The motorcycle considered, which had its inertial
604 properties influenced by the presence of the pillion passenger, was self-stable at
605 the speed of the tests: removing or even reducing the steering torque led to a
606 straightening of the vehicle. This behaviour benefits the system's safety: even
607 if the rider does not apply a steering action after the external steering torque
608 ceased, he would not fall. This phenomenon is generally true for most motorcy-
609 cles in wide speed ranges [22]. In particular, motorcycles tend to be unstable at
610 low enough speeds; however, as swerving becomes more effective than braking
611 at high enough speeds [9], such a system would apply in place of an autonomous

612 braking system only starting from medium speeds. A successful lane change re-
613 quires restoring the initial heading while bringing all the dynamical states back
614 to zero: this is achieved by applying a total steering torque having the opposite
615 sign to the one used to start the manoeuvre, which can be left to the rider (Sin-
616 gle Actuation trial) or assisted by an external action (Double Actuation trial).
617 The motorcycle does not have a clearly distinct behaviour in the second part
618 of the manoeuvre in the case of the Double Actuation runs compared to those
619 of the Single Actuation trial, apart from slightly less smooth dynamics of the
620 yaw rate. The test runs were consistent, with modest variation in the external
621 steering torque inputs and the consequent motorcycle response. In each of the
622 16 runs conducted, the external action produced a lateral acceleration higher
623 than the lowest value (0.3 g) considered in the study on injury mitigation. This
624 value was sufficient to avoid one of the nine crashes considered. As the inter-run
625 variability was modest, the four lane changes shown are descriptive of the whole
626 experimental test.

627 The survey showed that, although the rider confirmed the moderately high
628 intensity of the external action, he seconded it. In a real scenario, he thought
629 an inexperienced rider would probably be able to maintain control, even though
630 they would find it demanding. Only one question received a different answer
631 depending on the trial: he found it easy to regain control in the case of single
632 activation and very easy in the case of double activation. The rider preferred the
633 external action to continue throughout the manoeuvre instead of terminating
634 in the middle of it.

635 The experiment showed the feasibility of changing the lateral motorcycle
636 dynamics through external steering actions, albeit in a controlled environment.
637 Although straightforward, the test constitutes a first step towards experimen-
638 tally testing the compatibility of steering assistance systems with a real rider,
639 and the resulting pieces of evidence look promising.

640 5. Conclusions

641 Active steering assistance systems for powered two-wheelers have yet to be
642 studied extensively; however, they have the potential to be highly effective in
643 preventing and mitigating motorcycle crashes while complementing the well-
644 researched brake assist systems. For the first time, this study presented an
645 exploratory assessment of such systems. This preliminary analysis indicates
646 that the three systems we proposed - MCA, MS, and MAES - are applicable
647 in different emergency scenarios and are complementary, responding well to
648 different situations.

649 Among the three systems, MAES appears to have the highest potential ben-
650 efits, with good estimated applicability across a wide range of emergency scenar-
651 ios and promising estimated effects in reducing injuries and preventing crashes.
652 This evidence motivated us to conduct exploratory field trials: remarkably, ap-
653 plying a superimposed steering action to produce a lateral avoidance manoeuvre
654 was easily manageable by a real rider. These findings highlight the potential
655 of active steering assistance systems to enhance motorcycle safety, potentially
656 fostering further research in this area.

657 CRediT authorship contribution statement

658 **Mirco Bartolozzi:** Conceptualisation, Methodology, Software, Validation,
659 Investigation, Writing - Original Draft, Writing - Review & Editing, Visualisa-
660 tion, Project administration. **Adelmo Niccolai:** Conceptualisation, Method-
661 ology, Software, Validation, Formal analysis, Investigation, Writing - Original
662 Draft, Writing - Review & Editing, Visualisation. **Cosimo Lucci:** Software,
663 Resources, Writing - Original Draft, Writing - Review & Editing. **Giovanni**
664 **Savino:** Conceptualisation, Writing - Review & Editing, Supervision, Formal
665 analysis, Project administration.

666 **Declaration of Competing Interest**

667 The authors declare that they have no known competing financial interests or
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679 **Data availability**

680 The data that support the findings of this study are available from the
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682 **Appendix A. Flow Chart for Applicability Evaluation**

683 **References**

- 684 [1] M. Rizzi, A. Kullgren, C. Tingvall, The combined benefits of motorcycle
685 antilock braking systems (ABS) in preventing crashes and reducing crash
686 severity, *Traffic Injury Prevention* 17 (2016) 297 – 303.

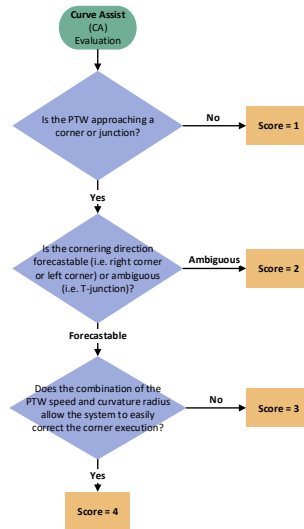


Figure A.1: Flow chart used to evaluate the applicability of the *Curve Assist* Safety Function.

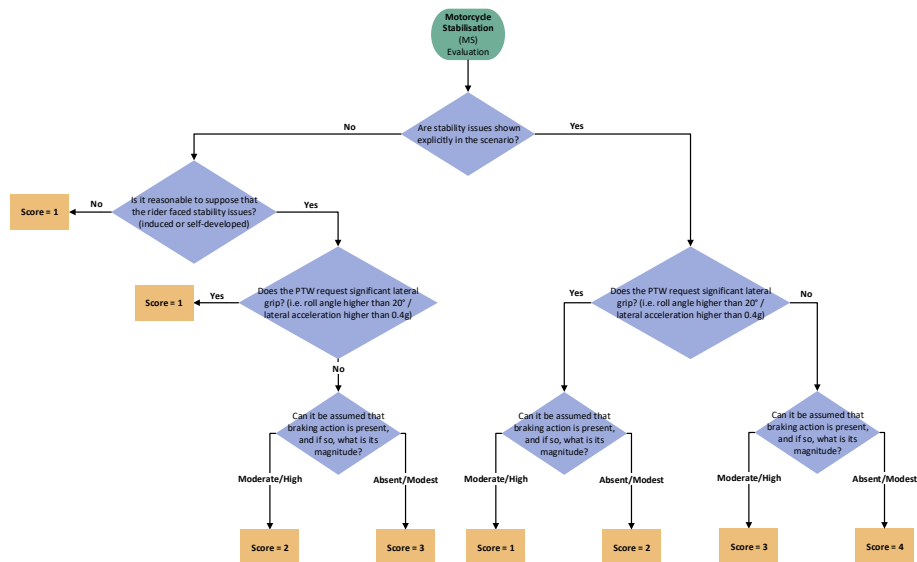


Figure A.2: Flow chart used to evaluate the applicability of the *Motorcycle Stabilisation* Safety Function.

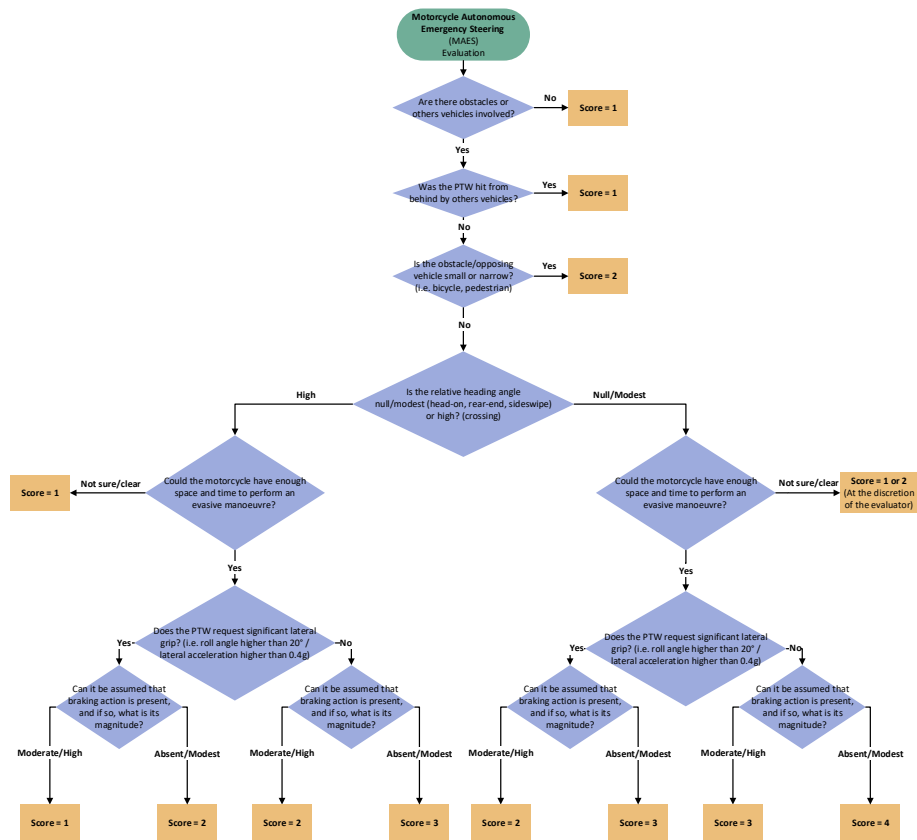


Figure A.3: Flow chart used to evaluate the applicability of the *Motorcycle Autonomous Emergency Steering* Safety Function.

- 687 [2] M. C. Rizzi, M. Rizzi, A. Kullgren, B. Algurén, The potential of dif-
688 ferent countermeasures to prevent injuries with high risk of health loss
689 among bicyclists in Sweden, *Traffic Injury Prevention* 21 (3) (2020) 215–
690 221. doi:10.1080/15389588.2020.1730827.
- 691 [3] T. Lich, W. G. Block, S. N. Prashanth, B. Heiler, Motorcycle stability
692 control - the next generation of motorcycle safety and riding dynamics,
693 *SAE International journal of engines* 9 (2015) 491–498.
- 694 [4] L. Beck, A. Dellinger, M. O’Neil, Motor vehicle crash injury rates by
695 mode of travel, united states: Using exposure-based methods to quan-
696 tify differences, *American journal of epidemiology* 166 (2007) 212–8.
697 doi:10.1093/aje/kwm064.
- 698 [5] G. Savino, R. Lot, M. Massaro, M. Rizzi, I. Symeonidis, S. Will,
699 J. Brown, Active safety systems for powered two-wheelers: A
700 systematic review, *Traffic Injury Prevention* 21 (1) (2020) 78–86.
701 doi:10.1080/15389588.2019.1700408.
- 702 [6] P. Terranova, M. Dean, C. Lucci, S. Piantini, T. Allen, G. Savino,
703 H. Gabler, Applicability assessment of active safety systems for mo-
704 torcycles using population-based crash data: Cross-country compari-
705 son among Australia, Italy, and USA, *Sustainability* 14 (2022) 7563.
706 doi:10.3390/su14137563.
- 707 [7] C. Lucci, T. Allen, M. Pierini, G. Savino, Motorcycle Autonomous Emer-
708 gency Braking (MAEB) employed as enhanced braking: Estimating the
709 potential for injury reduction using real-world crash modeling, *Traffic In-*
710 *jury Prevention* 22 (2021) S104 – S110.
- 711 [8] C. Lucci, G. Savino, N. Baldanzini, Does motorcycle autonomous emer-
712 gency braking (MAEB) mitigate rider injuries and fatalities? Design of
713 effective working parameters and field test validation of their acceptabil-
714 ity, *Transportation Research Part C: Emerging Technologies* 145 (2022)
715 103865. doi:https://doi.org/10.1016/j.trc.2022.103865.

- 716 [9] F. Giovannini, G. Savino, M. Pierini, N. Baldanzini, Analysis of the
717 minimum swerving distance for the development of a motorcycle au-
718 tonomous braking system, *Accident Analysis & Prevention* 59 (2013) 170–
719 184. doi:10.1016/j.aap.2013.05.020.
- 720 [10] G. Gil, G. Savino, S. Piantini, N. Baldanzini, R. Happee, M. Pierini, Are
721 automatic systems the future of motorcycle safety? A novel methodology to
722 prioritize potential safety solutions based on their projected effectiveness,
723 *Traffic Injury Prevention* 18 (2017) 877 – 885.
- 724 [11] C. Lucci, T. Allen, N. Baldanzini, G. Savino, Motorcycle curve as-
725 sist: A novel approach based on active speed control for crash in-
726 jury reduction, *Traffic Injury Prevention* 23 (sup1) (2022) S56–S61.
727 doi:10.1080/15389588.2022.2106370.
- 728 [12] VicRoads, DCA: VicRoads 2013 Crash Stats user guide - Road Crash
729 Statistics - Victoria, 2013 Edition, Standard, VicRoads (2013).
- 730 [13] G. Savino, M. Pierini, M. Fitzharris, Motorcycle active safety sys-
731 tems: Assessment of the function and applicability using a population-
732 based crash data set, *Traffic Injury Prevention* 20 (2019) 1–7.
733 doi:10.1080/15389588.2019.1594795.
- 734 [14] J. Cohen, A coefficient of agreement for nominal scales, *Educational and*
735 *Psychological Measurement* 20 (1960) 37 – 46.
- 736 [15] J. Cohen, Weighted kappa: nominal scale agreement with provision for
737 scaled disagreement or partial credit., *Psychological bulletin* 70 4 (1968)
738 213–20.
- 739 [16] S. Piantini, D. Grassi, M. Mangini, M. Pierini, G. Zagli, R. Spina, A. Peris,
740 Advanced accident research system based on a medical and engineering
741 data in the metropolitan area of florence, *BMC emergency medicine* 13
742 (2013) 3. doi:10.1186/1471-227X-13-3.

- 743 [17] C. Ding, M. Rizzi, J. Strandroth, U. Sander, N. Lubbe, Motorcyclist injury risk as a function of real-life crash speed and other contributing factors, *Accident Analysis & Prevention* 123 (2019) 374–386.
744
745 doi:<https://doi.org/10.1016/j.aap.2018.12.010>.
746
- 747 [18] E. W. Thomas A. Gennarelli, Abbreviated Injury Scale © - Update 2008, Standard, Association for the Advancement of Automotive Medicine
748 (2008).
749
- 750 [19] M. Bartolozzi, G. Savino, M. Pierini, Motorcycle steering torque estimation using a simplified front assembly model: experimental validation and manoeuvrability implications, *Vehicle System Dynamics* (2023 (Forthcoming)).
751
752
753
- 754 [20] Road vehicles - vehicle dynamics and road-holding ability - vocabulary, Standard, International Organization for Standardization (2011).
755
- 756 [21] V. Cossalter, J. Sadauckas, Elaboration and quantitative assessment of manoeuvrability for motorcycle lane change, *Vehicle System Dynamics* 44 (12)
757 (2006) 903–920.
758
- 759 [22] V. Cossalter, R. Lot, M. Massaro, An advanced multibody code for handling and stability analysis of motorcycles, *Meccanica* 46 (2010) 943–958.
760 doi:[10.1007/s11012-010-9351-7](https://doi.org/10.1007/s11012-010-9351-7).
761
- 762 [23] J. R. Landis, G. G. Koch, The measurement of observer agreement for categorical data., *Biometrics* 33 1 (1977) 159–74.
763
- 764 [24] J. Kooijman, A. L. Schwab, A review on bicycle and motorcycle rider control with a perspective on handling qualities, *Vehicle System Dynamics* 51
765 (2013) 1722 – 1764.
766