

1 **DEVELOPMENT OF A NEW STEADY STATE ZERO-DIMENSIONAL SIMULATION MODEL FOR**
2 **WOODY BIOMASS GASIFICATION IN A FULL SCALE PLANT**

3
4 **Marco Formica^a, Stefano Frigo^b, Roberto Gabbrielli^{c*}**

5 ^a Dipartimento di Ingegneria dell'Energia, dei Sistemi, del Territorio e delle Costruzioni - Università di Pisa,
6 Largo L. Lazzarino, 56126 Pisa (Italy), email: marco.formica11@gmail.com

7 ^b Dipartimento di Ingegneria dell'Energia, dei Sistemi, del Territorio e delle Costruzioni - Università di Pisa,
8 Largo L. Lazzarino, 56126 Pisa (Italy), email: s.frigo@ing.unipi.it

9 ^c Dipartimento di Ingegneria Civile e Industriale - Università di Pisa, via Bonanno Pisano, 25/b, 56126 Pisa (Italy),
10 phone: +39-050-2217138 email: r.gabbrielli@ing.unipi.it

11 * corresponding author

12
13 **ABSTRACT**

14 A new steady state zero-dimensional simulation model for a full-scale woody biomass
15 gasification plant with fixed-bed downdraft gasifier has been developed using Aspen Plus®.
16 The model includes the technical characteristics of all the components (gasifier, cyclone,
17 exchangers, piping, etc.) of the plant and works in accordance with its actual main control
18 logics. Simulation results accord with those obtained during an extensive experimental
19 activity. After the model validation, the influence of operating parameters such as the
20 equivalent ratio, the biomass moisture content and the gasifying air temperature on syngas
21 composition have been analyzed in order to assess the operative behavior and the energy
22 performance of the experimental plant. By recovering the sensible heat of the syngas at the
23 outlet of the gasifier, it is possible to obtain higher values of the gasifying air temperature and
24 an improvement of the overall gasification performances.

25

26 **KEYWORDS**

27 Downdraft gasifier, Biomass gasification, Steady state simulation, Aspen Plus[®], Experimental
28 activity

29

30 **NOMENCLATURE**

31 CGE [-]: cold gas efficiency

32 C_{p_a} [J/kg K]: specific heat of the wind air outside of the gasifier

33 C_{p_i} [J/kg K]: specific heat of the air/syngas within chipped biomass bed

34 $D_{e_insulation}$ [m]: external diameter of the ceramic fiber insulation

35 $D_{e_refractory}$ [m]: external diameter of the protective refractory layer

36 D_{e_shell} [m]: external diameter of the reactor shell

37 D_i [m]: internal diameter of the protective refractory layer

38 d_p [m]: mean equivalent diameter of the chipped biomass that is supposed as sphere

39 E_m [-]: the emissivity of the cover surface of the external thermal insulation of the gasifier

40 ER [-]: equivalent ratio

41 k_a [W/m K]: conductivity of the wind air outside the gasifier

42 k_i [W/m K]: conductivity of the air/syngas within chipped biomass bed

43 $k_{insulation}$ [W/m K]: conductivity of the ceramic fiber insulation

44 $k_{refractory}$ [W/m K]: conductivity of the refractory layer

45 k_{shell} [W/m K]: conductivity of the shell

46 L [m]: length of the reactor

47 l [m]: height of the chipped biomass bed within the gasifier

48 LHV [kJ/kg]: lower heating value

49 LHV_b [kJ/kg]: lower heating value of biomass

50 LHV_s [kJ/kg]: lower heating value of the syngas

51 MC [-]: moisture content

52 \dot{m}_b [kg/s]: biomass mass flow

53 \dot{m}_s [kg/s]: syngas mass flow

54 m_{a_a} [kg/s]: actual gasifying air mass flow

55 m_{a_s} [kg/s]: stoichiometric gasifying air mass flow

56 Nu_a [-]: Nusselt number for the convective heat exchange between the wind air and the cover
57 surface of the external thermal insulation of the gasifier

58 Nu_i [-]: Nusselt number for the convective heat exchange between the air/syngas and the
59 internal surface of the refractory layer of the gasifier

60 Pr_a [-]: Prandtl number of the wind air outside of the gasifier

61 Pr_i [-]: Prandtl number of the air/syngas within chipped biomass bed

62 \dot{Q} [W]: thermal power that is dispersed by the gasifier into the environment

63 R_{c1} [K/W]: conductive thermal resistance of the internal refractory layer

64 R_{c2} [K/W]: conductive thermal resistance of the gasifier shell

65 R_{c3} [K/W]: conductive thermal resistance of the external thermal insulation of the gasifier
66 shell

67 R_e [K/W]: thermal resistance of the convective heat exchange between the wind air and the
68 cover surface of the external thermal insulation of the gasifier shell

69 Re_a [-]: Reynolds number of the wind air outside of the gasifier

70 Re_i [-]: Reynolds number of the air/syngas within chipped biomass bed

71 R_i [K/W]: thermal resistance of the convective heat exchange between the air/syngas and
72 the internal surface of the refractory layer of the gasifier

73 R_r [K/W]: equivalent thermal resistance of the radiative heat exchange between the cover
74 surface of the external thermal insulation of the gasifier shell and the environment
75 R_{tot} [K/W]: total thermal resistance from the reactor core to the environment
76 T_e [K]: environment temperature
77 T_p [K]: the temperature of the cover surface of the external thermal insulation of the gasifier
78 T_r [K]: mean temperature of air/syngas within the reactor
79 u_a [m/s]: velocity of the wind air outside of the gasifier
80 u_i [m/s]: mean velocity of the air/syngas across the chipped biomass bed within the gasifier
81 *Greek symbols*
82 ΔP [Pa]: pressure drop of the air/syngas across the gasifier
83 ε [-]: mean porosity of the chipped biomass bed within the gasifier
84 μ_a [kg/m s]: dynamic viscosity of the wind air outside of the gasifier
85 μ_i [kg/m s]: dynamic viscosity of the air/syngas across the chipped biomass bed within the
86 gasifier
87 ρ_a [kg/m³]: density of the wind air outside of the gasifier
88 ρ_i [kg/m³]: density of the air/syngas across the chipped biomass bed within the gasifier
89 σ [W/m² K⁴]: the Boltzmann constant
90

91 **1. INTRODUCTION**

92 Recently the growing awareness of the shortage of the traditional energy sources and the
93 concern for environmental protection have encouraged the wider use of renewable energy
94 sources. Among these, biomass is certainly one of the most important because of its
95 inexhaustibility and wide availability. In addition, more than wind and photovoltaic, energy
96 conversion of biomass can create concrete local economic opportunities.

97 The exploitation of energy ~~from~~ through biomass comes off bio-chemical and thermo-
98 chemical processes [1]. Bio-chemical process involves biomethanization of biomass,
99 characterized by low cost effectiveness and efficiency. Actually, the three main thermo-
100 chemical processes are combustion, pyrolysis and gasification. Combustion, apart from the
101 applications in small fireplaces and stoves, is used mainly to supply heat and power ~~with~~ by
102 means of large scale systems (typically above 500 kW_e), and the net efficiency for electricity
103 generation is usually very low and ranges from 15-20 % for the smallest plants (< 1 MW_e) [2].
104 Pyrolysis converts biomass to bio-fuels and bio-char in absence of oxygen (O₂), but the
105 application of this technology is limited due to the thermal system complexity and the low
106 quality of the fuels that are produced. Gasification [3] converts biomass through a partial
107 oxidation into a gaseous mixture, called *syngas*, and represents, especially in the low power
108 range (< 500 kW_e), the process with the greatest development prospects mainly for its high
109 electric efficiency (20-25 %) [4-5]. Other advantages of gasification are the plant simplicity and
110 the lower capital cost for small scale applications with respect to other technologies. The main
111 drawback is represented by the syngas cleaning system complexity and efficiency.

112 The development of numerical simulation models is an important tool in order to provide
113 more accurate qualitative and quantitative information on biomass gasification. The possible
114 approaches for the modelling of the gasification process are: steady state models, transient
115 state models and models based on the computational fluid dynamics. The steady state
116 models, that do not consider the time derivatives, are further classified as kinetic rate models
117 and kinetics free equilibrium models [6-9]. For the evaluation of the syngas composition and
118 temperature as function of the process parameters, the kinetics free equilibrium models are
119 the most preferred models because they are very simple and reliable. They have the inherent
120 advantage of being generic but, at the same time, they have thermodynamic limitations, even

121 though researchers have successfully demonstrated that this approach describes sufficiently
122 well the gasification process in downdraft gasifiers [10-13].

123 A commercial code, such as Aspen Plus®, can be usefully and effectively adopted for the
124 construction of a reliable kinetic free equilibrium simulation model. This article aims at
125 presenting an innovative simulation approach, where the whole experimental gasification
126 plant, containing all the elements such as cyclone, heat exchangers, turbomachineries etc.,
127 works following the main control logics of the real plant. Besides, it gives an experimental
128 contribution to the validation of a zero-dimensional steady state simulation model of a full-
129 scale wood-fueled downdraft gasifier. Furthermore, it tries to demonstrate that it is possible
130 to define and tune a reliable equilibrium Aspen Plus® simulation model using detailed
131 experimental data of a real gasification plant (equipment and streams). This model makes it
132 possible to effectively predict the performance of the plant over a wide range of operative
133 conditions.

134 To the best of the authors' knowledge, simulative models for a whole gasification plant with
135 fixed-bed downdraft gasifier have never presented in literature considering the actual
136 performance characteristics and operative behavior of the plant equipments.

137 Hence, the work described in this paper is very innovative and can be an useful tool for the
138 developers and users of biomass gasification combined heat and power plants.

139 On the other hand, there are several papers that describe a steady-state biomass gasification
140 model using Aspen Plus®, mainly in the field of fluidised bed gasifiers. These are briefly
141 summarized below. Ramzan et al. [14] reported an interesting comparative analysis between
142 the simulation performances of a lab-scale up-draft biomass gasifier and the experimental
143 data obtained in literature. Fu et al. [15] analyze without an experimental validation how the
144 performances of an autothermal biomass gasifier are affected by the gasifying air flow and

145 temperature. Doherty et al. [16-18] using experimental data from literature proposed and
146 validated an Aspen Plus® model based on the Gibbs free energy minimisation for a circulating
147 fluidised bed gasifier and for a steam blown dual fluidised bed gasifier, in order to show the
148 dependence of the gasifier performance on the gasifying air temperature.

149 Several kinds of fluidized bed gasifiers have been simulated and validated using a kinetic
150 model in [19-23], while other authors [24-28] used an equilibrium approach.

151 A semi detailed kinetic model coupling Aspen Plus® and dedicated fortran subroutines is
152 proposed in [29] for the simulation of an air-steam gasification of biomass in a bubbling
153 fluidised bed. The results of the modelling are well aligned with experimental results available
154 in literature.

155 Other authors focalized their studies on simulating and validating original two-stage biomass
156 gasifiers [30-31], while in [32] the entrained flow gasification of wood waste is simulated in
157 Aspen Plus® using a plug flow reactor with a kinetic approach. The model validation is
158 executed with experimental results.

159 In the present work, simulation results have been analyzed and compared with the
160 experimental ones obtained from a commercial-scale gasification plant based on a downdraft
161 gasifier. The plant, with the potential of roughly 80 kW_e, allows to control and adjust many
162 parameters like air flow and temperature into the gasifier or biomass moisture content (MC)
163 and also to measure chemical composition, temperature and flow of syngas coming out from
164 the gasifier.

165 Using a full-scale experimental biomass gasification plant many operative results were
166 available. This fact allowed both to make a detailed comparative analysis with simulation
167 results and to set some parameters of the model so to achieve an accurate model validation.

168 In this paper, after a brief introduction about the gasification principles, the technical and

169 operative characteristics of the full-scale experimental plant are described. Then, Aspen Plus®
170 model of the gasifier and the whole gasification plant are presented. After that, the
171 experimental and simulated data are compared and, successively, the performance
172 assessment of the gasification plant is discussed.

173

174 **2. GASIFICATION PRINCIPLES**

175 Gasification is a well-known thermochemical process that converts a solid fuel (usually
176 biomass or coal) into a combustible gaseous product (syngas) through partial oxidation, using
177 a gasifying agent in sub stoichiometric conditions [2-3]. When air is used as gasifying agent the
178 syngas consists mainly of carbon monoxide (CO), hydrogen (H₂), carbon dioxide (CO₂), steam
179 (H₂O), methane (CH₄) and nitrogen (N₂) with proportions that depend on air/biomass ratio
180 and MC. In addition there are trace amounts of higher hydrocarbons (such as acetylene,
181 ethene, ethane), and various contaminants such as small char particles, fly ash and tar [33-
182 34].

183 It is well known that the entire gasification process can be divided into four successive stages:
184 drying, pyrolysis, combustion and gasification [5,9].

185 In a downdraft fixed bed gasifier, the required heat for the endothermic biomass drying and
186 pyrolysis is provided via heat conduction through the biomass bed by the exothermic
187 combustion zone at the gasifying air inlet. The main reactions in combustion and gasification
188 processes are summarized in Table 1.

189 The thermodynamic performances of the gasification process can be evaluated using the
190 following parameters:

191 - the equivalent ratio (ER), defined as follows:

192

193
$$ER = \frac{m_{a-a}}{m_{a-s}} \quad (1)$$

194

195 - cold gas efficiency (CGE), defined as follows:

196

197
$$CGE = \frac{\dot{m}_s * LHV_s}{\dot{m}_b * LHV_b} \quad (2)$$

198

199 Therefore CGE represents the ratio between the inlet biomass chemical energy and the
200 corresponding chemical value of the syngas.

201

202

Table 1 Main gasification reactions.

Heterogeneous reactions:

$C_{(s)} + O_{2(v)} \rightarrow CO_{2(v)} + 394 \text{ kJ/mol}$	C complete combustion	(R1)
$C_{(s)} + 0.5 O_{2(v)} \rightarrow CO_{(v)} + 111 \text{ kJ/mol}$	C partial combustion	(R2)
$C_{(s)} + CO_{2(v)} \rightarrow 2 CO_{(v)} - 172 \text{ kJ/mol}$	Boudouard	(R3)
$C_{(s)} + H_2O_{(v)} \rightarrow CO_{(v)} + H_2 - 131 \text{ kJ/mol}$	Water-gas	(R4)
$C_{(s)} + 2 H_{2(v)} \rightarrow CH_{4(v)} + 75 \text{ kJ/mol}$	Methanation	(R5)

Homogeneous reactions:

$CO_{(v)} + 0.5 O_{2(v)} \rightarrow CO_{2(v)} + 283 \text{ kJ/mol}$	CO partial combustion	(R6)
$H_{2(v)} + 0.5 O_{2(v)} \rightarrow H_2O_{(v)} + 242 \text{ kJ/mol}$	H ₂ combustion	(R7)
$CO_{(v)} + H_2O_{(v)} \rightarrow CO_{2(v)} + H_2 + 41 \text{ kJ/mol}$	CO shift	(R8)
$CH_{4(v)} + H_2O_{(v)} \rightarrow CO_{(v)} + 3 H_2 - 206 \text{ kJ/mol}$	Steam-methane reforming	(R9)

H₂S and NH₃ formation reactions:



203

204 **3. THE EXPERIMENTAL GASIFICATION PLANT**

205 **3.1 Layout**

206 The experimental gasification plant (Figure 1) is the result of a long research activity that has
207 been performed at the “Dipartimento di Ingegneria Civile e Industriale” (DICI) and
208 “Dipartimento di Ingegneria dell’Energia, dei Sistemi, del Territorio e delle Costruzioni”
209 (DESTEC) of the University of Pisa (Italy).

210 The virgin chipped biomass is dried using a stand-alone concurrent rotating dryer that is
211 equipped, to accomplish the drying process, with a LPG fired burner. In a future commercial
212 configuration, the hot exhaust gas of the internal combustion engine fuelled by syngas will be
213 used for drying. Periodically, a sample of dried chips is analysed to evaluate its MC and
214 composition.

215 The dry wood chips are then filled into the gasifier using a screw conveyor and a rotary valve,
216 while the air flow coming into the gasifier is preheated initially through an electric preheater
217 (during the starting of the gasification plant when the syngas temperature is not enough high).

218 Later, when the steady state regime is reached, the air is heated passing through a shell-and-
219 tube heat recuperator, where the high-temperature syngas at the outlet of the gasifier is
220 cooled. In order to avoid the ~~obstruction~~ blockage of the syngas outlet section, and
221 consequently the stoppage of the reactor, the unburnt char is periodically extracted from the
222 gasifier.

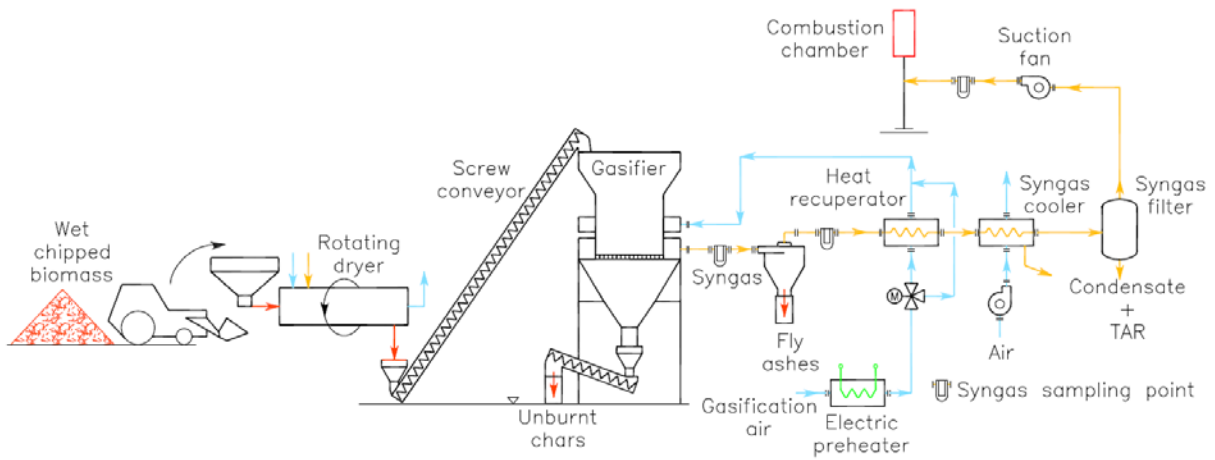
223 ~~The syngas light carbonaceous residues~~ The char residues and fly ash are removed from the
224 syngas in a cyclone. The syngas is further cooled in a second air-cooled shell-and-tube heat
225 exchanger. In the experimental facility this cooling air is dispersed into the atmosphere, but
226 in a commercial layout of the gasification plant the sensible heat of the syngas could be
227 effectively recovered for cogeneration application. At the outlet of the cooler, the contents of
228 pollutants in the syngas, such as fly ash and tar, are lowered using a custom-made filter. Then,
229 the syngas passes through the suction fan, which is responsible of the gasifying air-syngas
230 flow.

231 The syngas is finally oxidized in a custom combustion chamber equipped with a LPG burner.
232 This special combustion chamber has been adopted in place of a conventional torch for safety
233 reasons, since it ensures long residence time of CO at high temperatures and, consequently,
234 its complete oxidation. In the commercial version of the plant an internal combustion engine
235 in combination with a torch will replace the combustion chamber. The torch will be used to
236 oxidize the syngas when the quality of the gas is not suitable for the engine (for example,
237 during the plant starting) or when the engine does not work due to failures.

238 The operation of the experimental gasification plant is supervised by a programmable logic
239 controller (PLC) that can be managed by the user with a user-friendly touch screen system.
240 The temperature and pressure of each stream are measured and the measurement signals are
241 connected to the PLC system. Moreover, air and syngas flows are continuously measured via
242 two dedicated Honeywell flow transmitters (the syngas one includes the compensation of
243 temperature) based on the orifice plate method.

244 Using some sampling points located in different places along the syngas stream line, it is
245 possible to extract the syngas in order to evaluate its macro-components with an off-line gas-

246 chromatograph and also tar and ash content via a sampling probe. This probe was designed
247 and constructed in accordance with the tar Protocol [35-36].



248

249 **Figure 1** Layout of the biomass gasification power plant.

250

251 The most important design data of the experimental plant are summarized below:

252 - biomass mass flow feeding the gasifier with MC of 10 %: 90 kg/h

253 - gasifying air temperature at the inlet of the gasifier: 450 °C

254 - syngas mass flow: 200 kg/h

255 - syngas temperature at the inlet of the suction fan: 75 °C.

256

257 **3.2 Main control logics for the operative management of the experimental gasification plant**

258 The experimental gasification plant operates in accordance with some fundamental control

259 logics that were implemented and managed by a governing PLC. These ~~rules~~ logics assure large

260 flexibility from the operative point of view and the possibility to test different configurations.

261 In particular the logics are:

- 262 1. automatic adjustment of the opening of the motorized three way valve located upstream
263 of the heat recuperator (air-side) so that the air temperature just upstream the gasifier
264 reaches a specified set-point value;
- 265 2. the cooling air mass flow in the syngas cooler is tuned by modifying the rotational speed of
266 the fan via electric motor inverter, in order to obtain a set-point value of the syngas
267 temperature at the outlet of the cooler;
- 268 3. gasification flow logic: the speed of the syngas suction fan is modified via electric motor
269 inverter in order to obtain a specified syngas mass flow upstream the combustion chamber.
270 Similarly the logic can be modified using a set-point of the gasifying air mass flow as control
271 objective;
- 272 4. the filling of the reactor starts periodically in accordance with a time log and stops when
273 the level of the biomass chips inside the gasifier reaches the highest allowable level
274 activating a blade sensor level;
- 275 5. the unburned char is periodically discharged in order to avoid the blockage of the reactor
276 when the pressure drop across the reactor reaches the set-point level and then extracted
277 with a dedicated screw conveyor.

278

279 **4. ASPEN PLUS® MODEL**

280 Referring to the plant layout (Figure 1) the simulation flowsheet of the plant (Figure 2) has
281 been created.

282

283 **4.1 Gasifier**

284 A kinetic free equilibrium steady state model has been developed for the gasification process.

285 Initially the model simulates the biomass drying, reducing its MC up to a predetermined value.

286 Afterwards, biomass is decomposed into volatile components and char and then oxidation
 287 and gasification reactions are simulated by minimizing Gibbs free energy.

288 The block DRIER1 has been used to reduce the MC of moist biomass, simulating biomass drying
 289 controlled by a Fortran ~~statement~~ routine. Excess water is separated in the block DRYER2 (Sep
 290 type), while dry biomass with the right MC at the inlet of the gasification reactor is then
 291 decomposed into its conventional elements (C, H, O, N, S etc.) in the block DECO (Ryield type),
 292 that uses calculations based on the component yield specification, controlled by a Fortran
 293 statement. Ash and specified percentage of carbon of the dry biomass are separated in the
 294 block CHAR-SEP (Sep type) in order to simulate the unburnt char that is extracted from the
 295 bottom of the gasifier. The remaining elements are carried with the heat of reaction
 296 associated with the decomposition of the biomass into the block GASIFIER (RGibbs type),
 297 where the preheated gasifying air enters and the combustion and gasification reactions occur.

298 The gasification products are calculated by minimizing the Gibbs free energy and assuming
 299 complete chemical equilibrium. Finally, taking into account the reactor geometry and thermal
 300 insulation, pressure drop across the gasifier and heat losses to the ambient are calculated with
 301 a user routine (see Appendix A).

302 Biomass is specified as a non-conventional component, with a chemical composition defined
 303 by the ultimate and proximate analysis in accordance with the results of the laboratory
 304 analysis, as shown in Table 2.

305

306 **Table 2** Ultimate and proximate analyses of chestnut wood.

Proximate Analysis		Ultimate Analysis	
Moisture	10 %	Carbon	50.96 %

Fixed Carbon	50 %	Hydrogen	5.978 %
Volatile Matter	48 %	Nitrogen	0.49 %
Ash	2 %	Chlorine	0.0098 %
		Sulphur	0.0392 %
		Oxygen	40.523 %
		Ash	2 %

307

308 **4.2 Other equipment**

309 In order to reproduce the thermodynamic plant operation accurately and, after an
 310 experimental validation, to predict the behavior of the system in general operating conditions,
 311 the geometrical and operative characteristics of the equipment that are actually installed at
 312 the experimental facility have been inserted within the simulation model. In particular:

- 313 - pressure drop ratio factor, the pressure recovery factor and the valve flow coefficient of the
 314 valves (VA01, 3W-VALVE, VG01, VG02 in Figure 2) have been specified in accordance with
 315 the real data from the equipment datasheets. In this way it is possible to predict the
 316 pressure drop of the valve as a function of its geometrical dimensions and percent opening;
- 317 - geometrical data, such as internal diameter, length and material have been inserted for the
 318 piping (PIPE-1, PIPE-2, PIPE-3 in Figure 2). Further, with the addition of the calculator tool
 319 of Aspen Plus®, the heat losses to the ambient have been calculated in function of the
 320 actual insulation characteristics, the ambient air temperature and wind speed during the
 321 experimental tests;
- 322 - the geometry of the heat exchangers (HEATER and E02 in Figure 2) has been designed using
 323 the specific code of Aspen Plus® for the shell and tube heat exchangers and their simulation
 324 model has been linked within that of the gasification plant implemented into the main one.

325 In this way it is possible to assess the real thermodynamic off-design performance of the
326 heater when the operating conditions change with respect to the design point. The electric
327 heater (E-HEATER in Figure 2) has been simulated using a particular user routine in order
328 to assess its actual thermal performance in function of the thermal load and air mass flow.
329 The heat losses of the heaters to the environment have been calculated in function of their
330 specific geometry and the insulation characteristics;

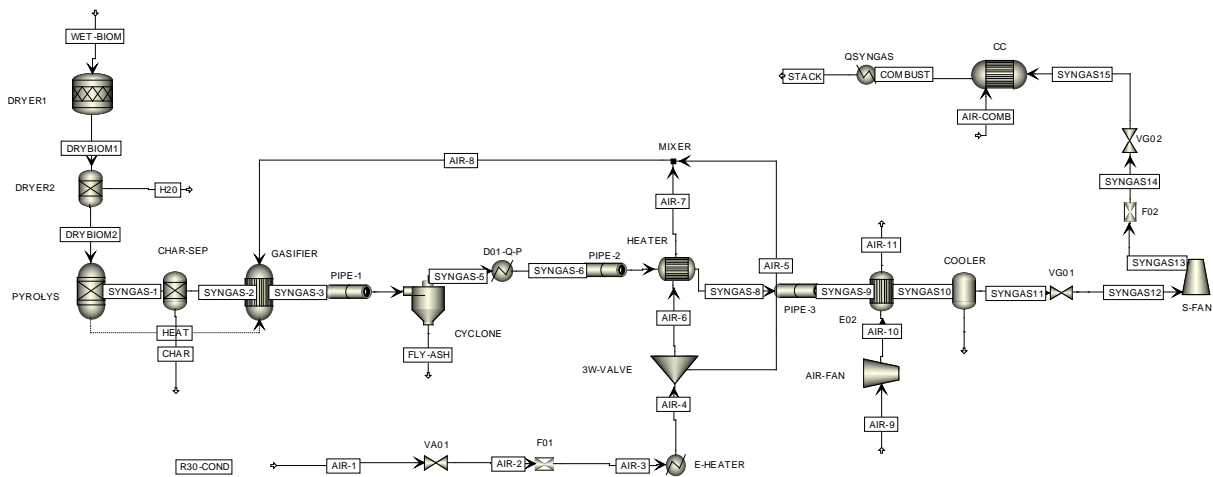
331 - geometrical data of the cyclone (CYCLONE in Figure 2) have been ~~inserted~~ considered in order
332 to assess its fly ash separation performance. A specific routine has been added in order to
333 evaluate the heat loss to the environment, adopting the approach described above for the
334 piping and inserting a heater block (D01-Q-P in Figure 2) downstream the cyclone;

335 - the simulation of the air and syngas fans (AIR-FAN and S-FAN in Figure 2) has been executed
336 inserting their characteristic curves in terms of head and efficiency as a function of flow at
337 different shaft rotational speeds in accordance with the manufacturer datasheets. The
338 actual operating speed of the fans is calculated once the flow and the head have been
339 evaluated in agreement with the control logics described in the previous section and
340 assuring the gas flow with the calculated pressure drop, respectively;

341 - the final complete oxidation of the syngas within the combustion chamber (CC in Figure 2)
342 has been simulated using a RGibbs type block. The pressure loss through the combustion
343 chamber has been inserted as an input of the model using the experimental data. The
344 overall chemical power that is associated with the syngas flow is calculated via the cooling
345 of the combustion products with a heater block (QSYNGAS in Figure 2);

346 - the simulation of the syngas filter (COOLER in Figure 2) that is positioned upstream the fan
347 is executed using a separation block (Sep type) with a pressure loss that has been
348 experimentally evaluated in function of the actual syngas flow;

349 - the air and syngas flowmeters (F01 and F02 in Figure 2), which are based on the orifice plate
 350 technology, have been simulated with valves whose pressure losses are in accordance with
 351 formulation reported in [37] as function of the volume flow.
 352 The control logics of the experimental plant have been implemented in the simulation model
 353 using the Design Specs tool of Aspen Plus®. In this way it is possible to find the value of one or
 354 more control variables, such as, for example, the motor speed of the syngas fan, in order to
 355 iteratively reach a specified goal, such as the syngas mass flow.
 356



357
 358 **Figure 2** Aspen Plus® simulation model of the experimental gasification plant.
 359

360 **4.3 Physical property method**

361 The equation of state that is used to estimate all physical properties of the conventional
 362 components is the Peng-Robinson equation with Boston-Mathias alpha function (PR-BM),
 363 which is appropriate for gasification processes where temperature is quite high.
 364

365 **4.4 Simplifying assumptions**

366 Within the model some simplifying assumptions that do not markedly affect the goodness of
367 the simulation results are:

- 368 • steady state conditions: as demonstrated by the experimental data, after roughly two
369 hours from the starting of the gasification reactions, the temperature profile within each
370 equipment, such as the gasifier and the heat exchangers, fluctuates slightly. This assures
371 that the operating conditions do not practically change in time.
- 372 • kinetic free model: as stated in the Introduction, the estimation of reliable kinetic data for
373 the specific gasification configuration can be an hard task without assuring the goodness
374 of the results. The adoption of the equilibrium approach in combination with a detailed
375 geometrical simulation of the plant equipment can assure in any case to obtain a good
376 representation of the experimental data;
- 377 • the sulphur reacts forming H_2S , as demonstrated by the experimental analysis;
- 378 • no nitrogen oxides are considered and N_2 forms only NH_3 : the study of the formation of
379 the micro-pollutant is not an objective of the paper. They do not affect the overall energy
380 balance of the gasifier and the macro-composition of the syngas;
- 381 • the formation of the tars and other heavy products at equilibrium conditions are not
382 simulated. It is important to note that their influence on the overall energy balance of the
383 gasifier is marginal.

384

385 **5. RESULTS AND DISCUSSION**

386

387 **5.1 Experimental activity vs simulation results**

388 Several different operative conditions have been considered during the experimental activity,
389 varying the ER (modifying the suction fan rotational speed and consequently the air mass
390 flows) and the gasifying air temperature at the inlet of the reactor (changing the opening of
391 the bypass valve of the air preheater). As stated above, the thermodynamic data of each
392 stream of the plant, the biomass characteristics and the syngas composition have been
393 measured during the tests. Some experimental data have been used as inputs of the Aspen
394 Plus® simulation model. In particular:

- 395 - the ambient gasifying air: temperature, pressure, relative humidity and mass flow,
396 temperature at the inlet of the gasifier;
- 397 - biomass: chemical composition, MC, mass flow;
- 398 - syngas: temperature at the outlet of the cooler;
- 399 - unburnt char that is extracted from the bottom of the gasifier: mass flow, chemical
400 composition;
- 401 - fly ashes from the gasifier: size distribution, concentration.

402 Using these inputs, the simulation model calculates the syngas composition, temperature and
403 mass flow at each point of the plant (and consequently the rotational speed of each fan), the
404 thermal power of each heat exchanger, the aperture of the control valve of the air preheater.

405 The comparison of the experimental data and the results of the simulations, that have been
406 executed using data of about twenty experimental tests (Appendix B), are reported in Figures
407 3, 4, 5, 6 and 7. The trend and the values of the mass composition of the syngas are well

408 simulated by Aspen Plus® and the percentage error is marginal, also considering the intrinsic
409 error of the experimental measurements, which can be summarized in the following:

410 (i) the measurements of the temperature that are executed using thermocouples of type K
411 have a standard intrinsic tolerance of $\pm 6 \%$;

412 (ii) the mass flow of the biomass has not been continuously monitored, but it is evaluated
413 measuring in average the biomass that is consumed;

414 (iii) the MC of biomass, that is not an homogeneous fuel, is not evidently measured in
415 continuous way, but some representative samples have been analysed. Some MC
416 differences between the measurement instant and the moment of gasification are
417 inevitable due to the fact that the material is not homogeneous and some moisture is
418 slightly absorbed from the environment;

419 (iv) as the experimental experience of the authors, the syngas composition is not perfectly
420 stable and fluctuates due to the fact that the biomass within the gasification bed is
421 evidently heterogeneous and the air and syngas fluid-dynamic through the biomass is
422 affected by inevitable variations;

423 (v) the volume flow measurement of air and syngas is affected by an error by about $\pm 1 \%$;

424 (vi) there are inevitable errors of the laboratory measurements that can be estimate equal to
425 about $\pm 0.5 \%$.

426 The average value and the standard deviation of the percentage difference between the
427 experimental and simulated results are summarized in Table 3. The parity plot of the molar
428 composition between the simulated values and the experimental ones, reported in Figure 8,
429 allows to assess the prediction accuracy of the simulation model. The average differences
430 between the measured LHV of the syngas and the simulated values and between the
431 experimental CGE and the simulated ones as well are about 7 % and 5 %, respectively.

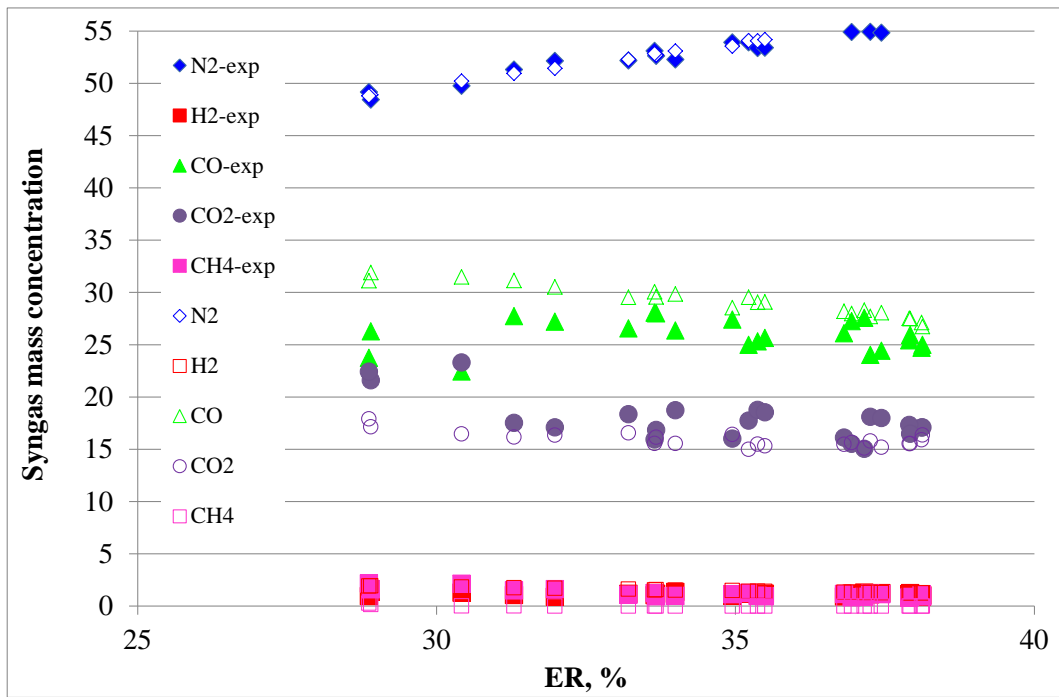
432 Moreover, the average difference between the simulated values of the syngas temperature
433 at the outlet of the gasifier and the experimental values is lower than 7 %. On the basis of
434 these negligible differences, the Aspen simulation model can be considered particularly
435 accurate for the estimation of the most important energy performance indicators and
436 operative data of the experimental facility. The most relevant difference between the
437 experimental and simulated results concern the mass and molar concentration of H₂, that is
438 overestimated, and CH₄, that is underestimated. Using the equilibrium hypothesis in the
439 simulation model, the conversion of the methane into hydrogen, which depends on the actual
440 crossing time of the gasification bed, is overestimated. Indeed, the methane that is produced
441 during the pyrolysis is progressively converted along the gasifying bed into H₂ and CO in
442 accordance with the steam reforming reaction. Using the hypothesis of equilibrium, the steam
443 reforming reaction is completed shifted toward the products, as reported also by other
444 authors [38-39]. This condition is hardly confirmed in real situations. However, it is important
445 to note that the differences are lower with higher values of ER, when the hypothesis of
446 equilibrium is more respected. Moreover, considering the overall amount of hydrogen moles
447 and mass the differences reduce largely (Table 3).

448 However, as a whole, the dependence of the syngas composition on the ER is in good
449 agreement with the values of literature [7, 40-41]. Notwithstanding this difference for the H₂
450 estimation, the results concerning the plant operation and its energy balance are well
451 simulated by the Aspen Plus® model.

452 In general, the increasing ER implies a larger extension of the combustion process within the
453 reactor (hence the temperature inside the reactor increases, as shown in Figure 5), with a
454 lower content of the combustible components in favor of the nitrogen content that increases.

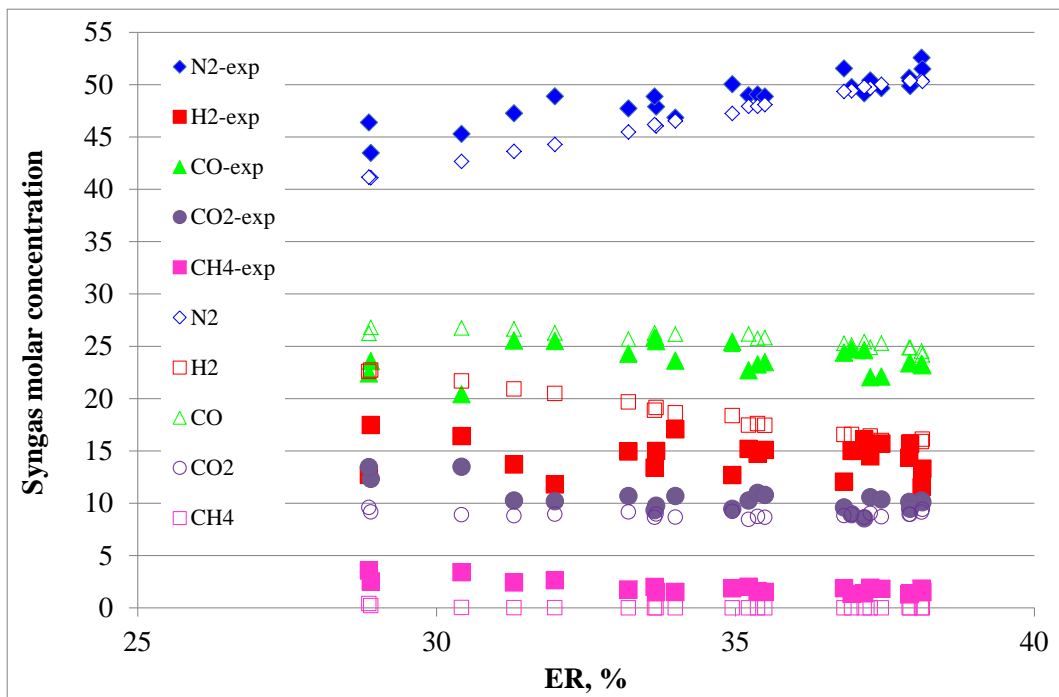
455 When the gasifying air increases, the LHV of the syngas consequently decreases (Figure 7) due

456 to the oxidation of the hydrogen and CO and the dilution due to the nitrogen. Consequently,
457 CGE decreases with ER (see Figure 7), too.



458

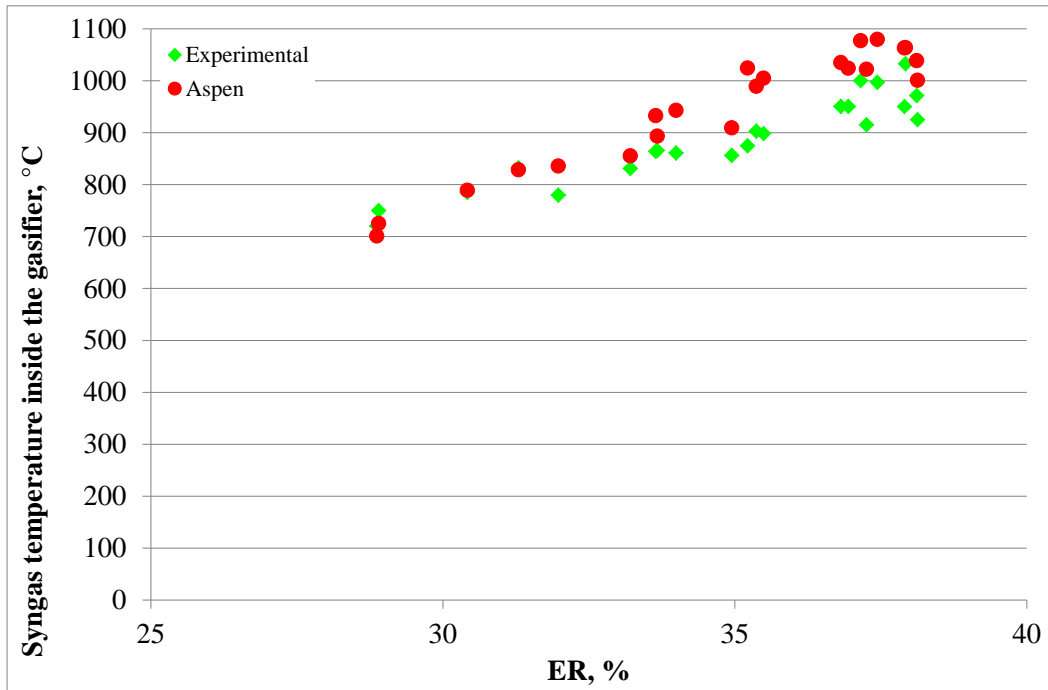
459 **Figure 3** Comparison of the experimental syngas mass composition (labelled with “exp”)
460 with the results of the Aspen Plus® simulation model (dry basis).
461



462

463 **Figure 4** Comparison of the experimental syngas molar composition (labelled with “exp”)
464 with the results of the Aspen Plus® simulation model (dry basis).

465

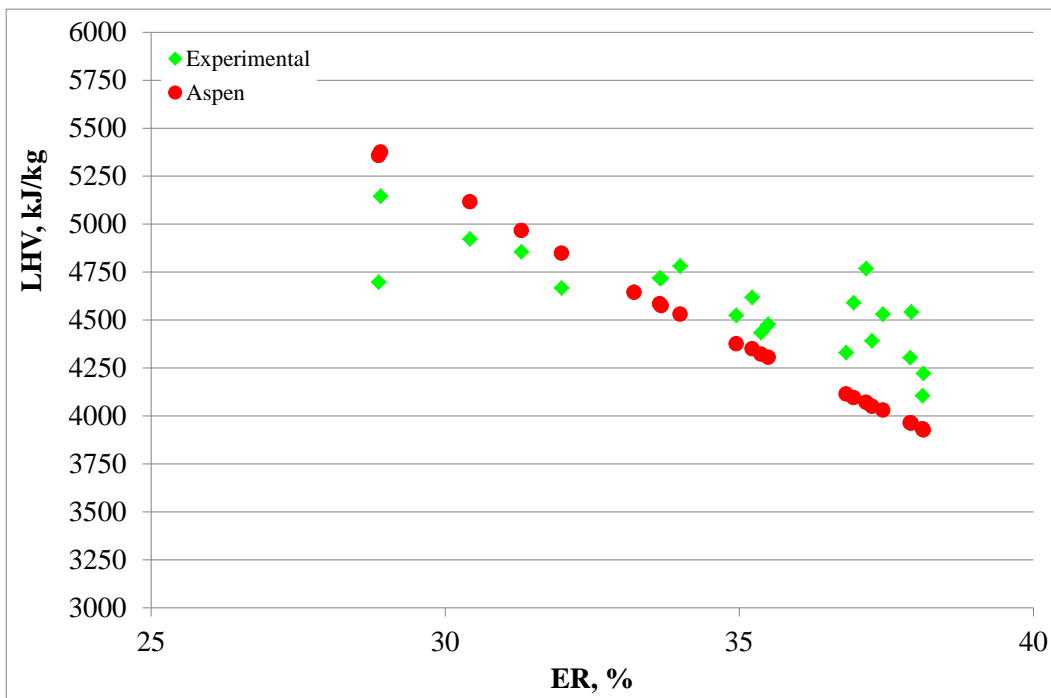


466

467 **Figure 5** Comparison of the experimental syngas temperature inside the gasifier with the

468 results of the Aspen Plus® simulation model.

469

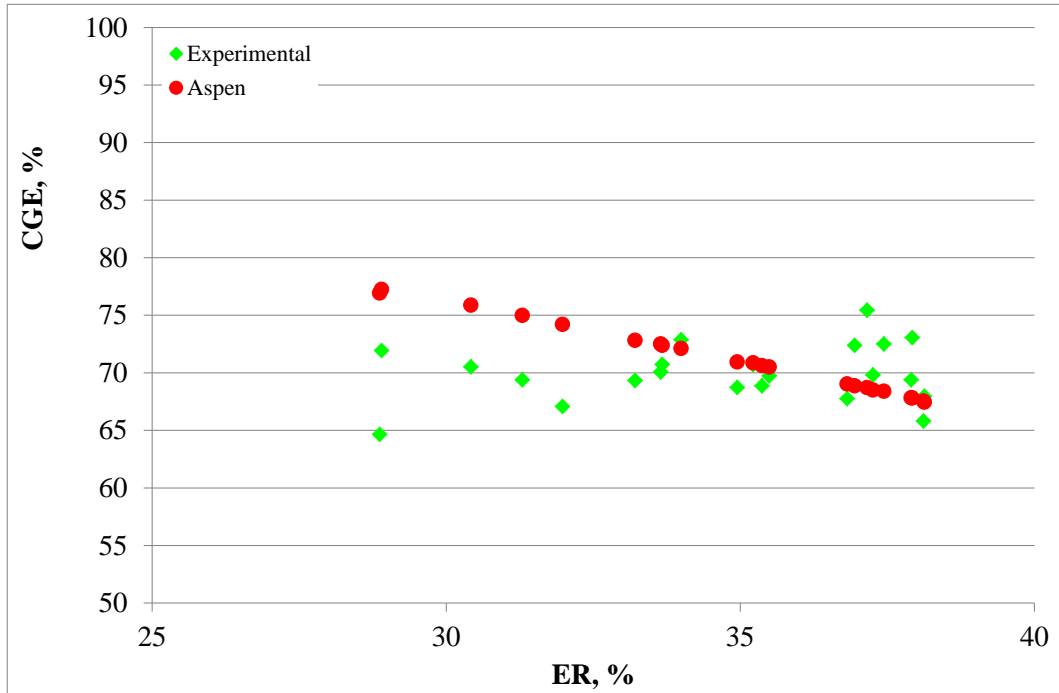


470

471 **Figure 6** Comparison of the experimental syngas lower heating value with the results of the

472 Aspen Plus® simulation model.

473

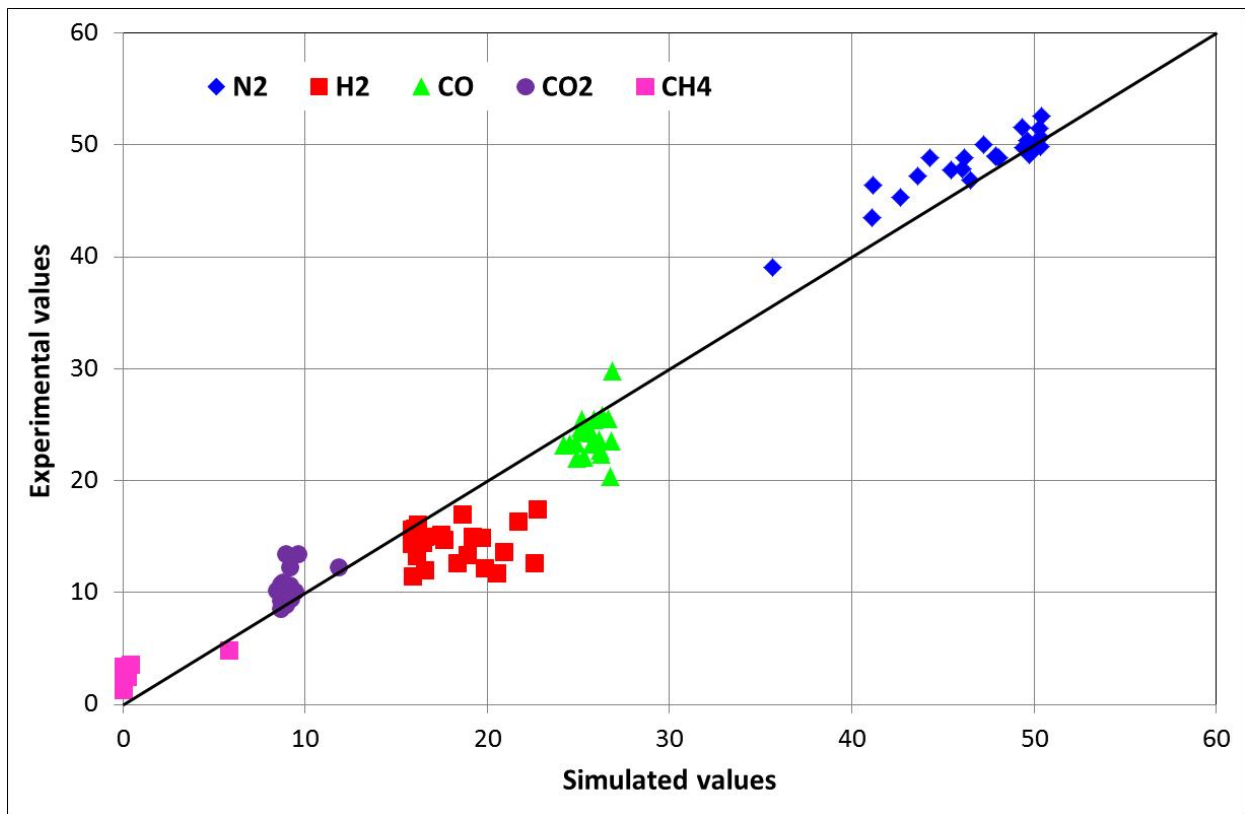


474

475 **Figure 7** Comparison of the experimental cold gas efficiency with the results of the Aspen
 476 Plus® simulation model.

477 **Table 3** Average value and standard deviation of the percentage difference between
 478 the experimental and simulated results.

	Syngas mass concentration				
	N ₂	H ₂	CO	CO ₂	CH ₄
Average	0.7	35.2	12.2	10.2	95.9
Standard deviation	0.4	27.4	9.1	7.7	14.9
	Syngas molar concentration				
	N ₂	H ₂	CO	CO ₂	CH ₄
Average	4.0	29.0	8.4	13.1	95.5
Standard deviation	3.1	22.2	7.0	8.8	17.1
	CGE	LHV	Syngas temperature	H ₂ mass	H ₂ mole
Average	5.5	6.6	6.3	13.2	4.6
Standard deviation	5.2	4.5	5.1	12.6	3.4



480

481

Figure 8 Parity plot for the molar composition of the syngas (dry basis).

482

483

5.2 Performance assessment of the experimental gasification plant with Aspen Plus®

484

Once the reliability of the simulation model has been demonstrated using the comparison

485

with the experimental data, it is possible to use it to predict and assess the thermodynamic

486

and energy performance of the experimental gasification plant in various operative conditions

487

without the necessity to execute further experimental tests. The most important controllable

488

parameters for the gasification plant user are:

489

- the MC of the biomass, that can largely vary from a supply to another. Indeed, the wood

490

chipped biomass can contain variable amount of the water during the year and the drier

491

cannot assure a constant MC of the dried biomass;

492

- the ER that largely affects the gasification efficiency and the syngas composition. It is the

493 most simple controllable plant parameter that the user can easily modify, operating the
494 suction fan;

495 • the gasifying air temperature, that can be simply modified with the control valve and
496 affects the overall thermal performance of the gasification plant.

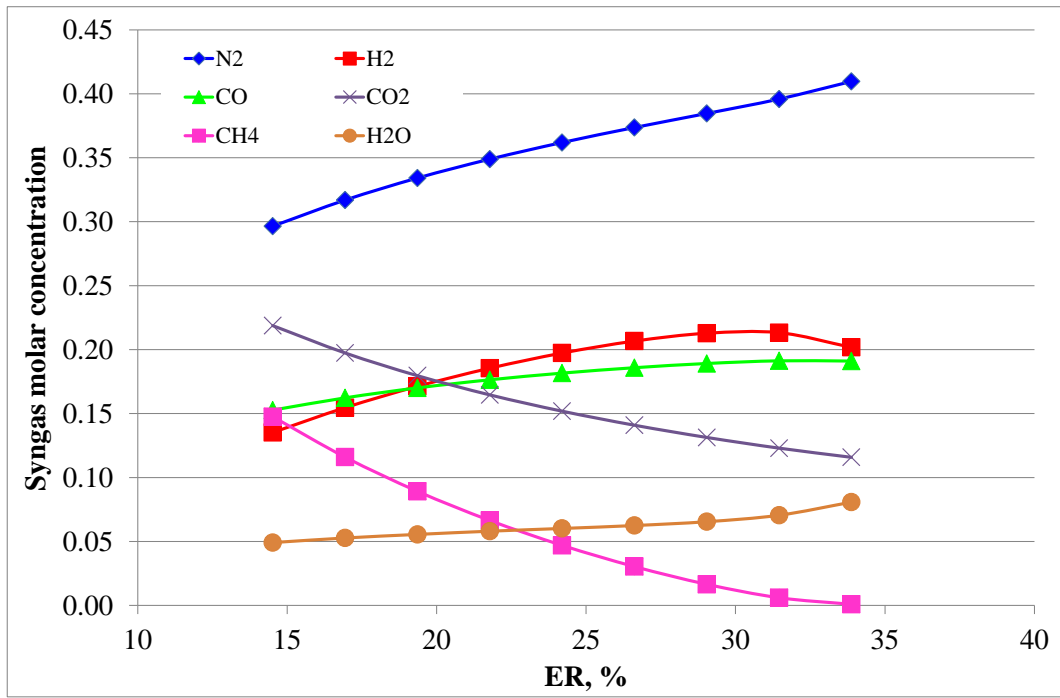
497 Hence, the simulations have been executed at different values of biomass MC and gasifying
498 air temperature vs. ER. In particular, two extreme values have been adopted for the biomass
499 MC (6 % and 14 %) and for the gasifying air temperature (20 °C and 300 °C). When the MC of
500 biomass has been increased, we assumed to maintain constant its dry matter mass flow equal
501 to 72 kg/h. The lowest value of the biomass MC can be considered the lower bound that can
502 be practically obtained using commercial industrial driers. The highest value is generally
503 considered the maximum allowable value in order to avoid an excessive production of tar in
504 the syngas. The maximum value of the gasifying air temperature can be easily obtained using
505 the air preheating with suitable gas-gas heat exchangers. The lowest value, that corresponds
506 to the atmospheric temperature, represents the absence of the air preheating and the heat
507 exchanger is completely bypassed by the air. So, ambient air is directly used as gasifying agent.
508 In this case the sensible heat of the syngas can be recovered during the successive cooling
509 with the atmospheric air.

510 The reduction of the biomass MC (Figures 9 and 10) ensures a higher production of CO, with
511 an increase in LHV_s (Figure 11) and CGE (Figure 12). Indeed, the absorption of latent heat
512 (required for the water vaporization) reduces the useful heat for the gasification reaction and
513 the presence of steam tends to dilute the syngas. The value of ER with the highest H₂ and CO
514 content in the syngas is slightly lower with high gasifying air temperature (Figures 9 and 10).

515 Figures 11 and 12 show that the reduction of ER increases syngas LHV and CGE, so the
516 adoption of low ER could be reasonable. Actually the model does not take into consideration

517 tar production which drastically increases at the lowest ER. Usually, an ER around 25-30 % is
518 adopted during operative conditions.

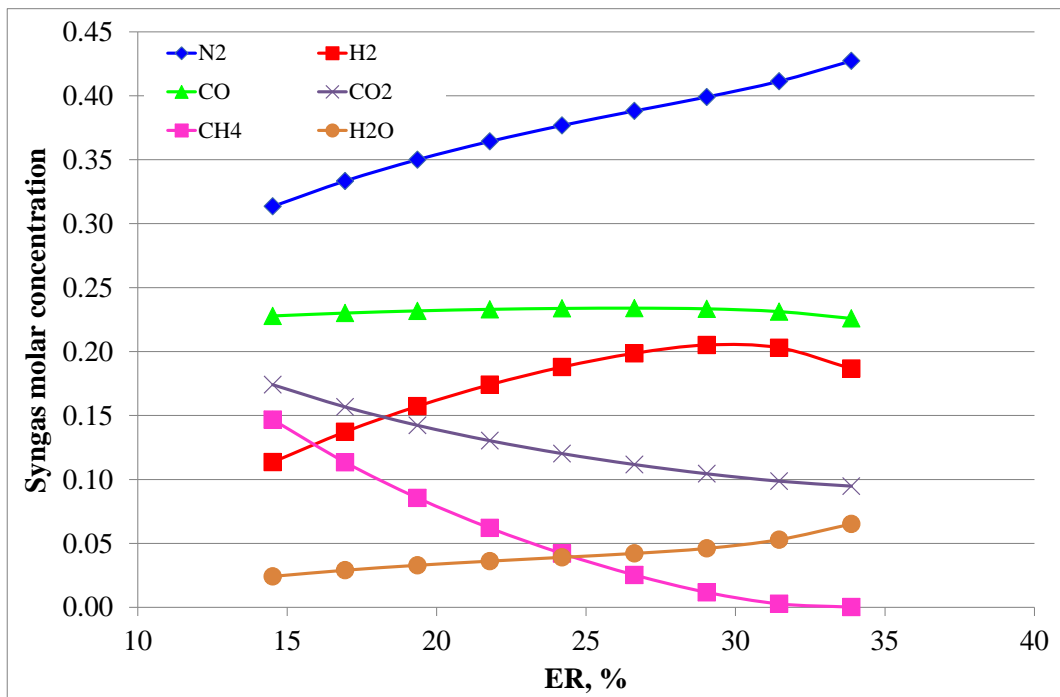
519 The effect of the gasifying air temperature on the gasifier performance is more relevant in
520 comparison with the MC (Figures 9, 10, 11, 12). On average, the change of the gasifying air
521 temperature from 20 °C to 300 °C implies the increase of the gasification efficiency by about
522 two percentage points. High values of the temperature assure an effective heating of the
523 biomass bed within the reactor and a more efficient development of the gasification reactions
524 with a higher syngas outlet temperature, as shown in Figure 13, and consequently higher
525 biomass conversion into syngas. Moreover, the syngas outlet temperature (Figure 13) can be
526 higher when the biomass has a low MC and, consequently, a higher LHV. With a low value of
527 MC it is possible to obtain higher gasification efficiencies as it happens increasing the air inlet
528 temperature. In particular, the change of MC from 6 % to 14 % implies the increase of the
529 gasification efficiency by about one percentage point.



530

531

(a)



532

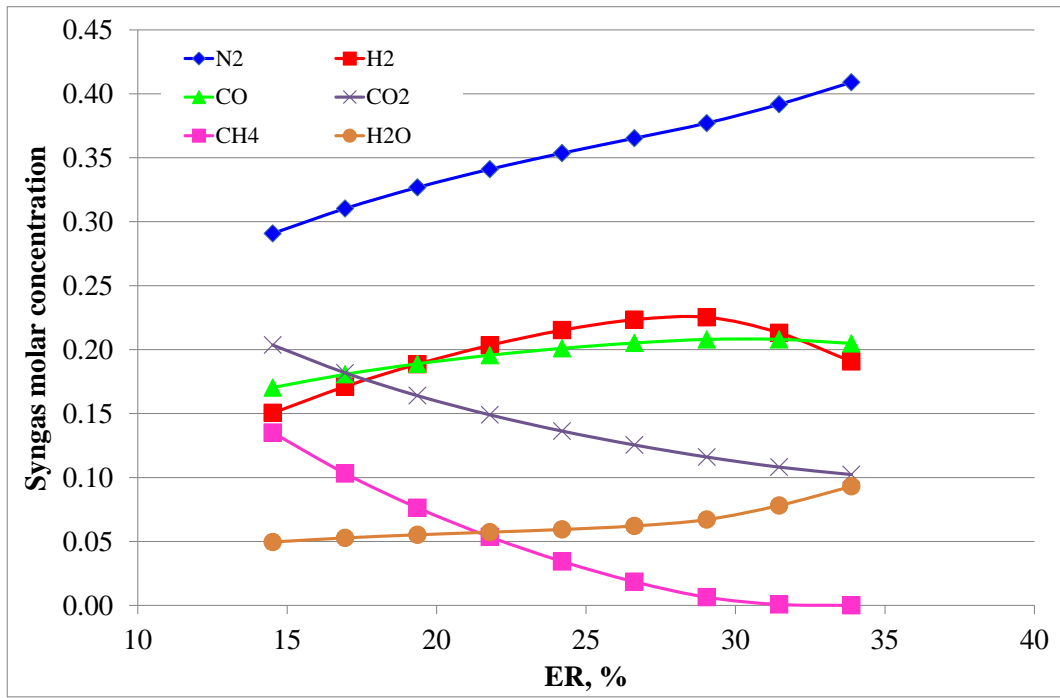
533

(b)

534

535

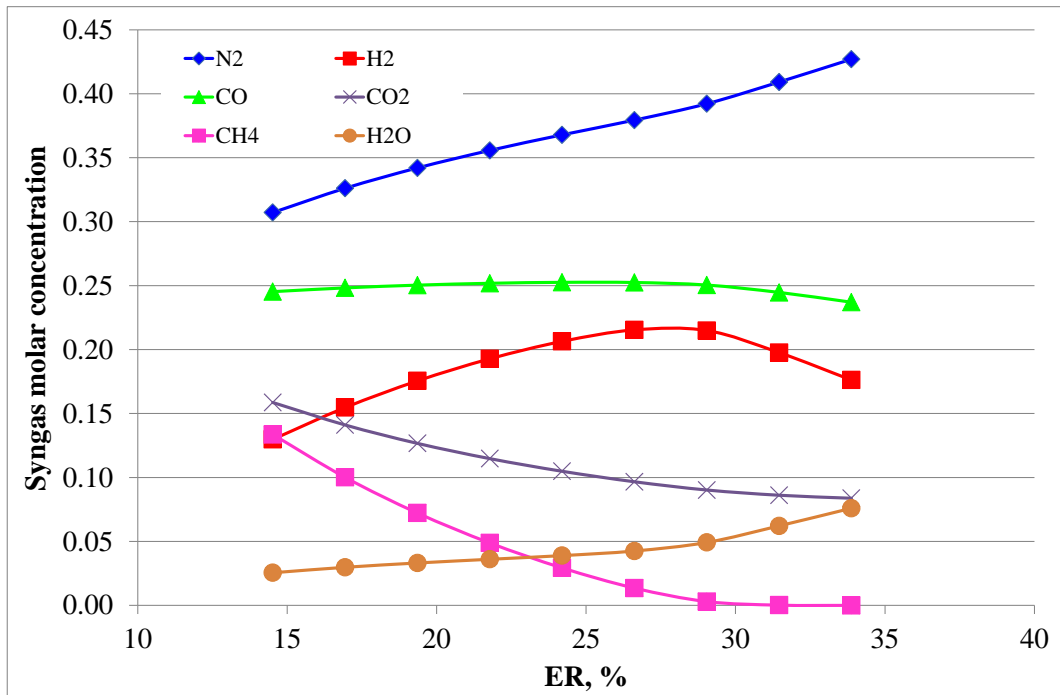
Figure 9 Syngas composition with the Aspen Plus® model, when the biomass moisture content is equal to 14 % (a) and 6 % (b) and the gasifying air temperature is equal to 20 °C.



536

537

(a)



538

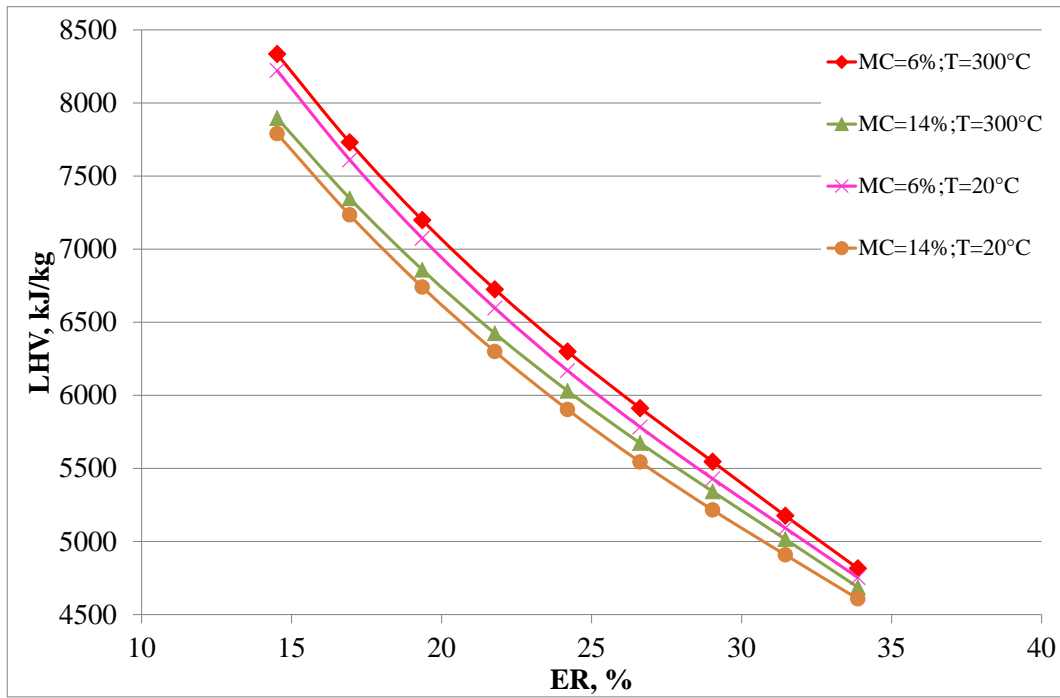
539

(b)

540

541

Figure 10 Syngas composition with the Aspen Plus® model, when the biomass moisture content is equal to 14 % (a) and 6 % (b) and the gasifying air temperature is equal to 300 °C.



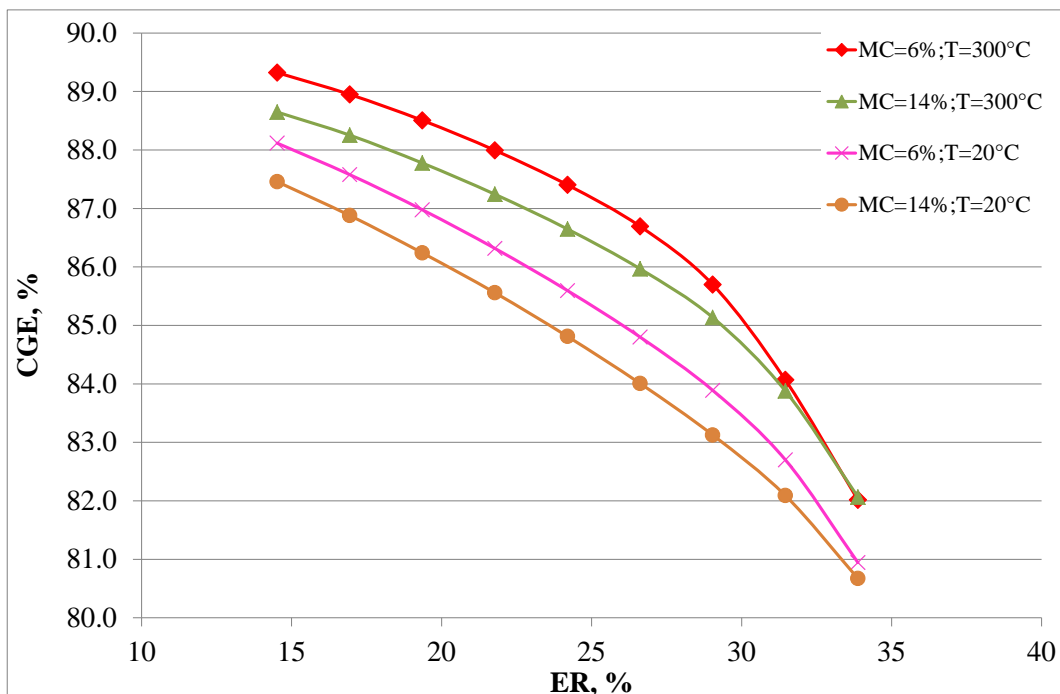
542

543 **Figure 11** Lower heating value of the syngas for two values of the biomass moisture content

544

and gasifying air temperature.

545

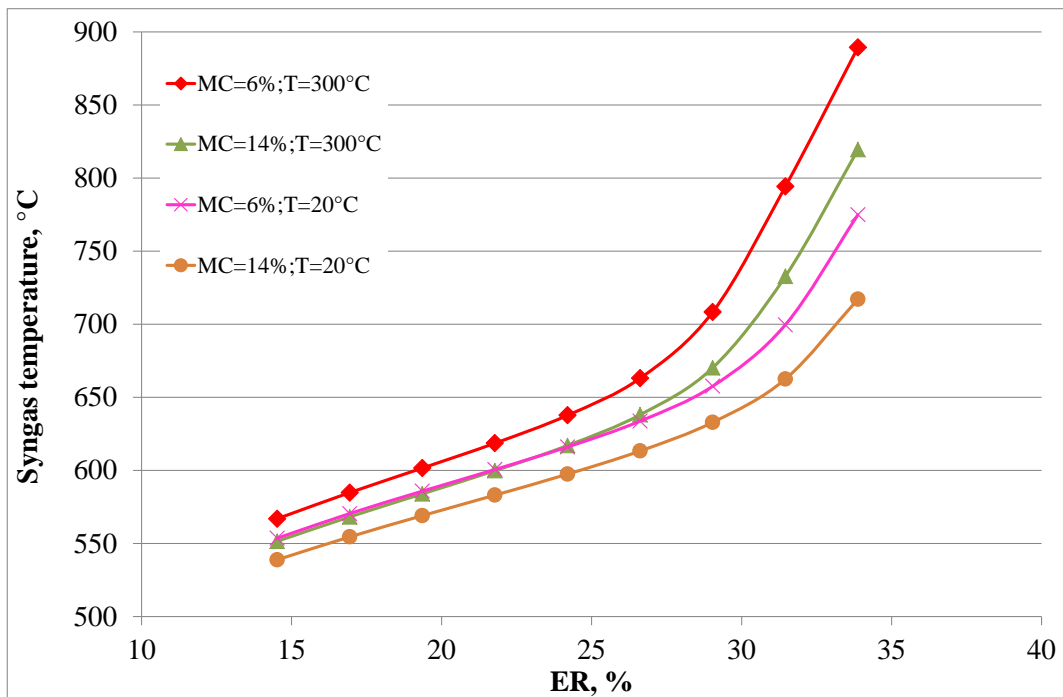


546

547 **Figure 12** Cold gasification efficiency for two values of the biomass moisture content and

548

gasifying air temperature.



549

550 **Figure 13** Syngas temperature at the outlet of the gasifier for two values of the biomass

551 moisture content and gasifying air temperature.

552

553 **5.3 Comparison of the simulated results with literature data**

554 It is interesting to compare the simulated results presented in the previous section with the

555 data that are available in the wide scientific literature. ~~In order to avoid the use of~~

556 ~~heterogeneous results achieved with markedly different hypotheses, we have taken into~~

557 ~~account only results that are obtained with equilibrium mathematical modelling and concern~~

558 ~~explicitly small-scale biomass downdraft gasifiers [23].~~ In order to make a reasonable

559 comparison, we have taken into account only results that are obtained with equilibrium

560 mathematical modelling and concern explicitly small-scale biomass downdraft gasifiers [23].

561 Moreover, further experimental reference [42] has been taken into account for the

562 comparison. As previously mentioned the simulations have been performed considering

563 biomass characteristics, gasifying air temperature and ER (see Table 4).

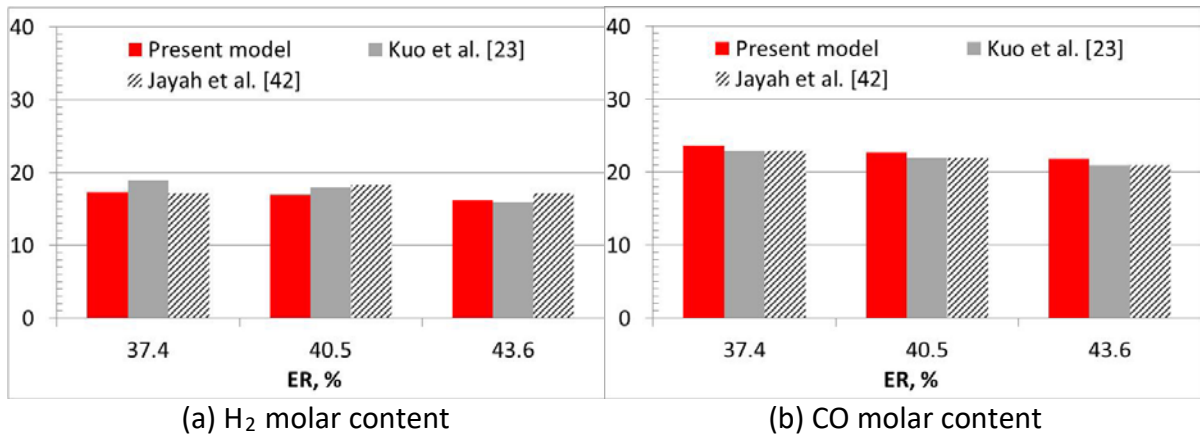
564

565 **Table 4** Ultimate and proximate analyses of biomass for the literature comparison
 566 [42].

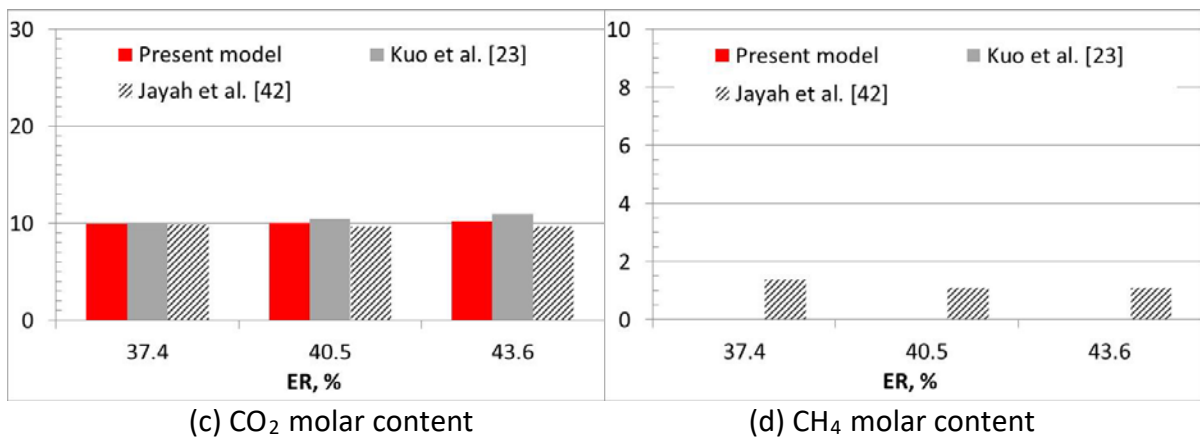
Proximate Analysis (wt%)		Ultimate Analysis (wt%)	
Moisture	5.76 %	Carbon	48.64 %
Fixed Carbon	14.4 %	Hydrogen	5.64 %
Volatile Matter	78.76 %	Nitrogen	0.52 %
Ash	1.08 %	Sulphur	0.03 %
		Oxygen	44.09 %
Higher heating value (MJ/kg)		18.94	
Gasifying air temperature (°C)		20	

567
 568 The results of the comparison, executed considering three values of ER, are summarized in
 569 Figure 14, where it is possible to note a good agreement between current simulated results
 570 and the references ones. The maximum relative error of H₂, CO and CO₂ molar concentration
 571 on dry basis between the current results and those of [23] is about 9 %, 4 % and 7 %, respectively.
 572 If the comparison is executed with the experimental reference [42], the maximum relative error is about 7 %, 4 % and 5.5 %, respectively. As previously mentioned in
 573 section 5.1, large differences are present concerning methane whose simulated predictions is
 574 close to zero. This depends on the fact that the hypothesis of equilibrium for large values of
 575 ER (as those used in this comparison) practically implies the complete conversion of methane
 576 into hydrogen and carbon monoxide [38-39], even if some incomplete conversion of pyrolysis
 577 products can occur in real operative conditions [23].
 578
 579

580
581



582
583
584



585 **Figure 14** Comparison of (a)H₂, (b) CO, (c) CO₂, (d) CH₄ concentration between present and
586 literature data.

587

588 6. CONCLUSIONS AND FUTURE REMARKS

589 In this paper, a detailed numerical model developed with Aspen Plus® for an experimental
590 full-scale biomass gasification plant has been proposed, simulating the gasification process
591 with a kinetic free equilibrium approach.

592 The model has been implemented with all the measured plant data, such as the exact
593 geometrical and performance characteristics of the plant equipment and the control
594 operative logics. This approach assured to obtain a good matching between the simulation
595 results and the plant data in terms of syngas composition and energy performance of the
596 gasification process. In particular, the syngas composition is well simulated and predicted

597 except for the hydrogen and methane components, because the equilibrium assumption of
598 the model implies the complete conversion of methane into hydrogen. The other parameters,
599 such as the LHV of the syngas and the CGE, are estimated by the simulation model with an
600 average percentage error lower than 7 %.

601 Once the reliability of the simulation model has been demonstrated with the experimental
602 results, it has been used to analyse the operative behavior and energy performance with
603 respect to some important plant parameters. The most meaningful results are summarized
604 below:

- 605 - ~~By~~ by recovering the sensible heat of the syngas at the outlet of the gasifier, it is possible
606 to obtain high values of the gasifying air temperature and an improvement of the overall
607 gasification performances.
- 608 - The adoption of dried biomass with higher LHV assures higher gasification efficiencies with
609 larger production of CO.
- 610 - The decrease of ER from about 35 % to about 15 % implies an increase of the gasification
611 efficiency by about 6-7 % in function of MC and gasifying air temperature.
- 612 - An increase of about 300 °C of the gasifying air temperature assures an improvement of
613 two percentage points of the gasification efficiency.
- 614 - As a whole, the influence of MC on the gasifier performance is lower than that of the
615 gasifying air temperature.

616 The simulation model here presented allows to develop other investigations about some
617 modified layouts of the experimental gasification plant:

- 618 - the syngas suction fan can be moved upstream the gasifier in order to obtain a pressurized
619 gasification process;

- 620 - the insertion within the simulation model of a specific external routine for the performance
621 assessment of the internal combustion engine;
- 622 - the flowing of some syngas through the gasifier to combine air gasification with CO₂
623 gasification.

624

625 **APPENDIX A**

626 **A.1 Calculation of the pressure drop of the syngas across the gasifier**

627 The pressure drop of the air/syngas across the gasifier (ΔP [Pa]) has been estimated with the
628 Ergun equation for flow through a randomly packed bed of spheres as follows [43]:

629

$$630 \quad \Delta P = \left(150 \frac{(1-\varepsilon)^2}{\varepsilon^3} \frac{\mu_i}{d_p^2} u_i + 1.75 \frac{(1-\varepsilon)}{\varepsilon^3} \frac{\rho_i}{d_p} u_i^2 \right) l \quad (\text{A.1})$$

631

632 **A.2 Calculation of the heat loss of the gasifier to the environment**

633 The reactor is constituted by a stainless steel shell, which is internally protected by a refractory
634 layer. The external surface of the reactor is insulated by ceramic fiber insulation that is
635 protected by an aluminum cover. The procedure for the estimation of the thermal losses from
636 the gasifier into the environment is described below [44-46]:

637

$$638 \quad \dot{Q} = \frac{T_r - T_e}{R_{tot}} \quad (\text{A.2})$$

639

640 R_{tot} can be calculated as follows:

641

642
$$R_{tot} = R_i + R_{c1} + R_{c2} + R_{c3} + \left(\frac{1}{R_e} + \frac{1}{R_r} \right) \quad (\text{A.3})$$

643

644 where the thermal resistances are summarized in Table A1.

645

646

Table A1 Thermal resistances of the gasifier.

$$R_i = \frac{1}{\alpha_i \pi D_i L}$$

$$R_{c1} = \frac{\ln\left(\frac{D_{e_refractory}}{D_i}\right)}{2\pi k_{refractory} L}$$

$$R_{c2} = \frac{\ln\left(\frac{D_{e_shell}}{D_{e_refractory}}\right)}{2\pi k_{shell} L}$$

$$R_{c3} = \frac{\ln\left(\frac{D_{e_insulation}}{D_{e_shell}}\right)}{2\pi k_{insulation} L}$$

$$R_e = \frac{1}{\alpha_e \pi D_{e_insulation} L}$$

$$R_r = \frac{1}{\alpha_r \pi D_{e_insulation} L}$$

647

648 The evaluation of α_i [W/m² K] for the convective heat exchange between the air/syngas and
 649 the internal surface of the refractory layer is executed with the following expression:

650
$$\alpha_i = \frac{Nu_i k_i}{D_i} \quad (\text{A.4})$$

651 where

652
$$Nu_i = 0.023 Re_i^{0.8} Pr_i^{1/3} \quad (\text{A.5})$$

653
$$Re_i = \frac{\rho_i u_i D_i}{\mu_i} \quad (A.6)$$

654
$$Pr_i = \frac{Cp_i \mu_i}{k_i} \quad (A.7)$$

655 α_e [W/m² K] is calculated considering the wind flow across the cylindrical shell using the
 656 relation of Churchill-Bernstein:

657
$$\alpha_e = \frac{Nu_a k_a}{D_{e_insulation}} \quad (A.8)$$

658 where

659
$$Nu_a = 0.3 + 0.62 \frac{Re_a^{0.5} Pr_a^{1/3}}{\left[1 + (0.4 / Pr_a^{2/3})\right]^{0.25}} \left[1 + \left(\frac{Re_a}{282000}\right)^{5/8}\right]^{0.8} \quad (A.9)$$

660
$$Re_a = \frac{\rho_a u_a D_{e_insulation}}{\mu_a} \quad (A.10)$$

661
$$Pr_a = \frac{Cp_a \mu_a}{k_a} \quad (A.11)$$

662

663 The evaluation of the radiative heat exchange between the cover of the external thermal
 664 insulation of the gasifier and the environment has been executed considering an equivalent
 665 coefficient of convective heat exchange. Hence, the evaluation of α_r is obtained with the
 666 following expression:

667

668
$$\alpha_r = E_m \sigma (T_p + T_e) (T_p^2 + T_e^2) \quad (A.12)$$

669

670 **APPENDIX B**

671 The experimental data that have been used for the simulation with the Aspen Plus® model
672 are summarized in Table B1.

673

674 **Table B1** Experimental data used in the Aspen Plus® simulation

ER, %	Gasifying air temperature at the inlet of the reactor, °C	Biomass moisture content, %
34.9	252	5.6
33.2	280	5.6
33.7	328	6.5
28.9	375	6.5
34.0	414	6.5
28.9	284	8.3
32.0	351	8.3
31.3	401	7.4
33.6	422	7.4
20.4	427	4.8
36.9	350	9.8
35.2	491	7.9
38.1	292	7.9
36.8	384	10.1
38.1	206	4.7
37.9	362	7.1
35.4	362	7.6
35.5	400	7.6
37.3	425	5.5
30.4	321	8.7
37.4	394	8.7
37.9	434	6.2
37.2	450	6.2

675

676 **ACKNOWLEDGMENTS**

677 The authors wish to thank Regione Toscana for financial support of the project ICGBL through
678 the fund POR CREO FESR 2007-2013 (Attività 1.5.a - 1.6). Moreover, the authors thank ENEL
679 SpA – Engineering & Research for the use of Aspen Plus®.

680

681 **REFERENCES**

- 682 [1] Grassi G, Gosse G, Dos-Santos G. Biomass for energy and industry. London: Elsevier
683 Applied Science; 1990.
- 684 [2] Wang L, Weller CL, Jones DD, Hanna MA. Review - Contemporary issues in thermal
685 gasification of biomass and its application to electricity and fuel production. Biomass and
686 Bioenergy 2008; 32: 573-581.
- 687 [3] Kirubakaran V, Sivaramakrishnan V, Nalini R, Sekar T, Premalatha M, Subramanian P. A
688 review on gasification of biomass. Renewable and Sustainable Energy Reviews 2009; 13:
689 179-186.
- 690 [4] Asadullah M. Barriers of commercial power generation using biomass gasification gas: a
691 review. Renewable and Sustainable Energy Reviews 2014; 29: 201-215.
- 692 [5] Ahmad AA, Zawawi NA, Kasim FH, Inayat A. Assessing the gasification performance of
693 biomass: a review on biomass gasification process conditions, optimization and economic
694 evaluation. Renewable and Sustainable Energy Reviews 2016; 53: 1333-1347.
- 695 [6] Janajreh I, Al Shrah M. Numerical and experimental investigation of downdraft
696 gasification of wood chips. Energy Conversion and Management 2013; 65: 783-792.
- 697 [7] Puig-Arnabat M, Bruno JC, Coronas A. Review and analysis of biomass gasification models.
698 Renewable and Sustainable Energy Reviews 2010; 14: 2841-2851.

- 699 [8] Baruah D, Baruah DC. Modelling of biomass gasification: a review. Renewable and
700 Sustainable Energy Reviews 2014; 39: 806-815.
- 701 [9] Patra TK, Sheth PN. Biomass gasification models for downdraft gasifier: a state-of-the-art
702 review. Renewable and Sustainable Energy Reviews 2015; 50: 583-593.
- 703 [10] Sharma AKr. Modeling and simulation of a downdraft biomass gasifier 1. Model
704 development and validation. Energy Conversion and Management 2011; 52: 1386-1396.
- 705 [11] Sharma S, Sheth PN. Air–steam biomass gasification: Experiments, modeling and
706 simulation. Energy Conversion and Management 2016; 110: 307–318.
- 707 [12] Ghassemi H, Shahsavan-Markadeh R. Effects of various operational parameters on
708 biomass gasification process; a modified equilibrium model. Energy Conversion and
709 Management 2014; 79: 18–24.
- 710 [13] Andrés Z, Mendiburu AZ, Carvalho Jr JA, Zanzi R, Coronado CR, Silveira JL. Thermochemical
711 equilibrium modeling of a biomass downdraft gasifier: Constrained and unconstrained
712 non-stoichiometric models. Energy 2014; 71: 624–637.
- 713 [14] Ramzan N, Ashraf A, Naveed S, Malik A. Simulation of hybrid biomass gasification using
714 Aspen plus: A comparative performance analysis for food, municipal solid and poultry
715 waste. Biomass and Bioenergy 2011; 35: 3962-3969.
- 716 [15] Fu Z, Zhang Y, Liu H, Zhang B, Li B. Simulation analysis of biomass gasification in an
717 autothermal gasifier using Aspen Plus. Cleaner Combustion and Sustainable World 2012;
718 479-483, Springer Berlin Heidelberg.
- 719 [16] Doherty W, Reynolds A, Kennedy D. The effect of air preheating in a biomass CFB gasifier
720 using ASPEN Plus simulation. Biomass and Bioenergy 2009; 33: 1158-1167.
- 721 [17] Doherty W, Reynolds A, Kennedy D. Simulation of a circulating fluidised bed biomass
722 gasifier using Aspen Plus - A performance analysis. Proc. 21st International Conference on

723 Efficiency, Cost, Optimization, Simulation and Environmental Impact of Energy Systems,
724 Krakow, Poland, 2008, 1241-1248.

725 [18] Doherty W, Reynolds A, Kennedy D. Aspen Plus Simulation of Biomass Gasification in a
726 Steam Blown Dual Fluidised Bed, Book Chapter: Materials and processes for energy:
727 communicating current research and technological developments, A. Méndez-Vilas (Ed.),
728 Formatex Research Centre, 2013.

729 [19] Abdelouahed L, Authier O, Mauviel G, Corriou JP, Verdier G, Dufour A. Detailed modeling
730 of biomass gasification in dual fluidised bed reactors under Aspen Plus. Energy Fuels 2012;
731 26: 3840-3855.

732 [20] Francois J, Abdelouahed L, Mauviel G, Feid M, Rogaume C, Mirgaux O, Patisson F, Dufour
733 A. Estimation of the energy efficiency of a wood gasification CHP plant using Aspen Plus.
734 Chemical Engineering Transactions 2012; 29: 769-774.

735 [21] Francois J, Abdelouahed L, Mauviel G, Patisson F, Mirgaux O, Rogaume C, Rogaume Y, Feid
736 M, Dufour A. Detailed process modeling of a wood gasification combined heat and power
737 plant. Biomass and Bioenergy 2013; 51: 68-82.

738 [22] Nikoo MB, Mahinpey N. Simulation of biomass gasification in fluidized bed reactor using
739 Aspen Plus. Biomass and Bioenergy 2004; 32: 1245-1254.

740 [23] Kuo PC, Wu W, Chen WH. Gasification performances of raw and torrefied biomass in a
741 downdraft fixed bed gasifier using thermodynamic analysis. Fuel 2014; 117: 1231-1241.

742 [24] Mathieu P, Dubuisson R. Performance analysis of a biomass gasifier. Energy Conversion
743 and Management 2002; 43: 1291-1299.

744 [25] Damartzis Th, Michailos S, Zabaniotou A. Energetic assessment of a combined heat and
745 power integrated biomass gasification-internal combustion engine system by using Aspen
746 Plus. Fuel Processing Technology 2012; 95: 37-44.

- 747 [26] Sreejith CC, Muraleedharan C, Arun P. Performance prediction of steam gasification of
748 wood using an Aspen Plus thermodynamic equilibrium model. *International Journal of*
749 *Sustainable Energy* 2014; 33: 416-434.
- 750 [27] Tremel A, Becherer D, Fendt D, Gaderer M, Spliethoff H. Performance of entrained flow
751 and fluidised bed biomass gasifiers on different scales. *Energy Conversion and*
752 *Management* 2013; 69: 95–106.
- 753 [28] Do TX, Lim Yi, Yeo H, Lee Ud, Choi Yt, Song Jh. Techno-economic analysis of power plant
754 via circulating fluidised-bed gasification from woodchips. *Energy* 2014; 70: 547-560.
- 755 [29] Beheshti SM, Ghassemi H, Shahsavan-Markadeh R. Process simulation of biomass
756 gasification in a bubbling fluidised bed reactor. *Energy Conversion and Management*
757 2015; 94: 345-352.
- 758 [30] Tapasvi D, Kempegowda RS, Tran KQ, Skreiberg O, Gronli M. A simulation study on the
759 torrefied biomass gasification. *Energy Conversion and Management* 2015; 90: 446–457.
- 760 [31] Parvez AM, Mujtaba IM, Wu T. Energy, exergy and environmental analyses of
761 conventional, steam and CO₂-enhanced rice straw gasification. *Energy* 2016; 94: 579-588.
- 762 [32] Adeyemi I, Janajreh I. Modeling of the entrained flow gasification: Kinetics-based ASPEN
763 Plus model. *Renewable Energy* 2015; 82: 77-84.
- 764 [33] Font Palma C. Modelling of tar formation and evolution for biomass gasification: A review.
765 *Applied Energy* 2013; 111: 129-141.
- 766 [34] Li C, Suzuki K. Tar property, analysis, reforming mechanism and model for biomass
767 gasification - An overview. *Renewable and Sustainable Energy Reviews* 2009; 13: 594-604.
- 768 [35] Technical Report “Sampling and analysis of tar and particles in biomass producer gases”
769 Prepared under CEN BT/TF 143 “Organic contamination (“tar”) in biomass producer
770 gases”.

- 771 [36] UNI CEN/TS 15439:2008 “Biomass gasification, Tar and particles in product gases,
772 Sampling and analysis”, European code.
- 773 [37] ISO 5167-2:2003 “Measurement of fluid flow by means of pressure differential devices
774 inserted in circular cross-section conduits running full -- Part 2: Orifice plates”,
775 International code.
- 776 [38] Jarunghammachote S, Dutta A. Thermodynamic equilibrium model and second law
777 analysis of a downdraft waste gasifier. *Energy* 2007; 32: 1660-1669.
- 778 [39] Baratieri M, Baggio P, Fiori L, Grigiante M. Biomass as an energy source: Thermodynamic
779 constraints on the performance of the conversion process. *Bioresource Technology* 2008;
780 99: 7063-7073.
- 781 [40] Zainal ZA, Rifau A, Quadir GA, Seetharamu KN. Experimental investigation of a downdraft
782 biomass gasifier. *Biomass and Bioenergy* 2002; 23: 283-289.
- 783 [41] Melgar A, Pe´rez JF, Laget H, Horillo A. Thermochemical equilibrium modelling of a
784 gasifying process. *Energy Conversion and Management* 2007; 48: 59-67.
- 785 [42] Jayah TH, Aye L, Fuller RJ, Stewart DF. Computer simulation of a downdraft wood gasifier
786 for tea drying. *Biomass and Bioenergy* 2003; 25: 459–469.
- 787 [43] Rhodes M. *Introduction of Particle Technology*. Chichester: John Wiley & Sons; 2008.
- 788 [44] Bejan A, Tsatsaronis G, Moran M. *Thermal Design and Optimization*. Chichester: John
789 Wiley & Sons; 1995.
- 790 [45] Bejan A. *Convection Heat Transfer*. Chichester: John Wiley & Sons; 2013.
- 791 [46] Mills AF. *Heat Transfer*. Homewood: Irwin Inc.; 1992.